



Volume 3: Residential Measures

Ameren Missouri TRM – Volume 3: Residential Measures Revision Log

Revision	Date	Description
1.0	05/30/2018	Initial version filed for Commission approval.
2.0	12/21/2018	Updated "Deemed Tables" with PY2017 Evaluation results per Stipulation and Agreement (File No. EO-2018-0211). Added Demand Response language per Stipulation and Agreement.
3.0	1/01/2020	Updated "Deemed Tables" with PY2018 Evaluation results. Also includes revisions to HVAC measures and multifamily measures, based on feedback from evaluation contractor. This includes updates to Volume 3 of the TRM.
4.0	10/15/2020	Updated "Deemed Tables" with PY2019 Evaluation results and other revisions to improve consistency with Deemed tables.
5.0	09/15/21	Updated "Deemed Tables" with PY2020 Evaluation results and other revisions to improve consistency with Deemed tables.
6.0	09/26/2022	Updated "Deemed Tables" with PY2021 Evaluation results and other revisions to improve consistency with Deemed Tables. Other revisions include updates to incremental costs for low flow showerheads, in-service rates for low flow showerheads and faucet aerators based on PY2021 evaluation, incorporation of SEER to SEER2 and HSPF to HSPF2 conversion factors due to upcoming Code of Federal Regulation testing procedures, and updates to PTHP and PTAC baseline code efficiencies.
7.0	10/05/2023	Addition of Pay As You Save (PAYS®) ISR's. Added language to clarify that ccASHP's must meet the majority of a home's heating needs. Updated HVAC baselines for heat pumps to CFR standards, with a TRM effective date of 1/1/2024 to allow for sell-through; Updates to lighting measures to address EISA updates to general service lamps (GSL), effective 8/1/2023. Updated deemed costs of light bulbs to reflect first year cost per bulb.
8.0	01/15/2025	Removed measures to align with approved measure list from Commission on 01/15/25. Removed several measures currently not active in the programs and added cool roof measures. Updated incremental cost for some measures, including Heating and Cooling CAC/ASHP. Reviewed and updated all measures including source documentation.
9.0	11/10/2025	Updated "Deemed Tables" with PY24 Evaluation results including updates to SEER and HSPF values for applicable Residential and Commercial HVAC equipment, Added new measures such as 9 multi-family air-source heat pump measure combinations based on PY2024 program data to fill gaps of SEER and baseline specific equipment, updated consistent naming convention for "Smart Thermostat measures", clean up the "program/channel" designations in appendix F to better align with implementation and updated Appendix G, H and I for consistency.

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Volume 3: Residential Measures

3.1 Appliances

3.1.1 Refrigerator

DESCRIPTION

A refrigerator meeting either ENERGY STAR®/CEE Tier 1 specifications or the higher efficiency specifications of CEE Tier 2 or CEE Tier 3 is installed instead of a new unit of baseline efficiency. The measure applies to TOS and early replacement programs.

This measure also includes a section accounting for the interactive effect of reduced waste heat on the heating and cooling loads.

This measure was developed to be applicable to the following program types: TOS, NC, and EREP.

If applied to other program types, the measure savings should be verified.

Energy usage specifications are defined in the table below (note, Adjusted Volume is calculated as the fresh volume + (1.63 * Freezer Volume):

Product Category	Existing Unit	Assumptions after September 2014	
	Based on Refrigerator Recycling algorithm	Federal Baseline Maximum Energy Usage in kWh/year ¹	ENERGY STAR Maximum Energy Usage in kWh/year ²
1. Refrigerators and Refrigerator-freezers with manual defrost	Method to measure to estimate existing unit consumption defined below.	6.79AV + 193.6	6.11 * AV + 174.2
2. Refrigerator-Freezer--partial automatic defrost		7.99AV + 225.0	7.19 * AV + 202.5
3. Refrigerator-Freezers--automatic defrost with top-mounted freezer without through-the-door ice service and all-refrigerators--automatic defrost		8.07AV + 233.7	7.26 * AV + 210.3
4. Refrigerator-Freezers--automatic defrost with side-mounted freezer without through-the-door ice service		8.51AV + 297.8	7.66 * AV + 268.0
5. Refrigerator-Freezers--automatic defrost with bottom-mounted freezer without through-the-door ice service		8.85AV + 317.0	7.97 * AV + 285.3
5A Refrigerator-freezer—automatic defrost with bottom-mounted freezer with through-the-door ice service		9.25AV + 475.4	8.33 * AV + 436.3
6. Refrigerator-Freezers--automatic defrost with top-mounted freezer with through-the-door ice service		8.40AV + 385.4	7.56 * AV + 355.3
7. Refrigerator-Freezers--automatic defrost with side-mounted freezer with through-the-door ice service		8.54AV + 432.8	7.69 * AV + 397.9

¹ See Department of Energy Federal Standards (10 CFR Part 430.32(a)), effective September 15th, 2014.

² See Version 5.1 ENERGY STAR specification.

DEFINITION OF EFFICIENT EQUIPMENT

The high-efficiency level is a refrigerator meeting ENERGY STAR® specifications effective September 15th, 2014 (10% above federal standard), a refrigerator meeting CEE Tier 2 specifications (15% above federal standard), or CEE Tier 3 specifications (20% above federal standards).

DEFINITION OF BASELINE EQUIPMENT

Time of Sale: Baseline efficiency is a new refrigerator meeting the minimum federal efficiency standard for refrigerators.

Early Replacement: the baseline is the existing refrigerator for the assumed remaining useful life of the unit and the time of sale baseline as defined above for the remainder of the measure life. Application of early replacement baseline is applicable to low income programs and requires information on pre-existing unit age and configuration.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 15 years.³

Remaining life of existing equipment is assumed to be 5 years.⁴

DEEMED MEASURE COST

The full cost of a baseline unit is \$742.⁵

The incremental cost to the ENERGY STAR® level is \$28, to CEE Tier 2 level is \$112, and to CEE Tier 3 is \$134.⁶

LOADSHAPE

Refrigeration RES

Algorithm

CALCULATION OF SAVINGS**ELECTRIC ENERGY SAVINGS**

Savings will be calculated based on model ENERGY STAR® data, if available. If applicable model ENERGY STAR® data is unavailable, savings by product class may be calculated according to the algorithm below:

Time of sale:

$$\Delta \text{kWh}_{\text{Unit}} = \text{kWh}_{\text{base}} - \text{kWh}_{\text{ee}}$$

Early replacement:

ΔkWh for remaining life of existing unit (1st 5 years):

$$= \text{kWh}_{\text{exist}} - \text{kWh}_{\text{ee}}$$

ΔkWh for remaining measure life (next 10 years):

$$= \text{kWh}_{\text{base}} - \text{kWh}_{\text{ee}}$$

³ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 35. Based on 2021 DOE Rulemaking Technical Support Document, 'DOE LCC Spreadsheet.xlsm'.

⁴ Standard assumption of one third of effective useful life.

⁵ Configurations weighted according to table under Energy Savings. Values inflated 8.9% from 2009 dollars to 2015. Table 8.1.1, DOE, 2011-08-23 Technical Support Document for Energy Conservation Standards for Residential Refrigerators, Refrigerator-Freezers, and Freezers. https://www1.eere.energy.gov/buildings/appliance_standards/pdfs/refrig_finalrule_tsd.pdf; 'refrig_finalrule_tsd.pdf'.

⁶ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, p. 35. Costs are estimated using the data provided in the Department of Energy, "Refrigerator, Refrigerator-Freezer, and Freezer Life-Cycle Cost (LCC) Analysis Spreadsheet" posted November 9, 2021 as part of the 'Energy Conservation Standards for Consumer Refrigerators, Refrigerator-Freezers, and Freezers' rulemaking docket (see 'DOE LCC Spreadsheet.xlsm'). Install cost data was trended to provide estimates at the efficiency levels specified in this measure, and then weighted based on available product on the ENERGY STAR® Refrigerator QPI.

Where:

- kWh_{ec}

= Actual. If unknown, calculate by product class:
= (kWh_{base} * (1 – %Savings))
- kWh_{base}

= Annual electric energy consumption of baseline unit as calculated in algorithm provided in table above,⁷ if new model known; otherwise, assuming 22.5 ft³ adjusted volume⁸ using “all-refrigerators-automatic defrost” formula 8.07AV + 233.7, use value 415.28.
- kWh_{exist}

= If pre-existing unit age and configuration known: see table below to determine electric energy consumption of pre-existing unit based on unit age and configuration.
If pre-existing unit age and configuration not known: do not apply early replacement baseline.
- %Savings

= Specification of energy consumption below Federal Standard – see table below.

Tier	%Savings
Energy Star® and CEE Tier 1	10%
Energy Star® Most Efficient and CEE Tier 2	15%
CEE Tier 3	20%

If an early replacement baseline is applicable, the following table may be used to calculate the baseline usage used to calculate savings for the first six years of measure life:

Age	Bottom Freezer (16 cu ft)	Side-by-Side (14 cu ft)	Side-by-Side (15 cu ft)	Side-by-Side (16 cu ft)	Top Freezer (cu ft 14)	Top Freezer (15 cu ft)	Top Freezer (16 cu ft)	Top Freezer (17 cu ft)	Top Freezer (18 cu ft)
2011-2015	483	592	592	592	374	374	374	412	412
2001 (after July-2010)	724	747	747	747	556	556	556	613	613
1993-2001(before June)	962	1,139	1,139	1,139	861	861	861	962	962
1990-1992	1,519	1,617	1,617	1,617	1,272	1,272	1,272	1,432	1,432
1980-1989	1,992	2,119	2,119	2,119	1,668	1,668	1,668	1,877	1,877
Before 1980	2,523	2,684	2,684	2,684	2,112	2,112	2,112	2,377	2,377

Additional Waste Heat Impacts

For units in conditioned spaces in the home (if unknown, assume unit is in conditioned space).

$\Delta kWh_{WasteHeat} = \Delta kWh * (WHFeHeatElectric + WHFeCool)$

Where:

- ΔkWh

= kWh savings calculated from either method above
- $WHFeHeatElectric$

= Waste Heat Factor for Energy to account for electric heating increase from removing waste heat from refrigerator/freezer (if fossil fuel heating – see calculation of heating penalty in that section).
= - (HF / $\eta_{HeatElectric}$) * %ElecHeat
- HF

= Heating Factor or percentage of reduced waste heat that must now be heated
= 58% for unit in heated space or unknown ⁹
= 0% for unit in unheated space

⁷ According to Federal Standard effective 9/15/14.

⁸ DOE Building Energy Data Book, <https://ieer.org/wp/wp-content/uploads/2012/03/DOE-2011-Buildings-Energy-DataBook-BEDB.pdf>; ‘DOE-2011-Buildings-Energy-DataBook-BEDB.pdf’

⁹ Based on 212 days where HDD 65>0, divided by 365.25.

$\eta_{HeatElectric}$ = Efficiency in COP of Heating equipment
= Actual - If not available, use table below:¹⁰
 $\%ElecHeat$ = Percentage of home with electric heat

System Type	Age of Equipment	HSPF2 Estimate	η_{Heat} (COP Estimate) = (HSPF2/3.413)*0.85
Heat Pump (if age unknown assume 2006-2014)	Before 2006	5.8	1.44
	After 2006 - 2014	6.5	1.62
	2015 on	7.0	1.74
Resistance	N/A	N/A	1.00
Unknown ¹¹	N/A	N/A	1.28

Heating Fuel	$\%ElecHeat$
Electric	100%
Fossil Fuel	0%
Unknown	35% ¹²

$WHFeCool$ = Waste Heat Factor for Energy to account for cooling savings from removing waste heat from refrigerator/freezer.
= $CoolF / (\eta_{Cool} / 3.412) * \%Cool$
 $CoolF$ = Cooling Factor or percentage of reduced waste heat that no longer needs to be cooled
= 40% for unit in cooled space or unknown¹³
= 0% for unit in uncooled space
 η_{Cool} = Efficiency in COP of Cooling equipment
= Actual - If not available, see table below
 $\%Cool$ = Percentage of home with cooling

Age of Equipment	η_{Cool} Estimate
Before 2006	9.5
2006 - 2014	12.4
Central AC After 1/1/2015	12.4
Heat Pump After 1/1/2015	13.3

Home	$\%Cool$
Cooling	100%
No Cooling	0%
Unknown	91% ¹⁴

¹⁰ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 and 2015 the federal standard for heat pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

¹¹ Calculation assumes 35% Heat Pump and 65% Resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, <https://www.eia.gov/consumption/residential/data/2009/hc/hc6.9.xls> ; ‘hc6.9.xls’. Average efficiency of heat pump is based on assumption that 50% are units from before 2006 and 50% from 2006-2014. Program or evaluation data should be used to improve this assumption if available.

¹² Based on data from Energy Information Administration, 2009 Residential Energy Consumption Survey, see <https://www.eia.gov/consumption/residential/data/2009/hc/hc6.9.xls>; ‘hc6.9.xls’.

¹³ Based on 148 days where CDD 65>0, divided by 365.25.air

¹⁴ Based on 2009 Residential Energy Consumption Survey, see “HC7.9 Air Conditioning in Midwest Region”, <https://www.eia.gov/consumption/residential/data/2015/hc/php/hc7.9.php>.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW = \Delta kWh * CF$

Where:

- ΔkWh = gross customer connected load kWh savings for the measure. Including any cooling system savings.
- CF = Summer Peak Coincident Factor
= 0.0001285253¹⁵

NATURAL GAS SAVINGS

Heating penalty for reduction in waste heat, only for units from conditioned space in gas heated home (if unknown, assume unit is from conditioned space).

$\Delta Therms = \Delta kWh_{Unit} * WHFeHeatGas * 0.03412$

Where:

- ΔkWh_{Unit} = kWh savings calculated from either method above, not including the $\Delta kWh_{WasteHeat}$
- $WHFeHeatGas$ = Waste Heat Factor for Energy to account for gas heating increase from removing waste heat from refrigerator/freezer
= - (HF / $\eta_{HeatGas}$) * %GasHeat
- HF = Heating Factor or percentage of reduced waste heat that must now be heated
= 58% for unit in heated space or unknown¹⁶
= 0% for unit in unheated space
- $\eta_{HeatGas}$ = Efficiency of heating system
= 74%¹⁷
- $\%GasHeat$ = Percentage of homes with gas heat
- 0.03412 = Converts kWh to therms

Heating Fuel	%GasHeat
Electric	0%
Gas	100%
Unknown	65% ¹⁸

Default values for each product class and unknown building characteristics are provided below:

Product Class	$\Delta Therms$		
	Energy Star [®] / CEE Tier 1	CEE Tier 2	CEE Tier 3
Top Freezer (PC 3)	-1.19	-1.78	-2.37
Side-by-Side w/ TTD (PC 7)	-1.29	-1.94	-2.58
Bottom Freezer (PC 5)	-1.07	-1.60	-2.13
Bottom Freezer w/ TTD (PC 5A)	-1.41	-2.12	-2.83

¹⁵ Ameren Missouri TRM Volume 1 - Appendix G: "Table 1 – Residential End Use Category Monthly Shapes and Coincident Peak Factors"

¹⁶ Based on 212 days where HDD 65>0, divided by 365.25.

¹⁷ This has been estimated assuming that natural gas central furnace heating is typical for Missouri residences (the predominant heating is gas furnace with 52% of Missouri homes - based on Energy Information Administration, 2009 Residential Energy Consumption Survey). Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: ((0.60*0.92) + (0.40*0.8)) * (1-0.15) = 0.74.

¹⁸ Based on data from Energy Information Administration, 2009 Residential Energy Consumption Survey, see <https://www.eia.gov/consumption/residential/data/2009/hc/hc6.9.xls>; 'hc6.9.xls'.

If product class is unknown, the following table provides a market weighting that is applied to give a single deemed savings for each efficiency level:

Product Class	Market Weight ¹⁹	ΔTherms		
		Energy Star [®] / CEE Tier 1	CEE Tier 2	CEE Tier 3
Top Freezer (PC 3)	52%	-1.22	-1.84	-2.45
Side-by-Side w/ TTD (PC 7)	22%			
Bottom Freezer (PC 5)	13%			
Bottom Freezer w/ TTD (PC 5A)	13%			

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

¹⁹ Personal Communication from Melisa Fiffer, ENERGY STAR[®] Appliance Program Manager, EPA 10/26/14.

3.2 Electronics

3.2.1 Advanced Tier 1 Power Strips

DESCRIPTION

This measure applies to Tier 1 Advanced Power Strips (APS), which are multi-plug power strips with the ability to automatically disconnect specific connected loads depending upon the power draw of a master control load, also plugged into the strip. Power is disconnected from the switched (controlled) outlets when the master control load power draw is reduced below a certain adjustable threshold, thus turning off the appliances plugged into the switched outlets. By disconnecting, the standby load of the controlled devices, the overall load of a centralized group of equipment (i.e. entertainment centers and home office) can be reduced. Uncontrolled outlets are also provided that are not affected by the control device and are always providing power to any device plugged into it. This measure characterization provides savings for use of an APS in a home entertainment system, home office, or unknown setting.

This measure was developed to be applicable to the following program types: TOS, NC, DI, and KITS.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient case is the use of a 4-8 plug Tier 1 master-controlled APS.

DEFINITION OF BASELINE EQUIPMENT

For TOS and NC applications, the baseline is a standard power strip that does not control connected loads.
For DI and KITS, the baseline is the existing equipment used in the home.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed lifetime of the Tier 1 APS is 10 years.²⁰

DEEMED MEASURE COST

For DI, the actual full installation cost of an APS (including equipment and labor) should be used. If DI cost is unknown, cost is assumed to be \$30.00.²¹

LOADSHAPE

Miscellaneous RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$\Delta kWh = (kWh_{Office} * Weighting_{Office} + kWh_{Ent} * Weighting_{Ent}) * ISR$

Where:

kWh_{Office} = Estimated energy savings from using an APS in a home office
= 31.0 kWh²²

$Weighting_{Office}$ = Relative penetration of use in home office

²⁰ “Advanced Power Strip Research Report,” NYSERDA, August 2011, <https://www.nyserda.ny.gov/-/media/Project/Nyserda/Files/EERP/Residential/Power-Management-Research-Report.pdf>; ‘NYSERDA Advanced Power Strip Research Report.pdf’, page 30

²¹ Ibid.

²² “Advanced Power Strip Research Report.” <https://www.nyserda.ny.gov/-/media/Project/Nyserda/Files/EERP/Residential/Power-Management-Research-Report.pdf>; ‘NYSERDA Advanced Power Strip Research Report.pdf’. Note that estimates are not based on pre/post metering but on analysis based on frequency and consumption of likely products in active, standby, and off modes. This measure should be reviewed frequently to ensure that assumptions continue to be appropriate.

Installation Location	Weighting ^{Office}
Home Office	100%

ISR = In-service rate. Actual, or if unknown, reference values in the table below dependent on program type

Program Type	ISR
Pay As You Save ²³	74.3%

Based on the default values above, default savings are provided in the table below:

Installation Location	Program Type	ΔkWh
Home Office	DI	23.03

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW = \Delta kWh * CF$

Where:

ΔkWh = Electric energy savings, as calculated above.
CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0001148238²⁴

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

²³ Ameren Missouri Pay As You Save (PAYS[®]) Evaluation Appendices: PY2022, <https://www.efis.psc.mo.gov/Document/Display/17591>, page 7.
²⁴ Based on Ameren Missouri TRM Volume 1 - Appendix G: "Table 1 – Residential End Use Category Monthly Shapes and Coincident Peak Factors"for residential miscellaneous end-use. This is deemed appropriate, because savings occur during hours when the controlled standby loads are turned off by the APS. This is estimated to be approximately 7,129, which representing the average of hours for controlled TV and computer from “Advanced Power Strip Research Report.”, <https://www.nyserda.ny.gov/-/media/Project/Nyserda/Files/EERP/Residential/Power-Management-Research-Report.pdf>; ‘NYSERDA Advanced Power Strip Research Report.pdf’.

3.2.2 Advanced Tier 2 Power Strip – Residential Audio Visual

DESCRIPTION

This measure relates to the installation of a Advanced Tier 2 Power Strip / surge protector for household audio visual environments (Tier 2 AV APS). Tier 2 AV APS are multi-plug power strips that remove power from audio visual equipment through intelligent control and monitoring strategies.

By utilizing advanced control strategies such as a countdown timer, external sensors (e.g. of infra-red remote usage and/or occupancy sensors, true RMS (Root Mean Square) power sensing; both active power loads and standby power loads of controlled devices are managed by Tier 2 AV APS devices.²⁵ Monitoring and controlling both active and standby power loads of controlled devices will reduce the overall load of a centralized group of electrical equipment (i.e. the home entertainment center). This more intelligent sensing and control process has been demonstrated to deliver increased energy savings and demand reduction compared with ‘Tier 1 Advanced Power Strips’.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient case is the use of a Tier 2 AV APS in a residential AV (home entertainment) environment that includes control of at least 2 AV devices, one being the television.²⁶

DEFINITION OF BASELINE EQUIPMENT

The assumed baseline equipment is the existing equipment used in the home (e.g., a standard power strip or wall socket) that does not control loads of connected AV equipment.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed lifetime of the Tier 2 AV APS is assumed to be 10 years.²⁷

DEEMED MEASURE COST

Direct Install: The actual installed cost (including labor) of the new Tier 2 AV APS equipment should be used, less baseline cost of \$20.²⁸

LOADSHAPE

Miscellaneous RES

Algorithm

CALCULATION OF ENERGY SAVINGS

ELECTRIC ENERGY SAVINGS

$\Delta kWh = ERP * BaselineEnergy_{AV} * ISR$

Where:

ERP = Energy Reduction Percentage of qualifying Tier2 AV APS product range as provided below. Savings are based upon independent field trials of two product manufacturers and the savings differences are assumed to relate to the product classifications provided below. Additional evaluation will be reviewed in future cycles to confirm if additional classification categories are appropriate.

Product Type	ERP used
Infrared Only	40% ²⁹

²⁵ Tier 2 AV APS identify when people are not engaged with their AV equipment and then remove power, for example a TV and its peripheral devices that are unintentionally left on when a person leaves the house or for instance where someone falls asleep while watching television.

²⁶ Given this requirement, an AV environment consisting of a TV and DVD player or a TV and home theater would be eligible for a Tier 2 AV APS installation.

²⁷ “Advanced Power Strip Research Report,” NYSERDA, August 2011, <https://www.nyserdera.ny.gov/-/media/Project/Nyserda/Files/EERP/Residential/Power-Management-Research-Report.pdf>; ‘NYSERDA Advanced Power Strip Research Report.pdf’, page 30.

²⁸ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 93. Price survey performed by Illume Advising LLC for IL TRM workpaper, see ‘Current Surge Protector Costs and Comparison 7-2016.xlsx’ spreadsheet

²⁹ Representative savings assumption based on the following independent field tests on Embertec’s IR-only product. This includes both simulated saving results (based on recording what action the APS would have taken, but where equipment is not actually switched off allowing evaluation of the expected length of savings), and pre/post metering studies.

• AESC (page 30) - Valmiki, MM., Corradini, Antonio PE. 2015. *Tier 2 Advanced Power Strips in Residential and Commercial Applications*. Prepared for San Diego Gas &

BaselineEnergy_{AV} = 466 kWh³⁰

ISR = In Service Rate, the percentage of units rebated that are actually in service. Actual, or if unknown, referencing the table below:

Program/Channel	In Service Rate (ISR)
Income Eligible ³¹	95%

Based on the default values above, default savings are provided in the table below:

Program Type	ΔkWh
	Infrared Only
Income Eligible	177.08

SUMMER COINCIDENT PEAK DEMAND SAVINGS

ΔkW = ΔkWh * CF

Where:

- ΔkWh = Electric energy savings, as calculated above.
- CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0001148238³²

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

Electric by Alternative Energy Systems Consulting, Inc. (Simulated 50%, pre/post 32%).

- AESC- Valmiki, MM., Corradini, Antonio PE., Feb 2016. *Energy Savings of Tier 2 Advanced Power Strips in Residential AV Systems*. (Simulated 50%, pre/post 29%)
- CalPlug research (Page 12) - Wang, M. e. 2014. “Tier 2 Advanced Power Strip Evaluation for Energy Saving Incentive”. California Plug Load Research Center (CalPlug), UC Irvine. (Simulated 51%)
- NMR Group Inc., *RLPNC 17-3: Advanced Power Strip Metering Study*, Revised March 18, 2019, submitted to Massachusetts Program Administrators and EEAC. (Pre/post with regression 50%, Pre/post only 20%).

³⁰ “Energy Savings of Tier 2 Advanced Power Strips in Residential AV Systems,” AESC, Inc., February 2016. Note this load represents the average *controlled* AV devices only and will likely be lower than total AV usage, https://www.aesc-inc.com/wp-content/uploads/2017/07/tier_2_aps_final_report_et13pge1441.pdf; ‘tier_2_aps_final_report_et13pge1441.pdf’, page 7.

³¹ Ameren Missouri Single Family Low Income Program Evaluation: PY2019, Table 10-10, <https://www.efis.psc.mo.gov/Document/Display/15877>, page 214.

³² Based on Ameren Missouri TRM Volume 1 - Appendix G: "Table 1 – Residential End Use Category Monthly Shapes and Coincident Peak Factors" for residential miscellaneous end-use. This is deemed appropriate, as savings occur during hours which the controlled standby loads are turned off by the APS, estimated to be approximately 7,129 representing the average of hours for controlled TV and computer from “Advanced Power Strip Research Report.”, <https://www.nyserda.ny.gov/-/media/Project/Nyserda/Files/EERP/Residential/Power-Management-Research-Report.pdf>; ‘NYSERDA Advanced Power Strip Research Report.pdf’.

3.3 Hot Water

3.3.1 Low Flow Faucet Aerator

This measure relates to the installation of a low flow faucet aerator in a household kitchen or bath faucet fixture.

This measure may be used for units provided through efficiency kits. However, the in-service rate for such measures should be derived through evaluation results specifically for this implementation methodology.

This measure was developed to be applicable to the following program types: TOS, NC, RF, DI, and KITS.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a low flow faucet aerator for bathrooms rated at 1.5 gallons per minute (GPM) or less or for kitchens rated at 2.2 GPM or less. Savings are calculated on an average savings per faucet fixture basis.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is assumed to be a standard bathroom faucet aerator rated at 2.25 GPM or greater or a standard kitchen faucet aerator rated at 2.75 GPM or greater. Average measured flow rates are used in the algorithm and are lower, reflecting the penetration of previously installed low flow fixtures, use of the faucet at less than full flow, debris buildup, and lower water system pressure than fixtures are rated at.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 10 years.³³

DEEMED MEASURE COST

The incremental cost for this measure is \$11.33³⁴ or actual cost.

LOADSHAPE

Water Heating RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Note these savings are *per* faucet retrofitted³⁵ (unless faucet type is unknown, then it is per household).

$$\Delta kWh = \%ElectricDHW * (GPM_{base} * L_{base} - GPM_{low} * L_{low}) * Household * 365.25 * DF / FPH * EPG_electric * ISR * (1 - Leakage)$$

Where:

$\%ElectricDHW$ = proportion of water heating supplied by electric resistance heating

DHW fuel	$\%ElectricDHW$
Electric	100%

GPM_{base} = Average flow rate, in gallons per minute, of the baseline faucet “as-used.” This includes the effect of existing low flow fixtures.

= 2.2³⁶ or custom based on metering studies³⁷ or if measured during DI:

³³ ComEd Effective Useful Life Research Report, Navigant, May 14, 2018, . <https://www.icc.illinois.gov/docket/P2017-0312/documents/287811/files/501915.pdf>; ‘ComEd Effective Useful Life Research Report.pdf’, page 20.

³⁴ Direct-install price per showerhead assumes cost of showerhead (market research average of \$3 and assess and install cost of \$8.33) and also assumes 20min at \$25 per hour, which is in line with the typical prevailing wage of a General Laborer, as per the Annual Wage Order No. 23 published by the Missouri Department of Labor.

³⁵ This algorithm calculates the amount of energy saved per aerator by determining the fraction of water consumption savings for the upgraded fixture.

³⁶ Federal rated maximum flow rate for faucets, <https://www.energy.gov/femp/best-management-practice-7-faucets-and-showerheads>.

³⁷ Measurement should be based on actual average flow consumed over a period of time rather than a one-time spot measurement for maximum flow. Studies have shown maximum flow rates do not correspond well to average flow rate due to occupant behavior, which does not always use maximum flow.

GPM_{low}

= Measured full throttle flow * 0.83 throttling factor³⁸

= Average flow rate, in gallons per minute, of the low-flow faucet aerator “as-used”

= 1.5³⁹ or custom based on metering studies⁴⁰ or if measured during DI:

L_{base}

= Rated full throttle flow * 0.95 throttling factor⁴¹

= Average baseline daily length faucet use per capita for faucet of interest in minutes

= if available custom based on metering studies, if not use:

Faucet Type	L _{base} (min/person/day)	
	Kitchen	Bathroom
Income Eligible, PAYS ⁴²	3.7	3.7

L_{low}

= Average retrofit daily length faucet use per capita for faucet of interest in minutes

= if available custom based on metering studies, if not use:

Faucet Type	L _{low} (min/person/day)	
	Kitchen	Bathroom
Income Eligible, PAYS ⁴³	3.7	3.7

³⁸ 2008, Schuldt, Marc, and Debra Tachibana, “Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes,” 2008 ACEEE Summer Study on Energy Efficiency in Buildings, pp. 1-265. https://map-testing.com/wp-content/uploads/2022/11/2008_Seattle_Study.pdf; ‘2008_Seattle_Study.pdf’.

³⁹ Program data, including PY2016 Program Data, per Community Savers 2016 EM&V report. <https://www.efis.psc.mo.gov/Document/Display/35141>, page 3-8.

⁴⁰ Measurement should be based on actual average flow consumed over a period of time rather than a one-time spot measurement for maximum flow. Studies have shown maximum flow rates do not correspond well to average flow rate due to occupant behavior, which does not always use maximum flow.

⁴¹ 2008, Schuldt, Marc, and Debra Tachibana, “Energy related Water Fixture Measurements: Securing the Baseline for Northwest Single Family Homes,” 2008 ACEEE Summer Study on Energy Efficiency in Buildings, pp. 1-265. https://map-testing.com/wp-content/uploads/2022/11/2008_Seattle_Study.pdf; ‘2008_Seattle_Study.pdf’.

⁴² Cadmus PY11 metering study. Cited in Ameren Missouri Low Income and Process Evaluation: program Year 2015. p.23 <https://www.efis.psc.mo.gov/Document/Display/12018>

⁴³ Cadmus PY11 metering study. Cited in Ameren Missouri Low Income and Process Evaluation: program Year 2015. p.23 <https://www.efis.psc.mo.gov/Document/Display/12018>

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Household = Average number of people per household

Program Delivery and Household Unit Type	Value
Income Eligible, PAYS	1.564 ⁴⁴
Custom	Actual Occupancy or Number of Bedrooms ⁴⁵

365.25 = Days in a year, on average.

DF = Drain Factor

Program Delivery	Drain Factor	
	Kitchen	Bath
Income Eligible, PAYS ⁴⁶	100%	100%

FPH = Faucets Per Household

Program Delivery	FPH	
	Kitchen (KFPH)	Bathroom (BFPH)
Income Eligible, PAYS	1.00 ⁴⁷	1.86 ⁴⁸

EPG_electric = Energy per gallon of water used by faucet supplied by electric water heater

$$= (8.33 * 1.0 * (\text{WaterTemp} - \text{SupplyTemp})) / (\text{RE_electric} * 3412)$$

8.33 = Specific weight of water (lbs/gallon)

1.0 = Heat Capacity of water (btu/lb-°F)

WaterTemp = Assumed temperature of mixed water

= 86F for Bathroom (80F for Income Eligible, PAYS and MFMR), 93F for Kitchen, 91F for Unknown⁴⁹

SupplyTemp = Assumed temperature of water entering house
 = 58.4F⁵⁰

RE_electric = Recovery efficiency of electric water heater
 = 98%⁵¹

3,412 = Converts Btu to kWh (btu/kWh)

ISR = In-service rate. Actual, or if unknown, reference applicable assumed value in the table below:

Selection	In-Service Rate	
	Kitchen	Bathroom
Income Eligible, PAYS ⁵²	95%	95%

Leakage = Percent homes outside service territory

Program	Leakage
Income Eligible, PAYS	0%

⁴⁴ PY2006 program data (not reported in PY2016).

⁴⁵ Bedrooms are suitable proxies for household occupancy and may be preferable to actual occupancy due to turnover rates in residency and non-adult population impacts.

⁴⁶ Ameren Missouri Community Savers Evaluation PY2018, <https://www.efis.psc.mo.gov/Document/Display/36053>.

⁴⁷ Ameren Missouri EE Kits PY2018 Program Data, <https://www.efis.psc.mo.gov/Document/Display/15870>, page 34.

⁴⁸ Ameren Missouri Community Savers Evaluation: PY2018, <https://www.efis.psc.mo.gov/Document/Display/36053>, page 23.

⁴⁹ Cadmus and Opinion Dynamics Showerhead and Faucet Aerator Meter Study Memorandum, dated June 2013, directed to Michigan Evaluation Working Group. If the aerator location is unknown, an average of 91% should be used which is based on the assumption that 70% of household water runs through the kitchen faucet and 30% through the bathroom $(0.7*93)+(0.3*86)=0.91$.

⁵⁰ National Weather Service. Average soil temperature at 40" depth during 2015 - 2023 data of six stations in Ameren Missouri service territory. https://www.weather.gov/ncrfc/LMI_SoilTemperatureDepthMaps.

⁵¹ Electric water heaters have recovery efficiency of 98%: NREL Building America Research Benchmark Definition, December 2009, page 12, <https://www.nrel.gov/docs/fy10osti/47246.pdf>.

⁵² Ameren Missouri Community Savers Evaluation PY2018, <https://www.efis.psc.mo.gov/Document/Display/36053>, page 23.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW = \Delta kWh * CF$

Where:

ΔkWh = as calculated above
 CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0000887318⁵³

NATURAL GAS SAVINGS

$\Delta Therms = \%GasDHW * (GPM_{base} * L_{base} - GPM_{low} * L_{low}) * Household * 365.25 * DF / FPH * EPG_{gas} * ISR * (1 - Leakage)$

Where:

$\%GasDHW$ = proportion of water heating supplied by Natural Gas heating

DHW fuel	%GasHW
Electric	0%
Natural Gas	100%
Unknown	48% ⁵⁴

EPG_{gas} = Energy per gallon of Hot water supplied by gas
= $(8.33 * 1.0 * (WaterTemp - SupplyTemp)) / (RE_{gas} * 100,000)$
 RE_{gas} = Recovery efficiency of gas water heater
= 78% For SF homes⁵⁵
= 67% For MF homes⁵⁶
100,000 = Converts Btus to therms (btu/therm)
Other variables as defined above.

WATER IMPACT DESCRIPTIONS AND CALCULATION

$\Delta Gallons = ((GPM_{base} * L_{base} - GPM_{low} * L_{low}) * Household * 365.25 * DF / FPH) * ISR * (1 - Leakage)$

Variables as defined above.

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

⁵³ Based on Ameren Missouri TRM Volume 1 - Appendix G: "Table 1 – Residential End Use Category Monthly Shapes and Coincident Peak Factors" for residential water heating end-use.
⁵⁴ Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of Missouri, ‘HC8.9 Water Heating in Midwest Region.xls’.
⁵⁵ DOE final rule discusses recovery efficiency with an average around 0.76 for gas- fired storage water heaters and 0.78 for standard efficiency gas fired tankless water heaters up to 0.95 for the highest efficiency gas fired condensing tankless water heaters. These numbers represent the range of new units however, not the range of existing units in stock. Review of AHRI Directory suggests range of recovery efficiency ratings for new gas DHW units of 70-87%. Average of existing units is estimated at 78%.
⁵⁶ Water heating in multifamily buildings is often provided by a larger central boiler. This suggests that the average recovery efficiency is somewhere between a typical central boiler efficiency of 0.59 and the 0.75 for single family homes. An average efficiency of 0.67 is used for this analysis as a default for multifamily buildings.

3.3.2 Low Flow Showerhead

DESCRIPTION

This measure relates to the installation of a low flow showerhead in a single or multifamily household.

This measure may be used for units provided through efficiency kit’s. However, the in-service rate for such measures should be derived through evaluation results specifically for this implementation methodology.

This measure was developed to be applicable to the following program types: TOS, RF, NC, DI, and KITS.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the installed equipment must be a low flow showerhead, typically rated at 2.0 gallons per minute (GPM) or less. Savings are calculated on a per showerhead fixture basis.

DEFINITION OF BASELINE EQUIPMENT

For DI programs, the baseline condition is assumed to be a standard showerhead rated at 2.5 GPM⁵⁷ or greater.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 10 years.⁵⁸

DEEMED MEASURE COST

For low flow showerheads provided in DI programs, the actual program delivery costs should be utilized; if unknown assume \$15.33⁵⁹ for standard showerheads and \$23.35 for handheld showerheads.

LOADSHAPE

Water Heating RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Note these savings are per showerhead fixture

$$\Delta kWh = \%ElectricDHW * ((GPM_{base} * L_{base} - GPM_{low} * L_{low}) * Household * SPCD * 365.25 / SPH) * EPG_electric * ISR * (1 - Leakage)$$

Where:

$\%ElectricDHW$ = proportion of water heating supplied by electric resistance heating

DHW fuel	%ElectricDHW
Electric	100%

GPM_{base} = Flow rate of the baseline showerhead

Program Delivery	GPM base
Direct-install	2.2 ⁶⁰

⁵⁷ Maximum showerhead flow rate at 80 PSI is 2.5 GPM in accordance with federal standard 10 CFR Part 430.32(p). See docket filed at <https://www.regulations.gov/document?D=EERE-2011-BT-TP-0061-0039>.

⁵⁸ Table C-6, “Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures,” GDS Associates, June 2007. Evaluations indicate that consumer dissatisfaction may lead to reductions in persistence, particularly in Multifamily, https://www.caetrm.com/media/reference-documents/HVAC_Ltg_measure_life_GDS_2007.pdf; ‘HVAC_Ltg_measure_life_GDS_2007.pdf’, page C-6.

⁵⁹ Direct-install price per showerhead assumes cost of showerhead (market research average of \$7) and also assumes assess and install cost of \$8.33 (20min at \$25 per hour, which is in line with the typical prevailing wage of a General Laborer, as per the Annual Wage Order No. 23 published by the Missouri Department of Labor).

⁶⁰ Ameren Missouri Community Savers Evaluation: PY2018, <https://www.efis.psc.mo.gov/Document/Display/36053>.

- GPM_{low}

= As-used flow rate of the lowflow showerhead⁶¹, which may, as a result of measurements of program evaluations deviate from rated flows. If the as-used flow rate is not available, the rated flow rate should be applied.
- L_{base}

= Shower length in minutes with baseline showerhead
= 8.66 for Income Eligible and PAYS⁶²
- L_{low}

= Shower length in minutes with low-flow showerhead
= 8.66 for Income Eligible and PAYS⁶³
- Household

= Average number of people per household

Program Delivery	Household
Single-Family	2.67 ⁶⁴
Multi-Family	1.52 ⁶⁵
Custom	Actual Occupancy or Number of Bedrooms ⁶⁶

- SPCD

= Showers Per Capita Per Day
= 0.66 for Income Eligible and PAYS⁶⁷
- 365.25

= Days per year, on average.
- SPH

= Showerheads Per Household so that per-showerhead savings fractions can be determined

Program Delivery	SPH
Income Eligible Single-Family, PAYS	2.05 ⁶⁸
Income Eligible Multi-Family	1.0 ⁶⁹
Custom	Actual

- EPG_{electric}

= Energy per gallon of hot water supplied by electric
= $(8.33 * 1.0 * (\text{ShowerTemp} - \text{SupplyTemp})) / (\text{RE}_{\text{electric}} * 3412)$
= $(8.33 * 1.0 * (101 - 60.83)) / (0.98 * 3412)$
= 0.100 kWh/gal
- 8.33

= Specific weight of water (lbs/gallon)
- 1.0

= Heat capacity of water (btu/lb-°)
- ShowerTemp

= Assumed temperature of water
= 105.0 F⁷⁰
- SupplyTemp

= Assumed temperature of water entering house
= 58.4F⁷¹
- RE_{electric}

= Recovery efficiency of electric water heater
= 98%⁷²
- 3,412

= Converts Btu to kWh (btu/kWh)
- ISR

= In service rate of showerhead

⁶¹ Note that actual values may be either: a) program-specific minimum flow rate, or b) program-specific evaluation-based value of actual effective flow-rate due to increased duration or temperatures. The latter increases in likelihood as the rated flow drops and may become significant at or below rated flows of 1.5 GPM. The impact can be viewed as the inverse of the throttling described in the footnote for baseline flowrate.

⁶² DeOreo, William, P. Mayer, L. Martien, M. Hayden, A. Funk, M. Kramer-Duffield, and R. Davis (2011). “California Single Family Water Use Efficiency Study.”, https://www.waterboards.ca.gov/waterrights/water_issues/programs/hearings/byron_bethany/docs/exhibits/pt/wr71.pdf; ‘California Single Family Water Use Study.pdf’, page 91.

⁶³ DeOreo, William, P. Mayer, L. Martien, M. Hayden, A. Funk, M. Kramer-Duffield, and R. Davis (2011). “California Single Family Water Use Efficiency Study.”, https://www.waterboards.ca.gov/waterrights/water_issues/programs/hearings/byron_bethany/docs/exhibits/pt/wr71.pdf; ‘California Single Family Water Use Study.pdf’, page 91.

⁶⁴ Ameren Missouri Efficient Products Impact and Process Evaluation: Planning Year 2015, provided by Cadmus, <https://www.efis.psc.mo.gov/Document/Display/13805>, page 38.

⁶⁵ Ameren Missouri Community Savers Evaluation: PY2018, <https://www.efis.psc.mo.gov/Document/Display/36053>, page 39.

⁶⁶ Bedrooms are suitable proxies for household occupancy and may be preferable to actual occupancy due to turnover rates in residency and non-adult population impacts.

⁶⁷ DeOreo, William, P. Mayer, L. Martien, M. Hayden, A. Funk, M. Kramer-Duffield, and R. Davis (2011). “California Single Family Water Use Efficiency Study”, https://www.waterboards.ca.gov/waterrights/water_issues/programs/hearings/byron_bethany/docs/exhibits/pt/wr71.pdf; ‘California Single Family Water Use Study.pdf’.

⁶⁸ Ameren Missouri Efficient Products Impact and Process Evaluation: Planning Year 2015, provided by Cadmus, <https://www.efis.psc.mo.gov/Document/Display/12014>, page 38.

⁶⁹ Ameren Missouri Community Savers Evaluation: PY2017, <https://www.efis.psc.mo.gov/Document/Display/36053>, page 22.

⁷⁰ Ameren Missouri Efficient Kits Evaluation: PY2018, <https://www.efis.psc.mo.gov/Document/Display/15870>, page 32.

⁷¹ National Weather Service. Average soil temperature at 40" depth during 2015 - 2023 data of six stations in Ameren Missouri service territory. https://www.weather.gov/ncrfc/LMI_SoilTemperatureDepthMaps.

⁷² Electric water heaters have recovery efficiency of 98%: <http://www.ahridirectory.org/ahridirectory/pages/home.aspx>.

= Actual, or if unknown, reference applicable assumed value in the table below:

Program Delivery	ISR
Income Eligible (Single Family Direct Install) , PAYS ⁷³	94%
Income Eligible (Multifamily Direct Install) ⁷⁴	96.4%

3,412

Leakage

= Converts Btu to kWh (btu/kWh)

= Percent homes outside service territory

Program	Leakage
SFIE, MFIE, PAYS	0%

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh * CF$$

Where:

ΔkWh = as calculated above

CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0000887318⁷⁵

NATURAL GAS SAVINGS

$$\Delta \text{Therms} = \% \text{GasDHW} * ((\text{GPM}_{\text{base}} * L_{\text{base}} - \text{GPM}_{\text{low}} * L_{\text{low}}) * \text{Household} * \text{SPCD} * 365.25 / \text{SPH}) * \text{EPG}_{\text{gas}} * \text{ISR} * (1 - \text{Leakage})$$

Where:

$\% \text{GasDHW}$ = proportion of water heating supplied by natural gas heating

DHW fuel	$\% \text{GasDHW}$
Electric	0%
Natural Gas	100%
Unknown	48% ⁷⁶

EPG_{gas} = Energy per gallon of Hot water supplied by gas
= $(8.33 * 1.0 * (\text{ShowerTemp} - \text{SupplyTemp})) / (\text{RE}_{\text{gas}} * 100,000)$
= 0.00429 therm/gal for SF homes
= 0.00499 therm/gal for MF homes

RE_{gas} = Recovery efficiency of gas water heater
= 78% For SF homes⁷⁷
= 67% For MF homes⁷⁸

100,000 = Converts Btus to therms (btu/Therm)
Other variables as defined above.

⁷³ Ameren Missouri Single Family Low Income Evaluation PY2019 (Table 10-10), <https://www.efis.psc.mo.gov/Document/Display/15877>, page 214.

⁷⁴ Ameren Missouri Community Savers Evaluation PY2018 Tenant Surveys and Site Visits, <https://www.efis.psc.mo.gov/Document/Display/36053>, page 22.

⁷⁵ Based on Ameren Missouri TRM Volume 1 - Appendix G: "Table 1 – Residential End Use Category Monthly Shapes and Coincident Peak Factors" for residential water heating end-use.

⁷⁶ Default assumption for unknown fuel is based on EIA Residential Energy Consumption Survey (RECS) 2009 for Midwest Region, data for the state of Missouri, 'HC8.9 Water Heating in Midwest Region.xls'.

⁷⁷ DOE final rule discusses recovery efficiency with an average around 0.76 for gas-fired storage water heaters and 0.78 for standard efficiency gas fired tankless water heaters up to 0.95 for the highest efficiency gas-fired condensing tankless water heaters. However, these numbers represent the range of new units, not the range of existing units in stock. Review of AHRI Directory suggests range of recovery efficiency ratings for new gas DHW units of 70-87%. Average of existing units is estimated at 78%.

⁷⁸ Water heating in multifamily buildings is often provided by a larger central boiler. This suggests that the average recovery efficiency is somewhere between a typical central boiler efficiency of 0.59 and the 0.75 for single family homes. An average efficiency of 0.67 is used for this analysis as a default for multifamily buildings.

WATER IMPACT DESCRIPTIONS AND CALCULATION

$$\Delta \text{Gallons} = ((\text{GPM}_{\text{base}} * \text{L}_{\text{base}} - \text{GPM}_{\text{low}} * \text{L}_{\text{low}}) * \text{Household} * \text{SPCD} * 365.25 / \text{SPH}) * \text{ISR} * (1 - \text{Leakage})$$

Variables as defined above

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

3.3.3 Heat Pump Water Heater

DESCRIPTION

This measure applies to the installation of a heat pump water heater (HPWH) in place of a standard electric water heater in a home. Savings are presented dependent on the heating system installed in the home due to the impact of the heat pump water heater on the heating and cooling loads.

This measure was developed to be applicable to the following program types: TOS, and NC.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure, the installed equipment must be an ENERGY STAR® heat pump water heater with a storage volume ≤ 55 gallons.⁷⁹

DEFINITION OF BASELINE EQUIPMENT

The baseline equipment is assumed to be a new, electric storage water heater meeting federal minimum efficiency standards⁸⁰ for units ≤55 gallons:

Draw Pattern	Federal Standard – Uniform Energy Factor ⁸¹
Very small	UEF = 0.8808 – (0.0008 * Rated Storage Volume in Gallons)
Low	UEF = 0.9254 – (0.0003 * Rated Storage Volume in Gallons)
Medium	UEF = 0.9307 – (0.0002 * Rated Storage Volume in Gallons)
High	UEF = 0.9349 – (0.0001 * Rated Storage Volume in Gallons)

The same draw pattern (very small, low, medium and high draw) should be used for both baseline and efficient units.

If unknown, assume a 50 gallon resistance tank baseline, at medium draw, therefore 0.9207 UEF.⁸²

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 13 years.⁸³

DEEMED MEASURE COST

Actual costs should be used where available. Default incremental cost values are provided in the table below.⁸⁴

Efficiency Range	Baseline Installed Cost	Efficient Installed Cost	Incremental Installed Cost
<2.6 UEF	\$1,032	\$2,062	\$1,030
≥2.6 UEF	\$1,032	\$2,231	\$1,199

LOADSHAPE

Water Heating RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta kWh_{ISR} = [(((1/EF_{BASE} - 1/EF_{EE}) * GPD * Household * 365.25 * \gamma_{Water} * (T_{OUT} - T_{In}) * 1.0)) / 3,412) + kWh_{cool} - kWh_{heat}] *$$

⁷⁹ Since the federal standard effectively requires a heat pump water heater for units over 55 gallons, this measure is limited to units ≤ 55 gallons.

⁸⁰ Minimum federal standard up to date as of 8/15/2024:

<https://www.ecfr.gov/current/title-10/chapter-II/subchapter-D/part-430>; ‘10 CFR Part 430 (up to date as of 8-15-2024).pdf’.

⁸¹ All Residential sized Federal Standards are from DOE Standard 10 CFR 430, Residential-Duty and Commercial Federal Standard are from DOE Standard 10 CFR 431.

⁸² Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 249.

⁸³ 2010 Residential Heating Products Final Rule Technical Support Document, U.S. DOE, Table 8.7.2.

⁸⁴ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, p. 250. Costs for <2.6 UEF are based upon averages from the NEEP Phase 3 Incremental Cost Study (‘NEEP Incremental Cost Study FINAL_061016.pdf’). The assumption for higher efficiency tanks is based upon averaged from NEEP Phase 4 Incremental Cost Study.

Where:

UEF_{BASE} = UEF of standard electric water heater according to federal standards
= If new unit draw pattern unknown, 0.9207⁸⁵.

UEF_{EE} = UEF of heat pump water heater
= Actual

GPD = Gallons per day of hot water use per person
= 17.6⁸⁶

Household = Average number of people per household

Household Unit Type ⁸⁷	Household
All	2.65 ⁸⁸
Custom	Actual Occupancy or Number of Bedrooms ⁸⁹

365.25 = Days per year
 γ_{Water} = Specific weight of water
= 8.33 pounds per gallon
 T_{OUT} = Tank temperature
= Actual, if unknown assume 125°F
 T_{IN} = Incoming water temperature from well or municipal system
= 58.4°F⁹⁰
1.0 = Heat capacity of water (1 Btu/lb*°F)
3,412 = Conversion factor from Btu to kWh
ISR = In-service rate. Actual, or if unknown, assume 100%⁹¹
 kWh_{cool} = Cooling savings from conversion of heat in home to water heat⁹²
= $(((1 - 1/\text{UEF}_{\text{EE}}) * \text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (T_{\text{OUT}} - T_{\text{IN}}) * 1.0) * \text{LF} * \text{WHF}_{\text{C}} * \text{LM}) / (\text{COP}_{\text{COOL}} * 3,412)] * \%_{\text{Cool}}$

Where:

LF = Location Factor
= 1.0 for HPWH installation in a conditioned space
= 0.0 for installation in an unconditioned space
= 0.81 if unknown⁹³
 WHF_{C} = Portion of reduced waste heat that results in cooling savings (if unknown, assume 53%)⁹⁴
 COP_{COOL} = COP of central air conditioner
= Actual, or if unknown, assume 2.8 COP⁹⁵

⁸⁵ [Federal Register :: Energy Conservation Program: Energy Conservation Standards for Consumer Water Heaters.](#)

⁸⁶ GPD based on 45.5 gallons of hot water per day per household and 2.59 people per household, from “Residential End Uses of Water Study 2013 Update,” by Deoreo, B., and P. Mayer, for the Water Research Foundation, 2014, https://www.awwa.org/Portals/0/AWWA/ETS/Resources/WaterConservationResidential_End_Uses_of_Water.pdf.

⁸⁷ If household type is unknown, as may be the case for TOS measures, then single family deemed value shall be used.

⁸⁸ Ameren Missouri Efficient Products Evaluation: PY2018, <https://www.efis.psc.mo.gov/Document/Display/15869>, page 32.

⁸⁹ Bedrooms are suitable proxies for household occupancy and may be preferable to actual occupancy due to turnover rates in residency and non-adult population impacts.

⁹⁰ National Weather Service. Average soil temperature at 40" depth during 2015 - 2023 data of six stations in Ameren Missouri service territory. https://www.weather.gov/ncrfc/LMI_SoilTemperatureDepthMaps.

⁹¹ Ameren Missouri Efficient Products Evaluation: PY2019, <https://www.efis.psc.mo.gov/Document/Display/15877>, page 140.

⁹² This algorithm calculates the heat removed from the air by subtracting the heat pump water heater electric consumption from the total water heating energy delivered. This is then adjusted to account for location of the heat pump unit and the coincidence of the waste heat with cooling requirements, the efficiency of the central cooling, and latent cooling demands.

⁹³ Wisconsin Focus on Energy 2023 Technical Reference Manual, https://assets.focusonenergy.com/production/inline-files/Focus_on_Energy_2023_TRM.pdf, page 787.

⁹⁴ Based on Ameren Missouri Efficient Products Evaluation PY2018, <https://www.efis.psc.mo.gov/Document/Display/15869>, page 32.

⁹⁵ Starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm $(-0.02 * \text{SEER}^2) + (1.12 * \text{SEER})$ (from Wassmer, M. (2003), “A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations,” (Masters thesis), University of Colorado at Boulder), converted to COP = EER/3.412 = 2.8COP.

LM = Latent multiplier to account for latent cooling demand ⁹⁶

Weather Basis (City based upon)	LM
St. Louis, MO	1.33

%Cool = Percentage of homes with central cooling

Home	%Cool
Cooling	100%
No Cooling	0%
Unknown	95% ⁹⁷

kWh_{heat} = Heating cost from conversion of heat in home to water heat (dependent on heating fuel)
= [(((1- 1/UEF_{EE}) * GPD * Household * 365.25 * γ_{Water} * (T_{OUT} – T_{In}) * 1.0) * LF * WHF_H *LM) / (COP_{HEAT} * 3,412)] * %ElectricHeat

Where:

WHF_H = Portion of reduced waste heat that results in increased heating load (if unknown, assume 43%) ⁹⁸
COP_{HEAT} = COP of electric heating system
= Actual, or if unknown, assume:

System Type	Age of Equipment	HSPF2 Estimate	ηHeat (COP Estimate) = (HSPF2/3.413)*0.85
Heat Pump (if age unknown assume 2006-2014)	Before 2006	5.8	1.44
	After 2006 - 2014	6.5	1.62
	2015 on	7.0	1.74
Resistance	N/A	N/A	1.00
Unknown ⁹⁹	N/A	N/A	1.28

%ElectricHeat = Percentage of homes with electric heat

Heating fuel	%ElectricHeat
Electric	100%
Natural Gas	0%
Unknown	35% ¹⁰⁰

SUMMER COINCIDENT PEAK DEMAND SAVINGS

ΔkW = ΔkWh * CF

Where:

ΔkWh = Electric energy savings, as calculated above
CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor

⁹⁶ The Latent Multiplier is used to convert the sensible cooling savings calculated to a value representing sensible and latent cooling loads. The values are derived from the methodology outlined in “Infiltration Factor Calculation Methodology” by Bruce Harley, Senior Manager, Applied Building Science, CLEAResult 11/18/2015, and are based upon an 8760 analysis of sensible and total heat loads using hourly climate data. (Ameren Missouri Efficient Products Evaluation PY2018, <https://www.efis.psc.mo.gov/Document/Display/15869>, page 32).

⁹⁷ Ameren Missouri PY2019 Residential Baseline Study (Saturation of non-low income homes with central cooling).

⁹⁸ Based on Ameren Missouri Efficient Products Evaluation PY2018, <https://www.efis.psc.mo.gov/Document/Display/15869>, page 31.

⁹⁹ Calculation assumes 35% Heat Pump and 65% Resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, <https://www.eia.gov/consumption/residential/data/2009/hc/hc6.9.xls> ; ‘hc6.9.xls’. Average efficiency of heat pump is based on assumption that 50% are units from before 2006 and 50% from 2006-2014. Program or evaluation data should be used to improve this assumption if available.

¹⁰⁰ Average (default) value of 35% electric space heating from 2009 Residential Energy Consumption Survey for Missouri. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then they should be used.

$$= 0.0000887318^{101}$$

NATURAL GAS SAVINGS

$$\Delta \text{Therms} = \frac{[(((1 - 1/\text{EF}_{\text{EE}}) * \text{GPD} * \text{Household} * 365.25 * \gamma_{\text{Water}} * (\text{T}_{\text{OUT}} - \text{T}_{\text{In}}) * 1.0) * \text{LF} * 0.43) / (\eta_{\text{Heat}} * 100,000)] * \% \text{GasHeat}}{\% \text{GasHeat}}$$

Where:

- ΔTherms = Heating cost from conversion of heat in home to water heat for homes with Natural Gas heat¹⁰²
- 100,000 = Conversion factor from Btu to therms
- η_{Heat} = Efficiency of heating system
= 71%¹⁰³
- $\% \text{GasHeat}$ = Percentage of homes with gas heat

Heating Fuel	%GasHeat
Electric	0%
Gas	100%
Unknown	65% ¹⁰⁴

Other factors as defined above

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

¹⁰¹ Based on Ameren Missouri TRM Volume 1 - Appendix G: "Table 1 – Residential End Use Category Monthly Shapes and Coincident Peak Factors" for residential water heating end-use.

¹⁰² This is the additional energy consumption required to replace the heat removed from the home during the heating season by the heat pump water heater. The variable kWh_heating (electric resistance) is that additional heating energy for a home with electric resistance heat (COP 1.0). This formula converts the additional heating kWh for an electric resistance home to the MMBtu required in a natural gas heated home, applying the relative efficiencies.

¹⁰³ This has been estimated assuming that natural gas central furnace heating is typical for Missouri residences (the predominant heating is gas furnace with 48% of Missouri homes (based on Energy Information Administration, 2009 Residential Energy Consumption Survey)). See reference <https://www.eia.gov/consumption/residential/data/2009/hc/hc6.9.xls> ; 'hc6.9.xls'. In 2000, 29% of furnaces purchased in Missouri were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years and so units purchased 15 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: ((0.29*0.92) + (0.71*0.8)) * (1-0.15) = 0.71.

¹⁰⁴ Based on data from Energy Information Administration, 2009 Residential Energy Consumption Survey, see <https://www.eia.gov/consumption/residential/data/2009/hc/hc6.9.xls>; 'hc6.9.xls'.

3.3.4 Pipe Insulation

DESCRIPTION

This measure applies to the addition of insulation to uninsulated domestic hot water (DHW) pipes. The measure assumes the pipe wrap is installed on the first length of both the hot and cold pipes up to the first elbow or the first three feet of pipe length, whichever is longer. This is the most cost-effective section to insulate since, close to the tank, the water pipes act as an extension of the hot water tank, which acts as a heat trap. Insulating this section helps to reduce standby losses.

This measure was developed to be applicable to the following program types: DI, and RF

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient condition is a domestic hot or cold water pipe with pipe wrap installed that has an R value that meets program requirements.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is an uninsulated, domestic hot or cold water pipe.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 12 years.¹⁰⁵

DEEMED MEASURE COST

The measure cost is the actual cost of material and installation. If the actual cost is unknown, assume a default cost of \$7.10¹⁰⁶ per linear foot, including material and installation.

LOADSHAPE

Water Heating RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Custom calculation below for electric systems, otherwise assume 24.7 kWh per 6 linear feet of ¾ in, R-4 insulation or 35.4 kWh per 6 linear feet of 1 in, R-6 insulation:

$$\Delta \text{kWh} = \% \text{ElectricDHW} * ((C_{\text{Base}}/R_{\text{Base}} - C_{\text{EE}}/R_{\text{EE}}) * L * \Delta T * \text{Hours}) / (\eta_{\text{DHW}_{\text{Elec}}} * 3,412) * \text{ISR} * (1 - \text{Leakage})$$

Where:

%ElectricDHW = proportion of water heating supplied by electric resistance heating

DHW fuel	%ElectricDHW
Electric	100%

C_{Base} = Circumference (ft) of uninsulated pipe
= Diameter (in) * $\pi/12$
= Actual or, if unknown, assume 0.144” based on a weighted average of 80% 0.50-inch diameter pipe and 20% 0.75-inch diameter pipe.

¹⁰⁵ 2014 Database for Energy-Efficiency Resources (DEER), Version 2014, “Cost Values and Summary Documentation,” California Public Utilities Commission, January 2014. Average of values for electric DHW (13 years) and gas DHW (11 years).

¹⁰⁶ Average cost of R-5 pipe wrap installation from the National Renewable Energy Laboratory’s National Residential Efficiency Measures Database, <https://remdb.nrel.gov/>.

- R_{Base}

= Thermal resistance coefficient (hr-°F-ft²)/Btu) of uninsulated pipe
= 1.0¹⁰⁷
- C_{EE}

= Circumference (ft) of insulated pipe
= Diameter (in) * π/12
= Based on actual pipe diameter and insulation thickness; if unknown, assume 0.55” pipe diameter based on a weighted average of 80% 0.50-inch diameter pipe and 20% 0.75-inch diameter pipe and 0.5” insulation thickness – using both assumed values results in C_{EE} of (0.55 + (0.5 * 2)) * π / 12 = 0.4058
. For instance, for a pipe insulated with 3/4 in, R-4 wrap, assume 0.524 ft for a 0.46 in diameter pipe ((0.75 + 1/2 + 1/2) * π/12)
- R_{EE}

= Thermal resistance coefficient (hr-°F-ft²)/Btu) of insulated pipe
= 1.0 + R value of insulation
= Actual

L

= Length of pipe from water heating source covered by pipe wrap (ft)
= Actual

ΔT

= Average temperature difference (°F) between supplied water and outside air
= Actual or if unknown, assume 58.9°F¹⁰⁸ for low income programs or 60°F¹⁰⁹ for other programs.

Hours

= Hours per year
= 8,766

ηDHW_{Elec}

= Recovery efficiency of electric hot water heater
= 0.98¹¹⁰

3,412

= Conversion factor from Btu to kWh

ISR

= In Service Rate
=Actual, or if unknown, dependent on program delivery method as listed in table below
- | Program | ISR |
|-----------------------|--------------------|
| Income Eligible, PAYS | 96% ¹¹¹ |
- Leakage

= Percent homes outside service territory
- | Program | Leakage |
|-----------------------|---------|
| Income Eligible, PAYS | 0% |
- SUMMER COINCIDENT PEAK DEMAND SAVINGS
- ΔkW = ΔkWh * CF
- Where:
- ΔkWh

= Electric energy savings, as calculated above.

CF

= Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0000887318
- NATURAL GAS SAVINGS
- Custom calculation below for gas DHW systems, otherwise assume 1.1 therms per 6 linear feet of ¾ in, R-4 insulation or 1.5 therms per 6 linear feet of 1 in, R-6 insulation:
- ΔTherms = (1 - %ElectricDHW) * ((C_{Base}/R_{Base} - C_{EE}/R_{EE}) * L * ΔT * Hours)/(ηDHW_{Gas} * 100,000)
- ¹⁰⁷ “Measures and Assumptions for Demand Side Management (DSM) Planning; Appendix C Substantiation Sheets,” Navigant, April 2009, https://www.oeb.ca/oeb/_Documents/EB-2008-0346/Navigant_Appendix_C_substantiation_sheet_20090429.pdf; ‘Navigant_Appendix_C_substantiation_sheet_20090429.pdf’, page C-77.
- ¹⁰⁸ Ameren Missouri Community Savers Evaluation PY2018, page 24.
- ¹⁰⁹ Assumes 125°F water leaving the hot water tank and average basement temperature of 65°F.
- ¹¹⁰ Electric water heater recovery efficiency from AHRI database: <http://www.ahridirectory.org/ahridirectory/pages/home.aspx>.
- ¹¹¹ Ameren Missouri Community Savers Evaluation PY2018, page 24.
- 2025 MEEIA 4 Plan
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- Page 28

Where:

$\eta_{DHW_{Gas}}$ = Recovery efficiency of gas hot water heater
= 0.78¹¹²
100,000 = Conversion factor from Btu to therms
Other variables as defined above.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

¹¹² Review of AHRI directory suggests range of recovery efficiency ratings for new gas DHW units of 70-87%. Average of existing units is estimated at 78%.

3.4 HVAC

3.4.1 Smart Thermostat

DESCRIPTION

This measure characterizes the household energy savings from the installation of a new thermostat(s) for reduced heating and cooling consumption through a configurable schedule of temperature setpoints (like a programmable thermostat) *and* automatic variations to that schedule to better match HVAC system runtimes to meet occupant comfort needs. These schedules may be defaults, established through user interaction, and be changed manually at the device or remotely through a web or mobile app. Automatic variations to that schedule could be driven by local sensors and software algorithms and/or through connectivity to an internet software service. Data triggers to automatic schedule changes might include, for example: occupancy/activity detection, arrival & departure of conditioned spaces, optimization based on historical or population-specific trends, or weather data and forecasts.¹¹³ This class of products and services are relatively new, diverse, and rapidly changing. Generally, the savings expected for this measure aren't yet established at the level of individual features, but rather at the system level and how it performs overall. Like programmable thermostats, it is not suitable to assume that heating and cooling savings follow a similar pattern of usage and savings opportunity, so this measure treats these savings independently. This is a very active area of ongoing study to better map features to savings value and establish standards of performance measurement based on field data so that a standard of efficiency can be developed.¹¹⁴ That work is not yet complete but does inform the treatment of some aspects of this characterization and recommendations. Energy savings are applicable at the household level; installation of multiple smart thermostats per home does not accrue additional savings.

This measure was developed to be applicable to the following program types: TOS, NC, RF, and DI.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

This measure involves replacement of a manual-only or programmable thermostat with one that has the default-enabled capability or the automatic capability to establish a schedule of temperature setpoints according to driving device inputs above and beyond basic time and temperature data of conventional programmable thermostats. As summarized in the description, this category of products and services is broad and rapidly advancing with regard to thermostat capability, usability, and sophistication. At a minimum, a qualifying thermostat must be capable of two-way communication¹¹⁵ and exceed the typical performance of manual and conventional programmable thermostats through the automatic or default capabilities described above.

DEFINITION OF BASELINE EQUIPMENT

The baseline is either the actual thermostat type (manual or programmable), if known,¹¹⁶ or an assumed mix of both types based upon information available from evaluations or surveys that represent the population of program participants. This mix may vary by program, but as a default, 44% programmable and 56% manual thermostats may be assumed.¹¹⁷

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life for smart thermostats is assumed to be 11 years.¹¹⁸

¹¹³ For example, the capabilities of products and added services that use ultrasound, infrared, or geofencing sensor systems, automatically develop individual models of a home's thermal properties through user interaction. The thermostats optimize system operation based on equipment type and performance traits, such as using n weather forecasts, to demonstrate the type of automatic schedule change functionality that apply to this measure characterization.

¹¹⁴ The ENERGY STAR® program discontinued its support for basic programmable thermostats effective 12/31/09 and is presently developing a new specification for "Residential Climate Controls."

¹¹⁵ This measure recognizes that field data may be available, through the thermostat's two-way communication capability, to more accurately establish efficiency criteria and make savings calculations. It is recommended that program implementations incorporate this data into their planning and operation activities to improve understanding of the measure to manage risks and enhance savings results.

¹¹⁶ If the actual thermostat is programmable and is found to be used in override mode or otherwise is effectively being operated like a manual thermostat, then the baseline may be considered to be a manual thermostat.

¹¹⁷ Value for blend of baseline thermostats comes from an Illinois potential study conducted by ComEd in 2013; Opinion Dynamics Corporation, "ComEd Residential Saturation/End Use, Market Penetration & Behavioral Study," Appendix 3: Detailed Mail Survey Results, April 2013, p. 34.

¹¹⁸ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 204. Based on 2017 Residential Smart Thermostat Workpaper, prepared by SCE and Nest for SCE (Work Paper SCE17HC054, Revision #0, https://www.peakload.org/assets/SCE17HC054.0_Residential_Sma.pdf; 'SCE17HC054.0_Residential_Sma.pdf'). Estimate ability of smart systems to continue providing savings after disconnection and conduct statistical survival analysis which yields 9.2-13.8 year range.

DEEMED MEASURE COST

For DI and other programs where installation services are provided, the actual material, labor, and other costs should be referenced, and incremental cost of the smart thermostat is equal to the actual total smart thermostat material, labor, and other costs, minus the \$50 baseline thermostat cost.

For retail or other program types where actual smart thermostat costs are known, the incremental cost of the smart thermostat is equal to the total cost of the smart thermostat, minus the \$50 baseline thermostat cost.¹¹⁹

If actual costs are unknown, then the incremental cost for the smart thermostat is assumed to be \$79.¹²⁰

LOADSHAPE

Cooling RES

Heating RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Electrical savings are a function of both heating and cooling energy usage reductions. For heating, this is a function of the percent of electric heat (heat pumps) and fan savings in the case of a natural gas furnace.

$$\Delta kWh = \Delta kWh_{heating} + \Delta kWh_{cooling}$$
$$\Delta kWh_{heating} = \%ElectricHeat * HeatingConsumption_{Electric} * HF * HeatingReduction * Eff_ISR + (\Delta Therms * Fe * 29.3)$$
$$\Delta kWh_{cool} = \%AC * ((EFLH_{cool} * Capacity_{Cool} * 1/SEER2)/1000) * CoolingReduction * Eff_ISR$$

Where:
%ElectricHeat = Percentage of heating savings assumed to be electric

Heating fuel	%ElectricHeat
Electric	100%
Natural Gas	0%
Unknown	33% ¹²¹

HeatingConsumption_{Electric} = If heating equipment characteristics are known, equals ((EFLH_{heat} * Capacity_{Heat} * 1/HSPF2)/1000); otherwise, estimate of annual household heating consumption for electrically heated homes.¹²²

Weather Basis (Ameren Missouri Average)	Elec_Heating_Consumption (kWh) ¹²³		
	Electric Resistance	Electric Heat Pump	Unknown Electric
SF or MF	14,202	8,355	11,456

EFLH_{heat} = Equivalent Full Load Heating Hours: ¹²⁴

¹¹⁹ Actual costs include any one-time software integration, annual software maintenance, and/or individual device energy feature fees.

¹²⁰ Market prices vary considerably in this category, generally increasing with thermostat capability and sophistication. The core suite of functions required by this measure's eligibility criteria are available on units readily available in the market roughly in the range of \$100 and \$150, excluding the availability of time or market-limited wholesale or volume pricing. Analysis of the 2021 Pricing data from AIC's Retail Products Program finds an average retail cost of \$129 for Advanced Thermostats. The assumed cost for the baseline equipment (blend of manual and programmable thermostats) is \$50 which leads to an incremental cost of \$79 for the measure. Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 204.

¹²¹ Ameren Missouri Efficient Products Evaluation: PY2020, <https://www.efis.psc.mo.gov/Document/Display/13830>, page 40.

¹²² Ameren Missouri Efficient Products Evaluation PY2018 workpapers. For Comprehensive Envelope (CompE) Measures, the ratio of MF effective full load hours (1496) to the Opinion Dynamic recommendation for Comprehensive Envelope full load hours (509) was used to scale heating consumption values.

¹²³ Ibid.

¹²⁴ Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations.

Weather Basis (Ameren Missouri Average)	EFLHheat (Hours)
SF or MF	1496

CapacityHeat = Capacity of air cooling system (Btu/hr) (Note: One ton is equal to 12,000 Btu/hr.)
= Actual
HSPF2 = the cooling equipment’s Seasonal Energy Efficiency Ratio rating (kBtu/kWh)
= Actual
HF = Household factor, to adjust heating consumption for non-single-family households.

Household Type	HF
Single-Family	100%
Multi-Family	65% ¹²⁵
If heating equipment characteristics are referenced to calculate HeatingConsumption _{Electric}	100%
Actual	Custom ¹²⁶

HeatingReduction = Assumed percentage reduction in total household heating energy consumption due to smart thermostat

Existing Thermostat Type	Heating_Reduction ¹²⁷
Manual	8.8%
Programmable	5.6%
Blended Average	6.67%

Eff_ISR = Effective In-Service Rate, the percentage of thermostats installed and configured effectively for 2-way communication
= Actual, or if unknown, for Efficient Products, use 98.8%.¹²⁸, and for other programs, if using default savings, use 100%.¹²⁹
ΔTherms = Therm savings if natural gas heating system
= See calculation in natural gas section below
F_e = Furnace fan energy consumption as a percentage of annual fuel consumption
= 3.14%¹³⁰
29.3 = kWh per therm
%AC = Fraction of customers with thermostat-controlled air-conditioning

Thermostat control of air conditioning?	%AC
Yes	100%
No	0%

¹²⁵ Multifamily household heating consumption relative to single family households is affected by overall household square footage and exposure to the exterior. This 65% reduction factor is applied to multifamily homes with electric resistance, based on professional judgment that average household size, and heat loads of multifamily households are smaller than single family homes.

¹²⁶ Program-specific household factors may be utilized on the basis of sufficiently validated program evaluations.

¹²⁷ These values represent adjusted baseline savings values for different existing thermostats, as presented in Navigant’s IL TRM Workpaper on Impact Analysis from Preliminary Gas savings findings (page 28) https://www.ilsag.info/wp-content/uploads/SAG_files/Meeting_Materials/2015/December_2015_Meetings/Presentations/Smart_Tstat_Preliminary_Gas_Impact_Findings_2015-12-08_to_IL_SAG.pdf; ‘Smart_Tstat_Preliminary_Gas_Impact_Findings_2015-12-08_to_IL_SAG.pdf’. The unknown assumption is calculated by multiplying the savings for manual and programmable thermostats by their respective share of baseline. Further evaluation and regular review of this key assumption is encouraged. Ameren Missouri Efficient Products Evaluation PY2017, <https://www.efis.psc.mo.gov/Document/Display/14206>, page 47.

¹²⁸ Ameren Missouri Efficient Products Evaluation PY2019, <https://www.efis.psc.mo.gov/Document/Display/15877>, page 140.

¹²⁹ As a function of the method for determining savings impact of these devices, in-service rate effects are already incorporated into the savings value for heating reduction above.

¹³⁰ F_e is not one of the AHRI certified ratings provided for residential furnaces but can be reasonably estimated from a calculation based on the certified values for fuel energy (Ef in MMBTU/yr) and Eae (kWh/yr). An average of a 300- record sample (non-random) out of 1495 was 3.14%. This is appropriately ~50% greater than the ENERGY STAR® version 3 criteria for 2% F_e. See ‘Programmable Thermostats Furnace Fan Analysis.xlsx’ for reference.

Thermostat control of air conditioning?	%AC
Unknown	Actual population data, or 91% ¹³¹

EFLH_{cool} = Equivalent full load hours of air conditioning:

Weather Basis (Ameren Missouri Average)	EFLH _{cool} (Hours)
SF or MF	869 ¹³²

CapacityCool = Capacity of air cooling system (Btu/hr) (Note: One ton is equal to 12,000 Btu/hr.)
= Actual installed - If actual size unknown, assume 36,000 Btu/h
SEER2 = the cooling equipment’s Seasonal Energy Efficiency Ratio rating (kBtu/kWh)
= Use actual SEER2 rating where it is possible to measure or reasonably estimate. If unknown assume 13.4 SEER2.¹³³
1/1000 = kBtu per Btu
CoolingReduction = Assumed percentage reduction in total household cooling energy consumption due to installation of smart thermostat
= If programs are evaluated during program deployment then custom savings assumptions should be applied.
Otherwise use:
= 8.0%¹³⁴

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW = \Delta kWh_{Cooling} * CF$

Where:

$\Delta kWh_{Cooling}$ = Electric energy savings for cooling, calculated above
CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0009474181¹³⁵

NATURAL GAS ENERGY SAVINGS

$\Delta Therms = \%FossilHeat * HeatingConsumption_{Gas} * HF * HeatingReduction * Eff_{ISR}$

Where:

$\%FossilHeat$ = Percentage of heating savings assumed to be natural gas

Heating fuel	%FossilHeat
Electric	0%
Natural Gas	100%
Unknown	67% ¹³⁶

HeatingConsumption_{Gas}

¹³¹ 91% of homes have central cooling in Missouri (based on 2009 Residential Energy Consumption Survey, see ‘RECS 2009 Air Conditioning_hc7.9.xls’), <https://www.eia.gov/consumption/residential/data/2009/>.
¹³² PY2019 evaluation report, <https://www.efis.psc.mo.gov/Document/Display/15876>, page 30
¹³³ Based on minimum federal standard.
¹³⁴ This assumption is based upon the review of many evaluations from other regions in the United States. Cooling savings are more variable than heating due to significantly more variability in control methods and potential population and product capability.
¹³⁵ Based on Ameren Missouri TRM Volume 1 - Appendix G: "Table 1 – Residential End Use Category Monthly Shapes and Coincident Peak Factors"for Residential Cooling.
¹³⁶ Ameren Missouri Efficient Products Evaluation: PY2020, <https://www.efis.psc.mo.gov/Document/Display/13830>, page 41.

= Estimate of annual household heating consumption for gas heated single-family homes.¹³⁷

Weather Basis (City based upon)	Gas_Heating_Consumption (Therms)
St Louis, MO	682

Other variables as provided above.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

¹³⁷ Values in table are based on average household heating load (834 therms) for Chicago based on Illinois furnace metering study ((Table E-1, Energy Efficiency/Demand Response Nicor Gas Plan Year 1: Research Report: Furnace Metering Study, Draft, <https://www.icc.illinois.gov/docket/P2012-0528/documents/201939/files/355536.pdf> ; ‘355536.pdf’, Navigant, August 1 2013) and adjusted for Missouri climate region values using the relative climate- normal HDD data with a base temp ratio of 60°F. This load value is then divided by standard assumption of existing unit efficiency of 83.5% (estimate based on 29% of furnaces purchased in Missouri were condensing in 2000 (based on data from GAMA, provided to Department of Energy) (see ‘Thermostat_ FLH and Heat Load Calcs.xls’). The resulting values are generally supported by data provided by Laclede Gas, which showed an average pre-furnace replacement consumption of 1009 therms for St Louis, and a post-replacement consumption of 909. Assuming a typical hot water consumption at 225 therms (using defaults from <http://energy.gov/eere/femp/energy-cost-calculator-electric-and-gas-water-heaters-0#output>), this indicates a heating load of 684-784 therms. Ameren Missouri Efficient Products Evaluation PY2017, <https://www.efis.psc.mo.gov/Document/Display/14206>, page 47.

3.4.2 Air Source Heat Pump Including Dual Fuel Heat Pumps

DESCRIPTION

An air source heat pump (ASHP) provides heating and/or cooling by moving heat between indoor and outdoor air. A cold climate air source heat pump (ccASHP) operates the same as a traditional ASHP, but is able to meet a home's full heating load at lower outdoor temperatures approaching 0°F. A dual fuel heat pump (DFHP) pairs an air source heat pump with a gas furnace such that the air source heat pump provides heating in mild weather, and as temperature drops the heat pump shuts off and the furnace provides heating. This measure may also apply to replacing a Central Air Conditioner with non-electric heating with an Air Source Heat Pump. In this case, only cooling savings (ER1, ER2, ROF) may be claimed using the ASHP cooling algorithm. This measure applies to central ducted systems and single zone split-systems with ductless indoor units that are capable of meeting a home's full cooling and heating demand.

This measure characterizes:

1. TOS, NC: The installation of a new residential sized ($\leq 65,000$ Btu/hr) air source heat pump that is more efficient than required by federal standards. This could relate to the replacement of an existing ASHP at the end of its useful life or the installation of a new ASHP in a new home.
2. EREP: The early removal of functioning electric heating and cooling systems from service, prior to its natural end of life, and replacement with a new high efficiency ASHP unit. For the early replacement baseline to apply, program participants or installation contractors must provide documentation asserting that the existing units operated when turned on—whether or not they provided cooling—and the existing unit brand name and model number must be documented; or, if the nameplate is not readable, a photograph of the unit in context and the nameplate specifically must be provided. If the SEER and/or HSPF of the existing unit are known, the baseline SEER and/or baseline HSPF should be the actual values of the unit replaced. If unknown, use the assumptions provided in the variable list below ($SEER2_{exist}$ and $HSPF2_{exist}$). If the operational status of the existing unit is unknown, use TOS assumptions.

This measure was developed to be applicable to the following program types: TOS, NC, and EREP.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

A new residential-sized ($\leq 65,000$ Btu/hr) air source heat pump with specifications to be determined by the program.

The heating capacity of the efficient heat pump should be within 90% to 120% of the capacity of the existing equipment, unless the trade ally can provide documentation confirming the existing system is oversized. It is recommended to collect the existing and new unit capacities to confirm that the heat pump has sufficient capacity to minimize use of backup electric resistance heating.

Using a dual fuel heat pump, which uses a gas furnace for heating at lower outside air temperatures, or a cold climate rated heat pump are two options to ensure minimal use of backup electric resistance heating.

DEFINITION OF BASELINE EQUIPMENT

A new residential-sized ($\leq 65,000$ Btu/hr) air source heat pump meeting federal standards.

The baseline for the TOS measure is the federal standard efficiency level; 14.3 SEER2 and 7.5 HSPF2, when replacing an existing air source heat pump; and 13.4 SEER2 and 3.41 HSPF when replacing a central air conditioner and electric resistance heating. Non-electric heating replaced with an air source heat pump can only claim cooling savings. Under the new federal standards, the M1 testing protocol was revised, resulting in new SEER and HSPF performance metrics, now called SEER2 and HSPF2. When quantifying energy savings, the SEER2 and HSPF2 metrics should be used for the existing, baseline, and new equipment. The following conversion formulas can be used to convert between efficiency metrics:

$$\begin{aligned} SEER2 &= SEER * 0.96 \\ HSPF2 &= HSPF * 0.87 \end{aligned}$$

Non-electric heating replaced with an air source heat pump can only claim cooling savings.

The baseline for the early replacement measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit and the new baseline as defined above for the remainder of the measure life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 18 years.¹³⁸
Remaining life of existing ASHP/CAC equipment is assumed to be 6 years¹³⁹ and 18 years for electric resistance.

DEEMED MEASURE COST

New Construction and Time of Sale: The actual installed cost of the ASHP (including any necessary electrical or distribution upgrades required) should be used minus the assumed installation cost of the baseline equipment (\$6,865 + \$600 per ton for a new baseline ASHP¹⁴⁰, \$2,011 for a new baseline 80% AFUE furnace or \$4,053 for a new 84% AFUE boiler¹⁴¹ and \$3,338 for new baseline Central AC replacement¹⁴²).

Early Replacement: The actual full installation cost of the ASHP (including any necessary electrical or distribution upgrades required) should be used. The assumed deferred cost (after the appropriate number of years described above) of replacing existing equipment with a new baseline unit is assumed to be \$7,722 + \$674 per ton for a new baseline ASHP, or \$2,296 for a new baseline 80% AFUE furnace or \$4,627 for a new 84% AFUE boiler and \$3,670 for new baseline Central AC replacement.¹⁴³ This future cost should be discounted to present value using a 2.31% nominal societal discount rate, based on the ten year average (1/1/2014 – 12/31/2023) of the 10 year Treasury bond yield rates.¹⁴⁴

If the install cost of the efficient ASHP is unknown, assume the following (note these costs are per ton of unit capacity);¹⁴⁵

Efficiency (SEER2)	Full Efficient ASHP Cost (including labor)
15.2	\$7,000 + \$600/ ton
16.2	\$7,286 + \$600/ ton
17.1	\$7,495 + \$600/ ton
18.1	\$7,720 + \$600/ ton
19.0+	\$7,946 + \$600/ ton

When a non-electric heating system is replaced with an ASHP, and the program administrator is only claiming energy savings from cooling (i.e., no heating savings are claimed due to the fuel switch), the incremental costs should be adjusted to reflect the proportion of claimed savings.

The incremental cost of the ASHP shall be adjusted by applying a factor that represents the ratio of the claimed cooling savings to the total potential energy savings (inclusive of both cooling and unclaimed heating savings). This adjustment ensures that the incremental cost used in cost-effectiveness testing is proportional to the benefits being claimed, thus avoiding a mismatch between costs and benefits.

Calculation Method:

1. Determine Total Potential Savings: Calculate the sum of the potential energy savings from both cooling and heating (even if heating savings are not being claimed).
2. Determine Claimed Savings: Identify the portion of energy savings that is being claimed, typically the cooling savings only.
3. Apply the Adjustment Factor: The adjustment factor is calculated as the ratio of claimed savings to total potential savings. This factor is then applied to the full incremental cost of the ASHP.

Adjusted Incremental Cost = (Claimed Savings (kWh) / Total Potential Savings (kWh)) * Full Incremental Cost

LOADSHAPE

¹³⁸ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007, <https://energizect.com/sites/default/files/documents/Measure%20Life%20Report%202007.pdf>; ‘Measure Life Report 2007.pdf’, page 1-3.

¹³⁹ Assumed to be one third of effective useful life.

¹⁴⁰ Full install ASHP costs are based upon data provided by Ameren. See ‘ASHP Costs_06242022’. Efficiency cost increment consistent with Cadmus “HVAC Program: Incremental Cost Analysis Update”, December 19, 2016 study results.

¹⁴¹ See ‘Technical Standard Document_APPENDIX_E.pdf’.

¹⁴² See ‘CAC Costs 09.02.2024.xlsx’.

¹⁴³ All baseline replacement costs are consistent with their respective measures and include inflation rate of 1.98%.

¹⁴⁴ ‘Societal_Discount_Rate_Calculation_08082024.xlsx’.

¹⁴⁵ Full install ASHP costs are based upon data provided by Ameren. See ‘ASHP Costs_06242022’. Efficiency cost increment consistent with Cadmus “HVAC Program: Incremental Cost Analysis Update”, December 19, 2016 study results.

Cooling RES
Heating RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

TOS:

$$\Delta kWh = [((EFLH_{cool} * Capacity_{cool} * (1/SEER2_{base} - 1/SEER2_{ee})) / 1000) + ((EFLH_{heat} * Capacity_{heat} * (1/HSPF2_{base} - 1/HSFP2_{ee})) / 1,000)] * ISR$$

Cooling only for Central Air Conditioning and Non-Electric Heating Backup

$$\Delta kWh = [((EFLH_{cool} * Capacity_{cool} * (1/SEER2_{base} - 1/SEER2_{ee})) / 1000) * ISR$$

EREP:¹⁴⁶

ΔkWh for remaining life of existing unit (1st 6 years for replacing an ASHP, 18 years for replacing electric resistance):

$$= [((EFLH_{cool} * Capacity_{cool} * (1/SEER2_{exist} - 1/SEER2_{ee})) / 1000) + ((EFLH_{heat} * Capacity_{heat} * (1/HSPF2_{exist} - 1/HSFP2_{ee})) / 1,000)] * ISR$$

Cooling only for Central Air Conditioning and Non-Electric Heating Backup

$$\Delta kWh = [((EFLH_{cool} * Capacity_{cool} * (1/SEER2_{exist} - 1/SEER2_{ee})) / 1000) * ISR$$

ΔkWh for remaining measure life (next 12 years if replacing an ASHP):

$$= [((EFLH_{cool} * Capacity_{cool} * (1/SEER2_{base} - 1/SEER2_{ee})) / 1000) + ((EFLH_{heat} * Capacity_{heat} * (1/HSPF2_{base} - 1/HSFP2_{ee})) / 1,000)] * ISR$$

Cooling only for Central Air Conditioning and Non-Electric Heating Backup

$$\Delta kWh = [((EFLH_{cool} * Capacity_{cool} * (1/SEER2_{base} - 1/SEER2_{ee})) / 1000) * ISR$$

Where:

$EFLH_{cool}$ = Equivalent full load hours of air conditioning:¹⁴⁷

Weather Basis (Ameren Missouri Average)	$EFLH_{cool}$ (Hours)
SF or MF	869

$Capacity_{cool}$ = Cooling Capacity of Air Source Heat Pump (Btu/hr)

= Actual (1 ton = 12,000Btu/hr)

$SEER2_{exist}$ = Seasonal Energy Efficiency Ratio of existing cooling system (kBtu/kWh)

= Use actual SEER2 rating where it is possible to measure or reasonably estimate. If using rated efficiencies, derate the efficiency value based on the age of the existing equipment (up to a maximum of 30 years) to account for degradation over time.¹⁴⁸ If age is unknown, use 12 years.

= $SEER2 * (1-0.01)^{Age}$

If rated efficiency is unknown, use defaults provided below, which are already adjusted to account for age-related degradation:

¹⁴⁶ The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a first year savings (using the first equation) and then a “number of years to adjustment” and “savings adjustment” input, which would be the either the new base to efficient savings or the (existing to efficient savings).

¹⁴⁷ PY2019 HVAC Evaluation, <https://www.efis.psc.mo.gov/Document/Display/13830>, page 4.

¹⁴⁸ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 112. Justification for degradation factors can be found on page 14 of ‘AIC HVAC Metering Study Memo FINAL 2_28_2018.docx’. Estimate efficiency as (Rated Efficiency * (1-0.01)^Equipment Age).

Existing Cooling System	SEER2 _{exist} ¹⁴⁹
Air Source Heat Pump	6.91
Central AC	6.53
No central cooling ¹⁵⁰	Let ‘1/SEER2 _{exist} ’ = 0

- SEER2_{base}

= Seasonal Energy Efficiency Ratio of baseline Air Source Heat Pump (kBtu/kWh)¹⁵¹
= 14.3 SEER2¹⁵² when replacing an ASHP
= SEER2 13.4 when replacing a CAC
- SEER2_{ee}

= Seasonal Energy Efficiency Ratio of efficient Air Source Heat Pump (kBtu/kWh)
= Actual
- EFLH_{heat}

= Equivalent full load hours of heating:¹⁵³

Weather Basis (Ameren Missouri Average)	EFLH _{heat} (Hours)
SF or MF	1496 for ASHP, 1119 for DFHP, and 1769 ¹⁵⁴ for ccAHSP

- Capacity_{heat}

= Heating Capacity of Air Source Heat Pump (Btu/hr)
= Actual (1 ton = 12,000Btu/hr)
- HSPF2_{exist}

= Heating Seasonal Performance Factor of existing heating system (kBtu/kWh)
= Use actual HSPF2 rating where it is possible to measure or reasonably estimate. If using rated efficiencies, derate the efficiency value based on the age of the existing equipment (up to a maximum of 30 years) to account for degradation over time.¹⁵⁵ If age is unknown, use 12 years.
= HSPF2 * (1-0.01)^{Age}
If rated efficiency is unknown, use defaults provided below, which should not be further adjusted to account for age-related degradation:

Existing Heating System	HSPF _{exist}
Air Source Heat Pump	4.91 ¹⁵⁶
Electric Resistance	3.41 ¹⁵⁷

- HSPF2_{base}

= Heating Seasonal Performance Factor of baseline Air Source Heat Pump (kBtu/kWh)¹⁵⁸
= 7.5 HSPF2¹⁵⁹
- HSFP2_{ee}

= Heating Seasonal Performance Factor of efficient Air Source Heat Pump (kBtu/kWh)
= Actual

¹⁴⁹ ASHP existing efficiency assumes degradation and is sourced from the Ameren Missouri Heating and Cooling Program Impact and Process Evaluation: Program Year 2015. CAC assumed to follow the same trend in degradation as the ASHP: 9.12 SEER nameplate to 7.2 (6.91 SEER2) operations SEER represents degradation to 78.9% of nameplate. 78.9% of 8.6 SEER CAC nameplate gives an operational SEER of 6.8 (6.53 SEER2).

¹⁵⁰ If there is no central cooling in place but the incentive encourages installation of a new ASHP with cooling, the added cooling load should be subtracted from any heating benefit.

¹⁵¹ SEER to SEER2 conversion factor: SEER2 = SEER * 96%. Conversion factor for SEER to SEER2 is used when converting an existing system that is rated in SEER to SEER2. This is to meet the DOE M1 CFR Standard beginning January 1, 2023. The efficiency levels of the existing, baseline, and efficient case must be expressed in SEER2 terms before applying formulas.

¹⁵² Based on minimum federal standard effective 1/1/2023: 10 CFR 430.32(c)(5).

¹⁵³ Ameren Missouri HVAC Evaluation PY2017, <https://www.efis.psc.mo.gov/Document/Display/14208>, page 37.

¹⁵⁴ Evaluation – Opinion Dynamics review PY2022. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) for St. Louis and technological differences between traditional and cold climate heat pumps, which are capable of meeting whole home heating requirements at lower temperatures than traditional heat pumps, resulting in increased effective full load operating hours.

¹⁵⁵ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 112. Justification for degradation factors can be found on page 14 of ‘AIC HVAC Metering Study Memo FINAL 2_28_2018.docx’. Estimate efficiency as (Rated Efficiency * (1-0.01)^{Equipment Age}).

¹⁵⁶ Ibid., page 110.

¹⁵⁷ Electric resistance has a COP of 1.0 which equals 1/0.293 = 3.41 HSPF.

¹⁵⁸ HSPF to HSPF2 conversion factor: HSPF2 = HSPF * 87%. Conversion factor for HSPF to HSPF2 is used when converting an existing system that is rated in HSPF to HSPF2. This is to meet the DOE M1 CFR Standard beginning January 1, 2023. The efficiency levels of the existing, baseline, and efficient case must be expressed in HSPF2 terms before applying formulas.

¹⁵⁹ Based on minimum federal standard effective 1/1/2023: 10 CFR 430.32(c)(5).

ISR = In-service rate. Actual, or if unknown, assume 100%¹⁶⁰

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$$\Delta kW = \Delta kWh_{Cooling} * CF$$

Where:

$\Delta kWh_{Cooling}$ = Electric energy savings for cooling, calculated above
CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0009474181

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

¹⁶⁰ Ameren Missouri HVAC Evaluation: PY2020, <https://www.efis.psc.mo.gov/Document/Display/13831>, page 53.

3.4.3 Duct Sealing and Duct Repair

DESCRIPTION

This measure describes evaluating the savings associated with performing duct sealing to the distribution system of homes with central cooling and/or a ducted heating system. While sealing ducts in conditioned space can help with control and comfort, energy savings are largely limited to sealing ducts in unconditioned space where the heat loss is to outside the thermal envelope. Therefore, for this measure to be applicable at least 30% of ducts should be within unconditioned space (e.g., attic with floor insulation, vented crawlspace, unheated garages; basements should be considered conditioned space).

Three methodologies for estimating the savings associate from sealing the ducts are provided.

1. **Modified Blower Door Subtraction** – this technique is described in detail on p. 44 of the Energy Conservatory Blower Door Manual; <http://dev.energyconservatory.com/wp-content/uploads/2014/07/Blower-Door-model-3-and-4.pdf>. It involves performing a whole house depressurization test and repeating the test with the ducts excluded.
2. **Duct Blaster Testing** - as described in RESNET Test 803.7: <https://energyconservatory.com/wp-content/uploads/2014/09/RESNET-Standards-Chapter-8.pdf>. This involves using a blower door to pressurize the house to 25 Pascals and pressurizing the duct system using a duct blaster to reach equilibrium with the inside. The air required to reach equilibrium provides a duct leakage estimate.
3. **Deemed Savings per Linear Foot** – this method provides a deemed conservative estimate of savings and should only be used where performance testing described above is not possible.

This measure was developed to be applicable to the following program type: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient condition is sealed duct work throughout the unconditioned space in the home.

DEFINITION OF BASELINE EQUIPMENT

The existing baseline condition is leaky duct work with at least 30% of the ducts within the unconditioned space in the home.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The assumed lifetime of this measure is 20 years.¹⁶¹

DEEMED MEASURE COST

The actual duct sealing measure cost should be used.

LOADSHAPE

HVAC RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Methodology 1: Modified Blower Door Subtraction

- a. Determine Duct Leakage rate before and after performing duct sealing:

$$\text{Duct Leakage (CFM50}_{\text{DL}}) = (\text{CFM50}_{\text{Whole House}} - \text{CFM50}_{\text{EnvelopeOnly}}) * \text{SCF}$$

Where:

$\text{CFM50}_{\text{Whole House}}$ = Standard Blower Door test result finding Cubic Feet per Minute at 50 Pascal pressure differentials

¹⁶¹ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, June 2007, https://www.caetrm.com/media/reference-documents/HVAC_Ltg_measure_life_GDS_2007.pdf; 'HVAC_Ltg_measure_life_GDS_2007.pdf', page 1-3.

CFM50_{Envelope Only} = Blower Door test result finding Cubic Feet per Minute at 50 Pascal pressure differentials with all supply and return registers sealed

SCF = Subtraction Correction Factor to account for underestimation of duct leakage due to connections between the duct system and the home. Determined by measuring pressure with respect to the building in the sealed duct system, with the building pressurized to 50 Pascals with respect to the outside. Use the following look up table provided by energy conservatory to determine the appropriate subtraction correction factor:

House to Duct Pressure	Subtraction Correction Factor	House to Duct Pressure	Subtraction Correction Factor
50	1.00	30	2.23
49	1.09	29	2.32
48	1.14	28	2.42
47	1.19	27	2.52
46	1.24	26	2.64
45	1.29	25	2.76
44	1.34	24	2.89
43	1.39	23	3.03
42	1.44	22	3.18
41	1.49	21	3.35
40	1.54	20	3.54
39	1.60	19	3.74
38	1.65	18	3.97
37	1.71	17	4.23
36	1.78	16	4.51
35	1.84	15	4.83
34	1.91	14	5.20
33	1.98	13	5.63
32	2.06	12	6.12
31	2.14	11	6.71

Calculate duct leakage reduction, convert to CFM25_{DL},¹⁶² and factor in Supply and Return Loss Factors:

Duct Leakage Reduction (Δ CFM25_{DL}) = (PreCFM50_{DL} – PostCFM50_{DL}) * 0.64 * (SLF + RLF)

Where:

0.64 = Converts CFM50_{DL} to CFM25_{DL}¹⁶³

SLF = Supply Loss Factor¹⁶⁴

= % leaks sealed located in Supply ducts * 1

Default = 0.5¹⁶⁵

¹⁶² 25 Pascals is the standard assumption for typical pressures experienced in the duct system under normal operating conditions.

¹⁶³ To convert CFM50 to CFM25, multiply by 0.64 (inverse of the “Can’t Reach Fifty” factor for CFM25; see Energy Conservatory Blower Door Manual).

¹⁶⁴ Assumes that for each percent of supply air loss there is one percent annual energy penalty. This assumes supply leaks are direct losses to the outside and are not recaptured back to the house. This could be adjusted downward to reflect regain of usable energy to the house from duct leaks. For example, during the winter some of the energy lost from supply leaks in a crawlspace will probably be regained back to the house (sometimes 1/2 or more may be regained). More information provided in “Appendix E Estimating HVAC System Loss From Duct Airtightness Measurements” from Energy Conservatory Blower Door Manual.

¹⁶⁵ Assumes 50% of leaks are in supply ducts.

RLF = Return Loss Factor¹⁶⁶
= % leaks sealed located in Return ducts * 0.5
Default = 0.25¹⁶⁷

Calculate electric savings

$\Delta kWh = \Delta kWh_{Cooling} + \Delta kWh_{Heating}$

If the home has central cooling, the electric energy saved in annual cooling due to the duct sealing and repair is:

$\Delta kWh_{Cooling} = (\Delta CFM_{25_{DL}} / ((Capacity_{Cool} / 12,000 * 400)) * EFLH_{cool} * Capacity_{Cool}) / (1,000 * SEER2)$

Where:

$\Delta CFM_{25_{DL}}$ = Duct leakage reduction in CFM2 as calculated above
 $Capacity_{Cool}$ = Capacity of Air Cooling system (Btu/hr)
= Actual
12,000 = Converts Btu/H capacity to tons
400 = Conversion of Capacity to CFM (400CFM / ton)¹⁶⁸
 $EFLH_{cool}$ = Equivalent Full Load Cooling Hours:¹⁶⁹

Weather Basis (Ameren Missouri Average)	EFLHcool (Hours)
SF or MF	869

1,000 = Converts Btu to kBtu
SEER2 = Seasonal Energy Efficiency Ratio of Air Conditioning equipment (kBtu/kWh)
= Actual - If not available, following:¹⁷⁰

Age of Equipment	η_{Cool} Estimate
Before 2006	9.5
2006 - 2014	12.4
Central AC After 1/1/2015	12.4
Heat Pump After 1/1/2015	13.3

If the home is heated with electric heat (resistance or heat pump), the electric energy saved in annual heating due to the added insulation is:

$\Delta kWh_{Heating_{Electric}} = (\Delta CFM_{25_{DL}} / ((Capacity_{Heat} / 12,000 * 400)) * EFLH_{heat} * Capacity_{Heat}) / (COP * 3,412)$

Where:

$Capacity_{Heat}$ = Heating output capacity (Btu/hr) of electric heat
= Actual
 $EFLH_{heat}$ = Equivalent Full Load Heating Hours:¹⁷¹

¹⁶⁶ Assumes that for each percent of return air loss there is a half percent annual energy penalty. Note that this assumes that return leaks contribute less to energy losses than do supply leaks. This value could be adjusted upward if there was reason to suspect that the return leaks contribute significantly more energy loss than “average” (e.g., pulling return air from a super-heated attic), or can be adjusted downward to represent significantly less energy loss (e.g., pulling return air from a moderate temperature crawl space). More information provided in “Appendix E Estimating HVAC System Loss From Duct Airtightness Measurements” from Energy Conservatory Blower Door Manual.

¹⁶⁷ Assumes 50% of leaks are in return ducts.

¹⁶⁸ This conversion is an industry rule of thumb. E.g., see <https://www.brinco.com/2016/02/04/is-there-a-rule-of-thumb-that-i-can-use-that-would-tell-me-how-many-cfms-an-ac-would-need-per-ton-of-cooling-capacity/>.

¹⁶⁹ <https://www.efis.psc.mo.gov/Document/Display/15876>, page 30

¹⁷⁰ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. Note all ratings have been converted to SEER2 equivalents – since the new rating better reflects the actual efficiency of the units.

¹⁷¹ Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations.

Weather Basis (Ameren Missouri Average)	EFLHheat (Hours)
SF or MF	1496

COP = Efficiency in COP of Heating equipment
= Actual - If not available, use:

System Type	Age of Equipment	HSPF2 Estimate	η_{Heat} (COP Estimate) = (HSPF2/3.413)*0.85
Heat Pump (if age unknown assume 2006-2014)	Before 2006	5.8	1.44
	After 2006 - 2014	6.5	1.62
	2015 on	7.0	1.74
Resistance	N/A	N/A	1.00
Unknown ¹⁷²	N/A	N/A	1.28

3412 = Converts Btu to kWh

If the home is heated with natural gas, the electric energy saved in annual heating due to the added insulation is:

$\Delta \text{kWh}_{\text{HeatingGas}} = (\Delta \text{Therms} * F_e * 29.3)$

Where:

ΔTherms = Therm savings as calculated in Natural Gas Savings
 F_e = Furnace fan energy consumption as a percentage of annual fuel consumption
= 3.14%¹⁷³
29.3 = kWh per therm

Methodology 2: Duct Blaster Testing

$\Delta \text{kWh} = \Delta \text{kWh}_{\text{Cooling}} + \Delta \text{kWh}_{\text{Heating}}$
 $\Delta \text{kWh}_{\text{Cooling}} = ((\text{Pre_CFM25} - \text{Post_CFM25}) / (\text{CapacityCool} / 12,000 * 400) * \text{EFLHcool} * \text{CapacityCool}) / (1,000 * \text{SEER2})$
 $\Delta \text{kWh}_{\text{HeatingElectric}} = ((\text{Pre_CFM25} - \text{Post_CFM25}) / (\text{CapacityHeat} / 12,000 * 400) * \text{EFLHheat} * \text{CapacityHeat}) / (\text{COP} * 3,412)$
 $\Delta \text{kWh}_{\text{HeatingGas}} = \Delta \text{Therms} * F_e * 29.3$

Where:

Pre_CFM25 = Duct leakage in CFM25 as measured by duct blaster test before sealing
Post_CFM25 = Duct leakage in CFM25 as measured by duct blaster test after sealing
All other variables as provided above

Methodology 3: Deemed Savings¹⁷⁴

$\Delta \text{kWh} = \Delta \text{kWh}_{\text{Cooling}} + \Delta \text{kWh}_{\text{HeatingElectric}} + \Delta \text{kWh}_{\text{HeatingGas}}$
 $\Delta \text{kWh}_{\text{cooling}} = \text{CoolSavingsPerUnit} * \text{DuctLength}$
 $\Delta \text{kWh}_{\text{HeatingElectric}} = \text{HeatSavingsPerUnit} * \text{DuctLength}$
 $\Delta \text{kWh}_{\text{HeatingGa}} = \Delta \text{Therms} * F_e * 29.3$

¹⁷² Calculation assumes 35% Heat Pump and 65% Resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, <https://www.eia.gov/consumption/residential/data/2009/hc/hc6.9.xls>; 'hc6.9.xls'. Average efficiency of heat pump is based on assumption that 50% are units from before 2006 and 50% from 2006-2014. Program or evaluation data should be used to improve this assumption if available.

¹⁷³ F_e is not one of the AHRI certified ratings provided for residential furnaces but can be reasonably estimated from a calculation based on the certified values for fuel energy (E_f in MMBtu/yr) and E_{ae} (kWh/yr). An average of a 300 record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the ENERGY STAR® version 3 criteria for 2% F_e .

¹⁷⁴ Savings per unit are based upon analysis performed by Cadmus for the 2011 Iowa Joint Assessment of Potential. It was based on 10% savings in system efficiency. This would represent savings from homes with significant duct work outside of the thermal envelope. With no performance testing or verification, a deemed savings value should be very conservative and therefore the values provided in this section represent half of the savings – or 5% improvement. These values are provided as a conservative deemed estimate for Missouri, while encouraging the use of performance testing and verification for determination of more accurate savings estimates.

Where:

CoolSavingsPerUnit = Annual cooling savings per linear foot of duct¹⁷⁵

Building Type	HVAC System	CoolSavingsPerUnit (kWh/ft)
Multifamily	Cool Central	0.70
Single-family	Cool Central	0.81
Manufactured	Cool Central	0.95
Multifamily	Heat Pump—Cooling	0.70
Single-family	Heat Pump—Cooling	0.81
Manufactured	Heat Pump—Cooling	0.95

DuctLength = Linear foot of duct
= Actual

HeatSavingsPerUnit = Annual heating savings per linear foot of duct¹⁷⁶

Building Type	HVAC System	HeatSavingsPerUnit (kWh/ft)
Manufactured	Heat Pump—Heating	5.06
Multifamily	Heat Pump - Heating	3.41
Single-family	Heat Pump— Heating	4.11

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW = \Delta kWh * CF$

Where:

ΔkWh = Electric energy savings, as calculated above
 CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0004660805¹⁷⁷

NATURAL GAS SAVINGS

For homes with Natural Gas Heating:

Methodology 1: Modified Blower Door Subtraction

$\Delta Therm = (\Delta CFM_{25DL} / ((CapacityHeat * 0.0136)) * EFLH_{heat} * CapacityHeat * \eta_{Equipment} / \eta_{System}) / 100,000$

Where:

ΔCFM_{25DL} = Duct leakage reduction in CFM25
= As calculated in Methodology 1 under electric savings
 $CapacityHeat$ = Heating input capacity (Btu/hr)
= Actual
0.0125 = Conversion of Capacity to CFM (0.0125CFM / Btu/hr)¹⁷⁸
 $\eta_{Equipment}$ = Heating Equipment Efficiency

¹⁷⁵ MO TRM, page 97, <https://dnr.mo.gov/document-search/missouri-technical-reference-manual-2017-volume-3-residential-measures>
¹⁷⁶ MO TRM, page 97, <https://dnr.mo.gov/document-search/missouri-technical-reference-manual-2017-volume-3-residential-measures>
¹⁷⁷ Based on Ameren Missouri TRM Volume 1 - Appendix G: "Table 1 – Residential End Use Category Monthly Shapes and Coincident Peak Factors" for Residential Cooling."
¹⁷⁸ Based on natural draft furnaces requiring 100 CFM per 10,000 Btu, induced draft furnaces requiring 130CFM per 10,000Btu, and condensing furnaces requiring 150 CFM per 10,000 Btu (rule of thumb from <https://www.contractingbusiness.com/archive/article/20861289/calculating-heating-system-airflow>). Data provided by GAMA during the federal rulemaking process for furnace efficiency standards, suggested that in 2000, 29% of furnaces purchased in Missouri were condensing units. Therefore, a weighted average required airflow rate is calculated assuming a 50:50 split of natural v induced draft non-condensing furnaces, as 125 per 10,000Btu or 0.0125/Btu.

η_{System}

= Actual¹⁷⁹ - If not available, use 83.5%¹⁸⁰

= Pre duct sealing Heating System Efficiency (Equipment Efficiency * Pre Distribution Efficiency)¹⁸¹

= Actual - If not available use 71.0%¹⁸²

100,000

= Converts Btu to therms

Methodology 2: Duct Blaster Testing

ΔTherms

= ((Pre_CFM25 – Post_CFM25)/(\Delta CFM25DL/CapacityHeat) * 0.0136 * EFLHgasheat * Equipment / η_{System})/100,000

Where:

All variables as provided above

Methodology 3: Deemed Savings¹⁸³

ΔTherms

= HeatSavingsPerUnit*Duct_{Length}

Where:

HeatSavingsPerUnit

= Annual heating savings per linear foot of duct¹⁸⁴

Building Type	HVAC System	HeatSavingsPerUnit (Therms/ft)
Multifamily	Heat Central Furnace	0.19
Single-family	Heat Central Furnace	0.21
Manufactured	Heat Central Furnace	0.26

Duct_{Length}

= Linear foot of duct

= Actual

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

¹⁷⁹ The actual Heating Equipment Efficiency can be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test. If there is more than one heating system, the weighted (by consumption) average efficiency should be used.

If the heating system or distribution is being upgraded within a package of measures together with the insulation upgrade, the new average heating system efficiency should be used.

¹⁸⁰ In 2000, 29% of furnaces purchased in Missouri were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment; see Furnace Penetration.xls). Furnaces tend to last up to 20 years, so units purchased 16 years ago provide a reasonable proxy for the current mix of furnaces in the state. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: (0.29*0.92) + (0.71*0.8) = 0.835.

¹⁸¹ The distribution efficiency can be estimated via a visual inspection and by referring to a look-up table such as that provided by the Building Performance Institute - (https://www.bpi.org/_cms/docs/Guidance%20on%20Estimating%20Distribution%20Efficiency.pdf; ‘Guidance on Estimating Distribution Efficiency.pdf’) - or by performing duct blaster testing.

¹⁸² Estimated as follows: 0.835 * (1-0.15) = 0.710.

¹⁸³ Savings per unit are based upon analysis performed by Cadmus for the 2011 Joint Assessment of Potential. It was based on 10% savings in system efficiency. This would represent savings from homes with significant duct work outside of the thermal envelope. With no performance testing or verification, a deemed savings value should be very conservative and therefore the values provided in this section represent half of the savings – or 5% improvement. These values are provided as a conservative deemed estimate for Missouri, while encouraging the use of performance testing and verification for determination of more accurate savings estimates.

¹⁸⁴ Iowa TRM v8.0, page 204, https://wcc.efs.iowa.gov/cs/idcplg?IdcService=GET_FILE&allowInterrupt=1&RevisionSelectionMethod=latest&dDocName=2129208&noSaveAs=1

3.4.4 Mini/Multi-Split Air Source Heat Pump and Air Conditioners

DESCRIPTION

This measure is designed to calculate electric savings from retrofitting existing electric HVAC systems with ductless and/or ducted mini/multi-split heat pumps (MMSHPs) or mini/multi-split air conditioners. MMSHPs save energy in heating mode because they provide heat more efficiently than electric resistance heat and central ASHP systems. Additionally, MMSHPs use less fan energy to move heat and don't incur heat loss through a lengthy duct distribution system while operating at very low static pressure. Often MMSHPs are installed in addition to (do not replace) existing heating or cooling equipment because the existing heating or cooling equipment is inadequate to efficiently heat or cool the space. Both ductless and ducted indoor units can be installed as a mixed mini/multi-split heat pump or air conditioner under this measure. Duct runs for a ducted mini/multi-split indoor unit should be installed within the conditioned envelope, be well-sealed and insulated ducts, and maintain low static pressure per manufacturer specifications for the installation configuration to maximize energy savings.

For cooling, the proposed savings calculations are aligned with those of typical replacement systems. MMSHPs save energy in cooling mode because they provide cooling capacity more efficiently than other types of unitary cooling equipment. A MMSHPs installed in a home with an existing central ASHP or CAC system will save energy by offsetting some of the cooling energy of the ASHP or CAC. In order for this measure to apply, the control strategy for the heat pump or air conditioner is assumed to be chosen to maximize savings per installer recommendation.¹⁸⁵

This measure was developed to be applicable to the following program type: NC, ROF, and ER.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the new equipment must be a high-efficiency, variable-capacity (typically “inverter-driven” DC motor) ductless and/or ducted mini/multi-split heat pump or air conditioning system that exceeds the program minimum efficiency requirements.

DEFINITION OF BASELINE EQUIPMENT

In order for this characterization to apply, baseline equipment must include a permanent electric resistance heating source or a ducted air-source heat pump or ducted air conditioner. For residences with central air conditioner/non-electric heating, cooling savings will only apply. For multifamily buildings, each residence must have existing individual heating equipment. Multifamily residences with central heating do not qualify for this characterization. Existing cooling equipment is assumed to be standard efficiency. Note that in order to claim cooling savings, there must be an existing air conditioning system (e.g. central air conditioning, Window ACs, or air source heat pump).

New federal standards affecting heat pumps became effective January 1, 2023. Under the new standards, the baseline for a ROF measure is the federal standard efficiency; 14.3 SEER2 and 7.5 HSPF2 when replacing a ducted air-source heat pump; 13.4 SEER2 and 3.41 HSPF2 when replacing a central air conditioner and electric resistance heating; 13.4 SEER2 when replacing central air conditioner with non-electric heating or no heating.

Under the new federal standards, the M1 testing protocol was revised, resulting in new SEER and HSPF performance metrics, now called SEER2 and HSPF2. When quantifying energy savings, the SEER2 and HSPF2 metrics should be used for the existing, baseline, and new equipment. The following conversion formulas can be used to convert between efficiency metrics:

$$\begin{aligned}\text{SEER2} &= \text{SEER} * 0.96 \\ \text{HSPF2} &= \text{HSPF} * 0.87\end{aligned}$$

For the early replacement baseline to apply, program participants or installation contractors must provide documentation asserting that the existing units operated when turned on—whether or not they provided cooling—and the existing unit brand name and model number must be documented; or, if the nameplate is not readable, a photograph of the unit in context and the nameplate specifically must be provided. The baseline for the early replacement measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit and the new baseline as defined above for the remainder of the measure life.

¹⁸⁵ The whole purpose of installing ductless heat pumps is to conserve energy, so the installer can be assumed to be capable of recommending an appropriate control strategy. For most applications, the heating setpoint for the ductless heat pump should be at least 2F higher than any remaining existing system and the cooling setpoint should be at least 2F cooler than the existing system (this should apply to all periods of a programmable schedule, if applicable). This helps ensure that the ductless heat pump will be used to meet as much of the load as possible before the existing system operates to meet the remaining load. Ideally, the new ductless heat pump controls should be set to the current comfort settings, while the existing system setpoints should be adjusted down (heating) and up (cooling) to capture savings.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 18 years.¹⁸⁶
Remaining life of existing ASHP/CAC equipment is assumed to be 6 years¹⁸⁷ and 18 years for electric resistance.

DEEMED MEASURE COST

Default full cost of the MMSHP is provided below. Note, for smaller units a minimum cost of \$2,000 should be applied:¹⁸⁸

Unit HSPF2	Full Install Cost (\$/ton) ¹⁸⁹
8.1-8.9	\$1,443
9-9.8	\$1,605
9.9-11.6	\$1,715
11.7+	\$2,041

New Construction and Time of Sale: If the unit is not displacing electric resistance heating or facilitating fuel switching, apply the incremental cost of the MMSHP compared to a baseline minimum efficiency MMSHP provided in the table below:¹⁹⁰

Efficiency (HSPF2)	Incremental Cost (\$/ton) over an HSPF2 7.5 MMSHP
8.1-8.9	\$62
9-9.8	\$224
9.9-11.6	\$334
11.7+	\$660

Otherwise, the incremental cost should be calculated as the greater of:

- Actual installed cost of the MMSHP should be used (defaults are provided above), minus the assumed installation cost of the baseline equipment (\$2,011 for a new baseline 80% AFUE furnace,¹⁹¹ and \$3,338 for new baseline Central AC replacement¹⁹²). If replacing electric resistance heat, there is no deferred cost for replacing the electric resistance heating unit.
- Applicable incremental cost relative to MMSHP identified in the table above.

Early Replacement/retrofit (replacing existing equipment): If available, the actual full installation cost of the MMSHP should be used; if unavailable, the default full cost specified above should be used. The assumed deferred cost of replacing existing equipment with a new baseline unit is assumed to be \$1,518¹⁹³ per ton for a new baseline MMSHP, or \$2,296 for a new baseline 80% AFUE furnace and \$3,670 for new baseline Central AC replacement.¹⁹⁴ If replacing electric resistance heat, there is no deferred cost for replacing the electric resistance heating unit. This future cost should be discounted to present value using a 2.31% nominal societal discount rate, based on the ten year average (1/1/2014 – 12/31/2023) of the 10 year Treasury bond yield rates.¹⁹⁵

If the deferred replacement cost exceeds the full installation cost of the MMSHP, the incremental cost shall be set to zero.

When a non-electric heating system is replaced with an MMSHP, and the program administrator is only claiming energy savings from cooling (i.e., no heating savings are claimed due to the fuel switch), the incremental costs should be adjusted to reflect the proportion of claimed savings.

¹⁸⁶ Measure Life Report: Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, Inc., June 2007, https://www.caetrm.com/media/reference-documents/HVAC_Ltg_measure_life_GDS_2007.pdf; ‘HVAC_Ltg_measure_life_GDS_2007.pdf’, page 1-3.

¹⁸⁷ Assumed to be one third of effective useful life.

¹⁸⁸ The cost per ton table provides reasonable estimates for installation costs of DMSHP, which can vary significantly due to requirements of the home. It is estimated that all units, even those 1 ton or less will be at least \$2000 to install.

¹⁸⁹ Full costs based upon full install cost of an ASHP plus incremental costs provided in Memo from Opinion Dynamics Evaluation Team, Ductless Mini-Split Heat Pumps: Incremental Cost Analysis, April 27, 2017.

¹⁹⁰ Memo from Opinion Dynamics Evaluation Team, Ductless Mini-Split Heat Pumps: Incremental Cost Analysis, April 27, 2017

¹⁹¹ See ‘Technical Standard Document_APPENDIX_E.pdf’.

¹⁹² See ‘CAC Costs 09.02.2024.xlsx’.

¹⁹³ Based on implicit standard efficiency cost of \$1,381 per ton (8.1-8.9 HSPF2 per ton full cost minus incremental cost), account for inflation rate of 1.91%.

¹⁹⁴ All baseline replacement costs are consistent with their respective measures and include inflation rate of 1.91%.

¹⁹⁵ ‘Societal_Discount_Rate_Calculation_08082024.xlsx’.

The incremental cost of the MMSHP shall be adjusted by applying a factor that represents the ratio of the claimed cooling savings to the total potential energy savings (inclusive of both cooling and unclaimed heating savings). This adjustment ensures that the incremental cost used in cost-effectiveness testing is proportional to the benefits being claimed, thus avoiding a mismatch between costs and benefits.

Calculation Method:

- 1. Determine Total Potential Savings: Calculate the sum of the potential energy savings from both cooling and heating (even if heating savings are not being claimed).
- 2. Determine Claimed Savings: Identify the portion of energy savings that is being claimed, typically the cooling savings only.
- 3. Apply the Adjustment Factor: The adjustment factor is calculated as the ratio of claimed savings to total potential savings. This factor is then applied to the full incremental cost of the MMSHP.

Adjusted Incremental Cost = (Claimed Savings (kWh) / Total Potential Savings (kWh)) * Full Incremental Cost)

LOADSHAPE
Cooling RES
Heating RES

Algorithms

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Electric savings

$\Delta kWh = \Delta kWh_{heating} + \Delta kWh_{cooling}$

Heating savings:

TOS:

$\Delta kWh_{heating} = ((Capacity_{heat} * EFLH_{heat} * (1/HSPF2_{base} - 1/HSPF2_{ee})) / 1000) * HF * ISR$

EREP:

$\Delta kWh_{heating} = ((Capacity_{heat} * EFLH_{heat} * (1/HSPF2_{exist} - 1/HSPF2_{ee})) / 1000) * HF * ISR$

Where:

$Capacity_{heat}$ = Heating capacity of the ductless heat pump unit in Btu/hr
= Actual

$EFLH_{heat}$ = Equivalent Full Load Hours for heating. See table below:

Weather Basis (Ameren Missouri Average)	$EFLH_{heat}^{196}$
SF or MF	1,034

$HSPF2_{exist}$ = Use actual HSPF2 rating where it is possible to measure or reasonably estimate. HSPF2 rating of existing equipment. If rated efficiency is unknown, use defaults provided below

¹⁹⁶ Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations.

Existing Equipment Type	HSPF2 _{exist} ¹⁹⁷
Electric resistance heating	3.412
Air Source Heat Pump	6.58

HSPF2_{base} = HSPF2 rating of baseline equipment (kBtu/kWh)

= 7.5 HSPF2¹⁹⁸ when replacing an ASHP

= 3.412 when replacing electric resistance heating

HSPF2_{ee} = HSPF rating of new equipment (kBtu/kWh)

= Actual installed

ISR = In-service rate. Actual, or if unknown, assume 100%¹⁹⁹

Cooling savings calculated only in presence of non-electric heating or MMAC (Mini/Multi-Split AC):

TOS:

$$\Delta \text{kWh}_{\text{cooling}} = ((\text{Capacity}_{\text{cool}} * \text{EFLH}_{\text{cool}} * (1/\text{SEER2}_{\text{base}} - 1/\text{SEER2}_{\text{ee}})) / 1000) * \text{HF} * \text{ISR}$$

EREP:

$$\Delta \text{kWh}_{\text{cooling}} = ((\text{Capacity}_{\text{cool}} * \text{EFLH}_{\text{cool}} * (1/\text{SEER2}_{\text{exist}} - 1/\text{SEER2}_{\text{ee}})) / 1000) * \text{HF} * \text{ISR}$$

Where:

Capacity_{cool} = the cooling capacity of the ductless heat pump unit in Btu/hr.²⁰⁰

= Actual installed

SEER2_{exist} = SEER rating of existing equipment (kBtu/kWh)

= Use actual SEER rating where possible to measure or reasonably estimate. If using rated efficiencies, derate the efficiency value based on the age of the existing equipment (up to a maximum of 30 years) to account for degradation over time.²⁰¹ If age is unknown, use 12 years.

= SEER2 * (1-0.01)^{Age}

If unknown, see table below

Existing Cooling System	SEER2 _{exist} ²⁰²
Air Source Heat Pump	6.91
Central AC	6.53
Room AC	6.3 ²⁰³
No existing cooling ²⁰⁴	Let '1/SEER _{exist} ' = 0

SEER2_{base} = Seasonal Energy Efficiency Ratio of baseline equipment (kBtu/kWh)²⁰⁵

¹⁹⁷ Ameren Missouri Heating and Cooling Evaluation PY2018, <https://www.efis.psc.mo.gov/Document/Display/15871>, page 36.

¹⁹⁸ Based on minimum federal standard effective 1/1/2023: 10 CFR 430.32(c)(5).

¹⁹⁹ Ameren Missouri HVAC Evaluation: PY2020, <https://www.efis.psc.mo.gov/Document/Display/13831>, page 53.

²⁰⁰ 1 Ton = 12 kBtu/hr.

²⁰¹ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 112. Justification for degradation factors can be found on page 14 of 'AIC HVAC Metering Study Memo FINAL 2_28_2018.docx'. Estimate efficiency as (Rated Efficiency * (1-0.01)^{Equipment Age}).

²⁰² ASHP existing efficiency assumes degradation and is sourced from the Ameren Missouri Heating and Cooling Program Impact and Process Evaluation: Program Year 2015. CAC assumed to follow the same trend in degradation as the ASHP: 9.12 SEER nameplate to 7.2 (6.91 SEER2) operations SEER represents degradation to 78.9% of nameplate. 78.9% of 8.6 SEER CAC nameplate gives an operational SEER of 6.8 (6.53 SEER2). 78.9% of 8.0 SEER RAC nameplate gives an operational SEER of 6.3.

²⁰³ Estimated by converting the EER assumption using the conversion equation; EER_{base} = (-0.02 * SEER_{base}²) + (1.12 * SEER). From Wassmer, M. (2003), "A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations," (Masters thesis) University of Colorado at Boulder. Adjusted to account for degradation per above footnote.

²⁰⁴ If there is no existing cooling in place but the incentive encourages installation of a new DMSHP with cooling, the added cooling load should be subtracted from any heating benefit.

²⁰⁵ SEER to SEER2 conversion factor: SEER2 = SEER * 96%. Conversion factor for SEER to SEER2 is used when converting an existing system that is rated in SEER to SEER2. This is to meet the DOE M1 CFR Standard beginning January 1, 2023. The efficiency levels of the existing, baseline, and efficient case must be expressed in SEER2 terms before applying formulas.

SEER2_{ce}

EFLH_{cool}

= 14.3 SEER2²⁰⁶ when replacing an ASHP

= 13.4 SEER2 when replacing a CAC

= SEER rating of new equipment (kBtu/kWh)

= Actual installed²⁰⁷

= Equivalent Full Load Hours for cooling. See table below

Weather Basis (Ameren Missouri Average)	EFLH _{cool}
SF or MF	635

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW = \Delta kWh_{Cooling} \times CF$

Where:

$\Delta kWh_{Cooling}$

CF

= Electric energy savings for cooling, calculated above

= Summer peak coincidence demand (kW) to annual energy (kWh) factor

= 0.0009474181

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

²⁰⁶ Based on minimum federal standard effective 1/1/2023: 10 CFR 430.32(c)(5).

²⁰⁷ Note that if only an EER rating is available, use the following conversion equation; $EER_{base} = (-0.02 * SEER_{base}^2) + (1.12 * SEER)$. From Wassmer, M. (2003). A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder.

3.4.5 Standard Programmable Thermostat

DESCRIPTION

This measure characterizes the household energy savings from the installation of a new standard programmable thermostat for reduced heating and cooling energy consumption through temperature set-back during unoccupied or reduced demand times.

Energy savings are applicable at the household level; installation of multiple programmable thermostats per home does not accrue additional savings.

If the home has a heat pump, a programmable thermostat specifically designed for heat pumps should be used to minimize the use of backup electric resistance heat systems.

This measure was developed to be applicable to the following program types: DI. This measure is only applicable for low income programs. Savings will not be claimed for this measure for programs other than low income programs.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The criteria for this measure are established by replacement of a manual-only temperature control with one that has the capability to adjust temperature setpoints according to a schedule without manual intervention.

DEFINITION OF BASELINE EQUIPMENT

For new thermostats the baseline is a non-programmable thermostat requiring manual intervention to change temperature set point.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected equipment life of a programmable thermostat is assumed to be 10 years.²⁰⁸

DEEMED MEASURE COST

Actual material and labor costs should be used if the implementation method allows. If unknown (e.g., through a retail program), the capital cost for the new installation is assumed to be \$70.²⁰⁹

LOADSHAPE

Cooling RES

Heating RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

For central air conditioners and air source heat pumps:

$\Delta kWh_{cool} = EFLH_{cool} * Capacity_{Cooling} * (1/SEER2) * SBdegrees * SF * EF / 1,000 * ISR$

For air source heat pumps there are additional heating savings:

$\Delta kWh_{heat} = EFLH_{heat} * Capacity_{Heating} * (1/HSPF2) * SBdegrees * SF * EF / 1,000 * ISR$

²⁰⁸ Table 1, Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, <https://energizect.com/sites/default/files/documents/Measure%20Life%20Report%202007.pdf>; ‘Measure Life Report 2007.pdf’, page 1-3.

. Future evaluation is strongly encouraged to inform the persistence of savings to further refine measure life assumption. As this characterization depends heavily upon a large scale but only 2-year study of the energy impacts of programmable thermostats, the longer-term impacts should be assessed.

²⁰⁹ Market prices vary significantly in this category, generally increasing with thermostat capability and sophistication. The basic functions required by this measure's eligibility criteria are available on units readily available in the market for \$30. Labor is assumed to be one hour at \$40 per hour.

Where:

$EFLH_{cool}$ = Equivalent full load hours of air conditioning²¹⁰:

Weather Basis (Ameren Missouri Average)	$EFLH_{cool}$ (Hours)
SF or MF	869

$Capacity_{Cooling}$ = Cooling capacity of system in BTU/hr (1 ton = 12,000 BTU/hr)
= Use Actuals based upon units served

$SEER2$ = Seasonal Energy Efficiency Ratio of cooling system (kBtu/kWh)
= Actual. If unknown, use defaults provided below:

Cooling System	SEER
Air Source Heat Pump	10 ²¹¹
Central AC	10 ²¹²

$HSPF2$ = Heating Season Performance Factor of heating system (kBtu/kWh)
= Actual. If unknown, use defaults provided below:

Existing Heating System	HSPF2
Air Source Heat Pump	7.00 ²¹³
Electric Resistance	3.41 ²¹⁴

$EFLH_{heat}$ = Equivalent full load hours of heating:²¹⁵

Weather Basis (Ameren Missouri Average)	$EFLH_{heat}$ (Hours)
SF or MF	1496

$Capacity_{Heating}$ = Heating capacity of system in BTU/hr (1 ton = 12,000 BTU/hr)
= Use Actuals based upon units served

$SBdegrees$ = weighted sum of setback degrees to comfort temperature
= $SBdegrees_{Heating} = 1.8^{216}$
= $SBdegrees_{Cooling} = 1.91^{217}$

SF = Savings factors from ENERGY STAR® calculator
= 3% / degree heat, 6% / degree cool

EF = Efficiency ratio from Cadmus metering study
= 13% heat²¹⁸
= 100% cool²¹⁹

ISR = In-service rate
= Actual, or if unknown, assume 100%.

²¹⁰ Ameren Missouri Program Year 2019 Annual EM&V Report Volume 2: Residential Portfolio Appendices, <https://www.efis.psc.mo.gov/Document/Display/15876>, page 30.

²¹¹ Ameren Missouri Community Saver Program Evaluation PY2018 <https://www.efis.psc.mo.gov/Document/Display/36053>, page 26.

²¹² Ameren Missouri Community Saver Program Evaluation PY2018, <https://www.efis.psc.mo.gov/Document/Display/36053>, page 26.

²¹³ IL-TRM (Based on minimum federal standards between 1992 and 2006) – Ameren Missouri Community Saver Program Evaluation PY2018.

²¹⁴ Electric resistance has a COP of 1.0 which equals $1/0.293 = 3.41$ HSPF.

²¹⁵ Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations, <https://www.efis.psc.mo.gov/Document/Display/15876>, page 76.

²¹⁶ Ameren Missouri Community Saver Program Evaluation PY2018 Site Visit Thermostat SB Data.

²¹⁷ Ameren Missouri Community Saver Program Evaluation PY2018 Site Visit Thermostat SB Data.

²¹⁸ Ameren Missouri Community Saver Program Evaluation PY2014 Cadmus metering study, <https://www.efis.psc.mo.gov/Document/Display/15857>, (PY2014 pg. 31).

²¹⁹ Ameren Missouri Community Saver Program Evaluation PY2017, <https://www.efis.psc.mo.gov/Document/Display/28281>.

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW = \Delta kWh_{Cooling} * CF$

Where:

$\Delta kWh_{Cooling}$ = Electric energy savings for cooling, calculated above
 $CF = 0.0009474181$

NATURAL GAS ENERGY SAVINGS

$\Delta Therms = \%FossilHeat * HeatingConsumption_{Gas} * HF * Heating_{Reduction} * Eff_{ISR} * PF$

Where:

$\%FossilHeat$ = Percentage of heating savings assumed to be natural gas

Heating fuel	$\%FossilHeat$
Electric	0%
Natural Gas	100%
Unknown	65% ²²⁰

$HeatingConsumption_{Gas}$ = Estimate of annual household heating consumption for gas heated single-family homes.²²¹

Weather Basis (City based upon)	Gas_Heating_Consumption (Therms)
St Louis, MO	680

Other variables as provided above.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

²²⁰ Average (default) value of 65% gas space heating from 2009 Residential Energy Consumption Survey for Missouri. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then they should be used.

²²¹ Values in table are based on average household heating load (834 therms) for Chicago based on Illinois furnace metering study (Table E-1, Energy Efficiency/Demand Response Nicor Gas Plan Year 1: Research Report: Furnace Metering Study, Draft, <https://www.icc.illinois.gov/docket/P2012-0528/documents/201939/files/355536.pdf> ; ‘355536.pdf’, Navigant, August 1 2013) and adjusted for Missouri weather basis values using the relative climate normals HDD data with a base temp ratio of 60°F. This load value is then divided by standard assumption of existing unit efficiency of 83.5% (estimate based on 29% of furnaces purchased in Missouri were condensing in 2000 (based on data from GAMA, provided to Department of Energy) (see ‘Thermostat_FLH and Heat Load Calcs.xls’). The resulting values are generally supported by data provided by Laclede Gas, which showed an average pre-furnace replacement consumption of 1009 therms for St Louis, and a post-replacement consumption of 909. Assuming a typical hot water consumption at 225 therms (using defaults from <http://energy.gov/eere/femp/energy-cost-calculator-electric-and-gas-water-heaters-0#output>), this indicates a heating load of 684-784 therms.

3.4.6 HVAC Tune-Up (Central Air Conditioning or Air Source Heat Pump)

DESCRIPTION

This measure involves the measurement of refrigerant charge levels and airflow over the central air conditioning or heat pump unit coil, correction of any problems found, and post-treatment re-measurement. Tune-up activities include a general tune-up, refrigerant charge, indoor coil cleaning, and outdoor coil cleaning. These tune-up actions may be performed individually or as a packaged service with more than one tune-up activity.

This measure was developed to be applicable to the following program type: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

A tuned and commissioned residential central air conditioning unit or air source heat pump.

DEFINITION OF BASELINE EQUIPMENT

An existing residential central air conditioning unit or air source heat pump that has required tuning to restore optimal performance.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 2 years.²²²

DEEMED MEASURE COST

As a RF measure, actual costs should be used. If unavailable, the measure cost should be assumed to be \$175.²²³ The table below identifies more specific costs for varying services.

Tune- up Service for HP or AC	Incremental Cost (\$)
Tune-up / refrigerant charge	\$81.00
Tune-Up / Packaged Service	\$185 ²²⁴

LOADSHAPE

Cooling RES

Heating RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$\Delta kWh_{\text{Central AC}} = ((EFLH_{\text{cool}} * Capacity_{\text{cool}} * (1/SEER_{\text{test-in}} - 1/SEER_{\text{test-out}})) / 1,000)$

$\Delta kWh_{\text{ASHP}} = ((EFLH_{\text{cool}} * Capacity_{\text{cool}} * (1/SEER_{\text{test-in}} - 1/SEER_{\text{test-out}})) / 1,000) + ((EFLH_{\text{heat}} * Capacity_{\text{heat}} * (1/HSPF2_{\text{test-in}} - 1/HSPF2_{\text{test-out}})) / 1,000)$

When test-in and test-out data is not metered, savings may be estimated by:

$$\frac{\Delta kWh_{\text{Central AC}}}{\Delta kWh_{\text{ASHP}}} = \frac{EFLH_{\text{cool}} * Capacity_{\text{cool}} * (1/(SEER2 * (1-SF)) / 1,000) * SF}{EFLH_{\text{cool}} * Capacity_{\text{cool}} * (1/(SEER2 * (1-SF)) / 1,000) * SF + EFLH_{\text{heat}} * Capacity_{\text{heat}} * (1/(HSPF2*(1-SF)) / 1,000)}$$

²²² Sourced from DEER Database Technology and Measure Cost Data. Secondary source that supports this range of original cost values: <https://www.hvac.com/expert-advice/ac-tune-up-cost/>

²²³ Based on personal communication with HVAC efficiency program consultant Buck Taylor of Roltay Inc., 6/21/10, who estimated the cost of tune up at \$125 to \$225, depending on the market and the implementation details.

²²⁴ Estimated average packaged tune-up cost based on implementer data from 2015-2016.

Where:

EFLH _{cool}	= Equivalent full load hours of air conditioning = dependent on location: ²²⁵
Capacity _{cool}	= Cooling Capacity of Air Source Heat Pump (Btu/hr) = Actual
SEER _{test-in}	= Seasonal Energy Efficiency Ratio of existing cooling system before tuning (kBtu/kWh) = In most instances, test-in EER will be determined and noted prior to tuning. SEER rating can be estimated by using the following relationship: ²²⁶ $EER = (-0.02 * SEER^2) + (1.12 * SEER)$ When unknown, ²²⁷ assume SEER = 11.9
SEER _{test-out}	= Seasonal Energy Efficiency Ratio of existing cooling system after tuning (kBtu/kWh) = In most instances, test-out EER will be determined and noted after tuning. SEER rating can be estimated by using the following relationship: ²²⁸ $EER = (-0.02 * SEER^2) + (1.12 * SEER)$; if unknown, reference applicable assumed value in table below.
EFLH _{heat}	= Equivalent full load hours of heating:
Capacity _{heat}	= Heating Capacity of Air Source Heat Pump (Btu/hr) = Actual
HSPF2 _{test-in}	= Heating Seasonal Performance Factor of existing ASHP before tuning (kBtu/kWh) = Actual, or if unknown, assume HSPF = 6.3. ²²⁹
HSPF2 _{test-out}	= Heating System Performance Factor of existing ASHP after tuning (kBtu/kWh)

SEER2 = Seasonal Energy Efficiency Ratio of existing cooling system indicated by model number, nameplate, or AHRI number

SF =Savings factor by type of tuneup.

Effective full load heating and cooling hours by building type:

Weather Basis (Ameren Missouri Average)	EFLH _{cool} (Hours)	EFLH _{heat} (Hours)
SF or MF	869 ²³⁰	1496 ²³¹

When SEER test-in and test-out values are unknown, tune-ups are assumed to improve efficiency as follows:

Measure	% Improvement	SEER _{test-out} (based on default 11.9 test-in value)
Tune-up + Refrigerant charge adjustment	28.4% ²³²	15.28 (28.4% improvement)
Tune-Up + Refrigerant check	8.0% ²³³	12.85 (8.0% improvement)

When HSPF test-out values are unknown, use the following default test-out values based on the tune-up service(s) performed:

Measure	HSPF _{test-out} (based on default 6.3 test-in value)
Tune-up + Refrigerant charge adjustment	8.09 (28.4% improvement)
Tune-up + Refrigerant check	6.80(8.0% improvement)

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW = \Delta kWh_{Cooling} * CF$

²²⁵ PY2019 Residential Evaluation Report Appendices, <https://www.efis.psc.mo.gov/Document/Display/15876>, page 35.

²²⁶ Based on Wassmer, M. (2003),” A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations,” (Masters thesis) University of Colorado at Boulder. Note this is appropriate for single speed units only.

²²⁸ Based on Wassmer, M. (2003), “A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations,” (Masters thesis), University of Colorado at Boulder. Note: this is appropriate for single speed units only.

²²⁸ Based on Wassmer, M. (2003), “A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations,” (Masters thesis), University of Colorado at Boulder. Note: this is appropriate for single speed units only.

²³⁰. PY2019 Evaluation Report Appendices, <https://www.efis.psc.mo.gov/Document/Display/15876>, page 76.

²³⁰. PY2019 Evaluation Report Appendices, <https://www.efis.psc.mo.gov/Document/Display/15876>, page 76.

²³¹ Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations, <https://www.efis.psc.mo.gov/Document/Display/15876>, page 76.

²³² meren Missouri PY2015 Evaluation, page 42, <https://www.efis.psc.mo.gov/Document/Display/13806>.

²³³

Where:

$$\frac{\Delta kWh_{Cooling}}{CF}$$

= Electric energy savings for cooling, calculated above

= 0.0009474181

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

3.4.7 ECM/Blower Motor

DESCRIPTION

This measure describes savings from a brushless permanent magnet (BPM) motor (known and referred in this measure as an electronically commutated motor (ECM)) compared to a lower efficiency motor. Time of Sale and New Construction replacement scenarios no longer apply to this measure, as federal standards make ECM blower fan motors a requirement for residential furnaces. Savings however are available from retrofitting an ECM motor into an existing furnace, or replacing an operational inefficient furnace with a new furnace with an ECM prior to the end of its life.

This measure characterizes the electric savings associated with the fan and the interactive negative therm savings due to a reduction in waste heat of the fan when operating in heating mode.

Savings decrease sharply with static pressure so duct improvements, and clean, low pressure drop filters can maximize savings. Savings occur when the blower is used for heating, cooling as well as when it is used for continuous ventilation, but only if the non-ECM motor would have been used for continuous ventilation too. If the resident runs the ECM blower continuously because it is a more efficient motor and would not run a non-ECM motor that way, savings are near zero and possibly negative. This characterization uses a 2009 Focus on Energy study of BPM blower motor savings in Wisconsin, which accounted for the effects of this behavioral impact.

Retrofitting an existing blower motor with a new ECM reduces the potential impact of the high efficiency motor over a new system designed for an ECM blower motor because existing systems were not designed to capitalize and take advantage of the ECM's multi-staging features. Energy and demand savings are limited to the efficiency gains from the motor itself.

Note: as part of a Time of Sale measure, it is not appropriate to claim additional ECM fan savings due to installing a new furnace or CAC unit as ECM motors are now baseline for new furnaces and the SEER2/EER2 ratings of a CAC unit already account for this electrical load.

In an early replacement furnace situation, ECM fan heating savings can be claimed for the RUL of the existing furnace, and cooling savings can be claimed for the RUL of the CAC if an existing cooling unit is not replaced.

If a new CAC unit is installed in a home where the existing furnace is not replaced, heating ECM savings should only be claimed if it can be demonstrated that the new CAC motor will be used for the heating load.

This measure was developed to be applicable to the following program types: RF, EREP

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

A furnace with a brushless permanent magnet (BPM) blower motor, also known by the trademark ECM, BLDC, and other names.

DEFINITION OF BASELINE EQUIPMENT

A furnace with a non-BPM blower motor. As part of the Code of Federal Regulations, energy conservation standards for covered residential furnace fans became effective on July 3, 2019 (10 CFR 430.32(y)). This code requirement effectively makes ECMs part of the baseline for New Construction (NC), Replace-on-Fail (ROF), Time-of-Replacement (TOS), and Early Replacement (EREP) scenarios.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 6 years, which is the remaining life of existing furnaces.²³⁴

DEEMED MEASURE COST

The capital cost for this measure as a retrofit should be actual if known; if unknown, assume \$350.²³⁵ In cases of furnace early replacements, it is assumed the incremental cost of the ECM is \$0.

LOADSHAPE

HVAC RES

²³⁴ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 150.

²³⁵ Ibid., p. 151. The cost of a typical replacement motor is estimated at \$180 based on quotes from online suppliers, plus \$17 for the bracket. Typical labor costs are estimated at between \$140 and \$190 based on program experience provided by Staples in April 2022. A total retrofit measure cost is therefore estimated at \$350.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$\Delta kWh_{\text{Heating Mode}} = (1 - \%_{\text{with_New_ASHP}}) * (400 \text{ kWh/year} * \text{HeatingEFLH} / \text{WisconsinHeatingEFLH}) * \text{HF} * \text{ISR}$
 $\Delta kWh_{\text{Cooling Mode}} = (1 - \%_{\text{with_New_Central_Cooling}}) * (70 \text{ kWh/year} * \text{CoolingEFLH} / \text{WisconsinCoolingEFLH}) * \text{HF} * \text{ISR}$
 $\Delta kWh_{\text{Auto Circulation}} = (25 \text{ kWh/year} * \text{CoolingEFLH} / \text{WisconsinCoolingEFLH} + 2,960 \text{ kWh/year} * \text{RT}\% - 30 \text{ kWh/year}) * \text{HF} * \text{ISR}$

Where:

Parameter	Value
Wisconsin Cooling Savings kWh/year	70.00 ²³⁶
Cooling Savings All Systems	25.00 ²³⁷
Wisconsin Cooling EFLH	542.50 ²³⁸
Wisconsin Heating Savings kWh/year	400.00 ²³⁹
Wisconsin Heating EFLH	2,545.25 ²⁴⁰
Wisconsin Circulation Savings kWh/year	2,960.00 ²⁴¹
RT=Percent additional run time factor	8.81% ²⁴²
Standby losses	30 ²⁴³
Saint Louis Heating EFLH	2,009.00 ²⁴⁴
Saint Louis Cooling EFLH	1,215.00 ²⁴⁵
% with New Central Cooling	82% ²⁴⁶
% with New ASHP	16% ²⁴⁷
ISR (In Service Rate)	Actual, or if unknown, assume 100% ²⁴⁸
HF	100% ²⁴⁹

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW = \Delta kWh * CF$

Where:

ΔkWh = Electric energy savings, as calculated above
 CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0004660805

NATURAL GAS SAVINGS

²³⁶ Ameren Missouri HVAC Program Evaluation PY2017, page 41, <https://www.efis.psc.mo.gov/Document/Display/14208>
²³⁷ Ibid.
²³⁸ Ibid.
²³⁹ Ibid.
²⁴⁰ Ibid.
²⁴¹ Ibid.
²⁴² Ameren Missouri HVAC Program Evaluation PY2019, page 39, <https://www.efis.psc.mo.gov/Document/Display/15876>.
²⁴³ Ameren Missouri HVAC Program Evaluation PY2017, page 41, <https://www.efis.psc.mo.gov/Document/Display/14208>
²⁴⁴ Ibid.
²⁴⁵ Ibid.
²⁴⁶ Ameren Missouri HVAC Program Evaluation PY2019, <https://www.efis.psc.mo.gov/Document/Display/15877>, page 90.
²⁴⁷ Ibid.
²⁴⁸ Ameren Missouri HVAC Program Evaluation PY2020, <https://www.efis.psc.mo.gov/Document/Display/13831>, page 53.
²⁴⁹ Household Factor (HF) is assumed to be 100%. 65% multifamily value is not applicable for this measure, as savings should be based upon pressure drop in the system.

$\Delta\text{therms}^{250} = - \text{Heating Savings} * 0.03412 / \text{AFUE}$

Where:

- 0.03412 = Converts kWh to therms
- AFUE = Efficiency of the Furnace
- = Actual. If unknown assume 95%²⁵¹ if in new furnace or 64.4 AFUE%²⁵² if in existing furnace

Using defaults:

- For new Furnace = - (430 * 0.03412) / 0.95
- = - 15.4 therms
- For existing Furnace = - (430 * 0.03412) / 0.644
- = - 22.8 therms

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

²⁵⁰ The blower fan is in the heating duct so all, or very nearly all, of its waste heat is delivered to the conditioned space. Negative value since this measure will increase the heating load due to reduced waste heat.

²⁵¹ Minimum efficiency rating from ENERGY STAR® Furnace Specification v4.0, effective February 1, 2013, https://www.energystar.gov/sites/default/files/specs/private/Final_Version_4.0_Specification.pdf; 'Final_Version_4.0_Specification.pdf', page 2.

²⁵² Average nameplate efficiencies of all early replacement qualifying equipment in Ameren IL PY2003-PY2004.

3.4.8 Central Air Conditioner

DESCRIPTION

This measure characterizes:

1. TOS: The installation of a new residential sized ($\leq 65,000$ Btu/hr) central air conditioning ducted split system meeting ENERGY STAR® efficiency standards presented below. This could relate to the replacement of an existing unit at the end of its useful life, or the installation of a new system in a new home.
2. EREP: For the early replacement baseline to apply, program participants or installation contractors must provide documentation asserting that the existing units operated when turned on—whether or not they provided cooling—and the existing unit brand name and model number must be documented; or, if the nameplate is not readable, a photograph of the unit in context and the nameplate specifically must be provided. All other conditions will be considered TOS. The baseline SEER2 of the existing central air conditioning unit replaced: If the SEER2 of the existing unit is known and, the baseline SEER2 is the actual SEER2 value of the unit replaced. If the SEER2 of the existing unit is unknown, use assumptions in variable list below (SEER2_exist).

This measure was developed to be applicable to the following program types: TOS, NC, and EREP.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be a ducted split central air conditioning unit meeting the minimum ENERGY STAR® efficiency level standards. For reference, the minimum ENERGY STAR® version 6.1 efficiency level standards are provided below²⁵³:

- Split system central air conditioners – 15.2 SEER2 and 12.0 EER2
- Single package central air conditioners – 15.2 SEER2 and 11.5 EER2
- Space constrained units – 13.4 SEER2²⁵⁴

The measure characterization recommends sourcing the efficiency specifications from the actually installed equipment. If those values are not known, the default equipment efficiency recommendations are conservatively based on ENERGY STAR® version 6.1 specifications.

DEFINITION OF BASELINE EQUIPMENT

The baseline for the TOS measure is based on the current federal standard efficiency level²⁵⁵:

- Standard sized Split system air conditioners – 13.4 SEER2
- Standard sized Single-package air conditioners – 13.4 SEER2
- Space constrained air conditioners – 11.7 SEER2

Under the new federal standards, the M1 testing protocol was revised, resulting in a new SEER performance metric called SEER2. When quantifying energy savings, the SEER2 metric should be used for the existing, baseline, and new equipment. The following conversion formula can be used to convert between efficiency metrics:

²⁵³ ENERGY STAR Program Requirements Product Specification for Central Air Conditioner and Heat Pump Equipment, v6.1, effective January 1, 2023, <https://www.energystar.gov/sites/default/files/ENERGY%20STAR%20Version%206.1%20Central%20Air%20Conditioner%20and%20Heat%20Pump%20Final%20Specification%20%28Rev.%20January%20%202022%29.pdf>; 'ENERGY STAR Version 6.1 Central Air Conditioner and Heat Pump Final Specification (Rev. January 2022).pdf', are in terms of an updated metric, depicted as SEER2 and EER2. The updated test method as well as the updated ENERGY STAR specifications mimic the updated federal appliance standards. An equivalent stringency of these new standards for split system air conditioners are 16 SEER and 13 EER and for single-package air conditioners are 16 SEER and EER 12, as detailed in: Consortium for Energy Efficiency (CEE) Residential HVAC Specifications, Estimated Appendix M1 Equivalents, January 15 2021.

²⁵⁴ The ENERGY STAR specification does not provide an efficiency level for space constrained products but this is a proposed level for this product type that the marketplace has developed solutions to meet.

²⁵⁵ The 2023 federal standards (10 CFR 430.32(c)(5)) are in terms of an updated metric, depicted as SEER2 and manufacturers must certify their products meet the standard according to the new test procedure and new metrics. The updated test method as well as the updated energy conservation standards were negotiated under the appliance standards and rulemaking federal advisory committee (ASRAC) in accordance with the Federal Advisory Committee Act (FACA) and the negotiated rulemaking act. An equivalent stringency of these new standards for split system air conditioners are 14 SEER and for single-package air conditioners are 14 SEER, as detailed in: Federal Code of Regulations, Energy Conservation Program: Energy Conservations Standards for residential Central Air Conditioners and Heat Pumps; Confirmation of effective date and compliance date for direct final rule, May 26, 2017, Docket: EERE-2014-BT-STD-0048 (<https://www.regulations.gov/document/EERE-2014-BT-STD-0048-0200>).

SEER2 = SEER * 0.96

The baseline for the early replacement measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit and the new baseline as defined above²⁵⁶ for the remainder of the measure life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 18 years.²⁵⁷

Remaining life of existing equipment is assumed to be 6 years.²⁵⁸

DEEMED MEASURE COST

Time of sale: The incremental capital cost for this measure is dependent on efficiency. Assumed incremental costs are provided below:²⁵⁹

Efficiency Level (SEER2)	Incremental Cost
13.9	\$111
14.4	\$230
14.9	\$453
15.4	\$635
16.3	\$861
16.8	\$891
17.3	\$921
19.2	\$1,006
21.1	\$1,120
22.4	\$1,240
13.9	\$111
14.4	\$230
14.9	\$453
15.4	\$635

Early replacement: The full install cost for this measure is the actual cost of removing the existing unit and installing the new one. If this is unknown, assume defaults below.²⁶⁰

Efficiency Level (SEER2)	Full Retrofit Cost
13.9	\$3,450
14.4	\$3,569
14.9	\$3,791
15.4	\$3,973
16.3	\$4,200
16.8	\$4,229
17.3	\$4,259
19.2	\$4,345
21.1	\$4,458
22.4	\$4,579
13.9	\$3,450
14.4	\$3,569
14.9	\$3,791

²⁵⁶ Baseline SEER and EER should be updated when new minimum federal standards become effective.

²⁵⁷ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, <https://energizect.com/sites/default/files/documents/Measure%20Life%20Report%202007.pdf>; ‘Measure Life Report 2007.pdf’.

²⁵⁸ Assumed to be one third of effective useful life.

²⁵⁹ See ‘CAC Costs 09.02.2024.xlsx’.

²⁶⁰ Ibid.

Efficiency Level (SEER2)	Full Retrofit Cost
15.4	\$3,973

Assumed deferred cost (after 6 years) of replacing existing equipment with new baseline unit is assumed to be \$3,670.²⁶¹ This future cost should be discounted to present value using a 2.31% nominal societal discount rate, based on the ten year average (1/1/2014 – 12/31/2023) of the 10 year Treasury bond yield rates.²⁶²

LOADSHAPE
Cooling RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Time of sale:

$\Delta kWh = ((FLH_{cool} * Capacity * (1/SEER2_{base} - 1/SEER2_{ee}))/1,000) * HF * ISR$

Early replacement:²⁶³

ΔkWh for remaining life of existing unit (1st 6 years):

$=((FLH_{cool} * Capacity * (1/SEER2_{exist} - 1/SEER2_{ee}))/1,000) * HF * ISR$

ΔkWh for remaining measure life (next 12 years):

$= ((FLH_{cool} * Capacity * (1/SEER2_{base} - 1/SEER2_{ee}))/1,000) * HF * ISR$

Where:

FLH_{cool} = Full load cooling hours:²⁶⁴

Weather Basis (Ameren Missouri Average)	EFLHcool (Hours)
SF or MF	869

- Capacity
- = Size of new equipment in Btu/hr (note 1 ton = 12,000Btu/hr)
- = Actual installed, or if actual size unknown 33,600Btu/hr for single-family buildings²⁶⁵
- SEER2_{base}
- = Seasonal Energy Efficiency Ratio of baseline unit (kBtu/kWh)²⁶⁶
- = 13.4 SEER2 for standard sized units or 11.7 SEER2 for space constrained units²⁶⁷
- SEER2_{exist}
- = Seasonal Energy Efficiency Ratio of existing unit (kBtu/kWh)
- = Use actual SEER2 rating where it is possible to measure or reasonably estimate. If using rated efficiencies, derate the efficiency value based on the age of the existing equipment (up to a maximum of 30 years) to account for degradation over time.²⁶⁸ If age is unknown, use 12 years.
- = SEER2 * (1-0.01)^{Age}

²⁶¹ Ibid.

²⁶² ‘Societal_Discount_Rate_Calculation_08082024.xlsx’.

²⁶³ The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a First Year savings (using the first equation) and then a “number of years to adjustment” and “savings adjustment” input which would be the (new base to efficient savings)/(existing to efficient savings).

²⁶⁴ Evaluation - Opinion Dynamics review PY2019. <https://www.efis.psc.mo.gov/Document/Display/15876>, page 30.

²⁶⁵ Actual unit size required for multifamily building, no size assumption provided because the unit size and resulting savings can vary greatly depending on the number of units.

²⁶⁶ SEER to SEER2 conversion factor: SEER2 = SEER * 96%. Conversion factor for SEER to SEER2 is used when converting an existing system that is rated in SEER to SEER2. This is to meet the DOE M1 CFR Standard beginning January 1, 2023. The efficiency levels of the existing, baseline, and efficient case must be expressed in SEER2 terms before applying formulas.

²⁶⁷ Based on minimum federal standard effective 1/1/2023: 10 CFR 430.32(c)(5)

²⁶⁸ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 112. Justification for degradation factors can be found on page 14 of ‘AIC HVAC Metering Study Memo FINAL 2_28_2018.docx’. Estimate efficiency as (Rated Efficiency * (1-0.01)^{Equipment Age}).

SEER _{2ce}	If unknown, assume 8.9. ²⁶⁹ , which is already adjusted to account for age-related degradation. = Seasonal Energy Efficiency Ratio of ENERGY STAR® unit (kBtu/kWh) = Actual installed or 15.2 if unknown.
HF	= For Multifamily units, use a factor of 65% to convert residential single family to multifamily capacity. If actual capacity is used apply 100%.
ISR	= In service rate = Actual, or if unknown, assume 100% ²⁷⁰

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW = \Delta kWh * CF$

Where:

ΔkWh	= Electric energy savings, as calculated above
CF	= Summer peak coincidence demand (kW) to annual energy (kWh) factor = 0.0009474181

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

²⁶⁹ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 130.

²⁷⁰ Ameren Missouri HVAC Evaluation: PY2020, <https://www.efis.psc.mo.gov/Document/Display/13831>, page 53.

3.4.9 Filter Cleaning or Replacement and Dirty Filter Alarms

DESCRIPTION

An air filter on a central forced air heating system is replaced prior to the end of its useful life with a new filter, resulting in a lower pressure drop across the filter. As filters age, the pressure drop across them increases as filtered medium accumulates. Replacing filters before they reach the point of becoming ineffective can save energy by reducing the pressure drop required by filtration, subsequently reducing the load on the blower motor.

This measure was developed to be applicable to the following program type: RET.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

A new filter offering a lower pressure drop across the filter medium compared to the existing filter.

DEFINITION OF BASELINE EQUIPMENT

A filter that is nearing the end of its effective useful life, defined by having a pressure drop twice that of its original state.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 1 year²⁷¹ for a filter replacement and 5 years²⁷² for a dirty filter alarm.

DEEMED MEASURE COST

Actual material and labor cost should be used if known, since there is a wide range of filter types and costs. If unknown,²⁷³ the cost of a fiberglass filter is assumed to be \$7.33 and the cost of a pleated filter is assumed to be \$15.66. If unknown, the cost of a dirty filter alarm is assumed to be \$5.

LOADSHAPE

HVAC RES

Algorithm

CALCULATION OF SAVINGS

Electric energy savings are calculated by estimating the difference in power requirements to move air through the existing and new filter and multiplying by the anticipated operating hours of the blower during the heating season.

ELECTRIC ENERGY SAVINGS

ΔkWh

kWh_{heating}

kWh_{cooling}

= kWh_{heating} + kWh_{cooling}

= %Heating * kW_{motor} * EFLH_{heat} * EI * (1 – Leakage) * ISR

= %AC * kW_{motor} * EFLH_{cool} * EI * (1 – Leakage) * ISR

²⁷¹ Many manufacturers suggest replacing filters more often than an annual basis, however this measure assumes that a filter will generally last one full heating season before it needs replacement.

²⁷² CPUC Support Tables: Effective Useful Life and Remaining Useful Life. Air Filter Alarm. Accessed on June 11, 2024. <https://www.caetrm.com/cpuc/table/effusefullife/>

²⁷³ Assumes an average price of \$1.08 for fiberglass and \$9.41 for pleated, plus \$6.25 in labor (based on 15 minutes, including portion of travel time, and \$25 per hour, which is in line with the typical prevailing wage of a General Laborer, as per Annual Wage Order No. 23 documents published by the Missouri Department of Labor). Average filter costs sourced from “Air Filter Testing, Listing, and Labeling,” Docket #12-AAER-2E prepared for the California Energy Commission, July 23, 2013.

Where:

Factor	Term	School Value
%Heating	Fraction of participants with central heating	Actual
%AC	Fraction of participants with central cooling	Actual
kW _{motor}	Average motor full load electric demand (kW)	0.5 ²⁷⁴
EFLH _{heat}	Equivalent Full Load Hours (EFLH) Heating (hours/year) – SF or MF	1496 ²⁷⁵
EFLH _{cool}	Equivalent Full Load Hours (EFLH) Cooling (hours/year) - SF or MF	869 ²⁷⁶
EI	Efficiency Improvement (%)	10% ²⁷⁷
Leakage	% Homes outside Service Territory	0%
Persistence_long term	Measure retention after first installed	58% ²⁷⁸
ISR	In Service Rate	Actual, or if unknown, assume 100% ²⁷⁹

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW = \Delta kWh * CF$

Where:

- ΔkWh

= Electric energy savings, as calculated above
- CF

= Summer peak coincidence demand (kW) to annual energy (kWh) factor
- = 0.0004660805

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

²⁷⁴ Ameren Missouri EE Kits Evaluation PY2018, page 41, <https://www.efis.psc.mo.gov/Document/Display/15870>.
²⁷⁵ Ameren Missouri EE Kits Evaluation PY2018, page 41, <https://www.efis.psc.mo.gov/Document/Display/15870>.
²⁷⁶ Ameren Missouri EE Kits Evaluation PY2018, page 41, <https://www.efis.psc.mo.gov/Document/Display/15870>.
²⁷⁷ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 233. Based on Energy.gov website; “Maintaining Your Air Conditioner”, which states that replacing a dirty air filter with a clean one can lower total air conditioner energy consumption by 5-15%. See <https://www.energy.gov/energysaver/maintaining-your-air-conditioner>; ‘Maintaining Your Air Conditioner _ Department of Energy.pdf’.
²⁷⁸ Ameren Missouri Community Savers Evaluation: PY2018, . <https://www.efis.psc.mo.gov/Document/Display/36053>, page 27.
²⁷⁹ Ameren Missouri EE Kits PY18 Evaluation, <https://www.efis.psc.mo.gov/Document/Display/15870>, page 41.

3.4.10 Packaged Terminal Air Conditioner (PTAC) and Packaged Terminal Heat Pump (PTHP)

DESCRIPTION

A PTAC is a packaged terminal air conditioner that cools and provides heat through an electric resistance heater (heat strip). A PTHP is a packaged terminal heat pump. A PTHP uses its compressor year-round to heat or cool. In warm weather, it efficiently captures heat from inside a space and pumps it outside for cooling. In cool weather, it captures heat from outdoor air and pumps it into a space, adding heat from electric heat strips as necessary to provide heat.

This measure was developed to be applicable to the following program types: TOS, NC, and EREP.

This measure characterizes:

1. TOS: the purchase and installation of a new efficient PTAC or PTHP.
2. EREP: For the early replacement baseline to apply, program participants or installation contractors must provide documentation asserting that the existing units operated when turned on—whether or not they provided cooling—and the existing unit brand name and model number must be documented; or, if the nameplate is not readable, a photograph of the unit in context and the nameplate specifically must be provided. Savings are calculated between existing unit and efficient unit consumption during the remaining life of the existing unit, and between new baseline unit and efficient unit consumption for the remainder of the measure life. The measure is only valid for non-fuel switching installations – for example replacing a cooling only PTAC with a PTHP can currently not use the TRM.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, the efficient equipment is assumed to be PTACs or PTHPs that exceed baseline efficiencies.

DEFINITION OF BASELINE EQUIPMENT

TOS: the baseline condition is defined by the Code of Federal Regulations at 10 CFR 431.97(c), section §431.97.

EREP: the baseline is the existing PTAC or PTHP for the assumed remaining useful life of the unit and the new baseline as defined above for the remainder of the measure life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years.²⁸⁰

Remaining life of existing equipment is assumed to be 5 years.²⁸¹

DEEMED MEASURE COST

TOS: The incremental capital cost for this equipment is estimated to be \$84/ton.²⁸²

EREP: The measure cost is the full cost of removing the existing unit and installing a new one. The actual program cost should be used; if unknown, assume \$1,047 per ton.²⁸³

The assumed deferred cost (after 5 years) of replacing existing equipment with new baseline unit is assumed to be \$1,039 per ton.²⁸⁴ This future cost should be discounted to present value using a 2.31% nominal societal discount rate, based on the ten year average (1/1/2014 – 12/31/2023) of the 10 year Treasury bond yield rates.²⁸⁵

LOADSHAPE

Cooling RES

Heating RES

²⁸⁰ Measure Life Report: Residential and Commercial/Industrial Lighting and HVAC Measures, GDS Associates, Inc., June 2007, <https://energizect.com/sites/default/files/documents/Measure%20Life%20Report%202007.pdf>; ‘Measure Life Report 2007.pdf’, page 1-4.

²⁸¹ Standard assumption of one third of effective useful life.

²⁸² DEER 2008. This assumes that baseline shift from IECC 2012 to IECC 2015 carries the same incremental costs. Values should be verified during evaluation.

²⁸³ Based on DCEO – IL PHA Efficient Living Program data.

²⁸⁴ Based on subtracting TOS incremental cost from the DCEO data and incorporating inflation rate of 1.91%.

²⁸⁵ ‘Societal_Discount_Rate_Calculation_08082024.xlsx’.

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Electric savings for PTACs and PTHPs should be calculated using the following algorithms

Time of sale:

$$\Delta kWh = [(((EFLH_{cool} * Capacity_{cool} * (1/SEER2_{base} - 1/SEER2_{ee})) / 1000) + ((EFLH_{heat} * Capacity_{heat} * (1/HSPF2_{base} - 1/HSFP2_{ee})) / 1,000))] * ISR$$

Early replacement:²⁸⁶

ΔkWh for remaining life of existing unit:

$$= [(((EFLH_{cool} * Capacity_{cool} * (1/SEER2_{exist} - 1/SEER2_{ee})) / 1000) + ((EFLH_{heat} * Capacity_{heat} * (1/HSPF2_{exist} - 1/HSFP2_{ee})) / 1,000))] * ISR$$

ΔkWh for remaining measure life:

$$=[(((EFLH_{cool} * Capacity_{cool} * (1/SEER2_{base} - 1/SEER2_{ee})) / 1000) + ((EFLH_{heat} * Capacity_{heat} * (1/HSPF2_{base} - 1/HSFP2_{ee})) / 1,000))] * ISR$$

Where:

Capacity_{heat} = Heating capacity of the unit in Btu/hr

= Actual

EFLH_{heat} = Equivalent Full Load Hours for heating.

= Custom input if program or regional evaluation results are available, otherwise, per the following table:

Weather Basis (Ameren Missouri Average)	EFLH _{heat} ²⁸⁷
SF or MF	1040

HSPF2_{ee} = HSPF rating of new equipment

= Actual installed

HSPF2_{base} =Heating System Performance Factor of baseline unit (kBtu/kWh)

Equipment Type	Unit Size	Federal Regulations Minimum Efficiency (HSPF)
PTHP (Heating mode)	Standard Sized	$(3.7 - (0.052 * Capacity_{heat} / 1,000)) * 3.41$
PTHP (Heating mode)	Non-Standard Size*	$(2.9 - (0.026 * Capacity_{heat} / 1,000)) * 3.41$

HSPF2_{exist} = Actual HSPF rating of existing equipment. If unknown, assume:

Existing Equipment Type	HSPF2 _{exist}
Electric resistance heating (PTAC)	3.41 ²⁸⁸
PTHP	5.44 ²⁸⁹

Capacity_{cool} = the cooling capacity of the ductless heat pump unit in Btu/hr.²⁹⁰

²⁸⁶ The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a first year savings (using the first equation) and then a “number of years to adjustment” and “savings adjustment” input which would be the (new base to efficient savings)/(existing to efficient savings).

²⁸⁷ Base values reported in *All-Electric Homes PY6 Metering Results: Multifamily HVAC Systems*, Cadmus, October 2015, Ameren. Illinois were adjusted to fit Missouri climate zones by a comparison of relative annual heating and cooling degree hours (base 65). See ‘3.4.8 EFLH 06022016.xlsx’ for derivation. FLH values are based on metering of multifamily units that were used as the primary heating source to the whole home, and in buildings that had received weatherization improvements. A DMSHP installed in a single-family home may be used more sporadically, especially if the DMSHP serves only a room, and buildings that have not been weatherized may require longer hours. Additional evaluation is recommended to refine the EFLH assumptions for the general population.

²⁸⁸ Electric resistance has a COP of 1.0 which equals 1/0.293 = 3.41 HSPF.

²⁸⁹ This is estimated based on finding the average HSPF/SEER ratio from the AHRI directory data (using the least efficient models – SEER 12 and SEER 13) – 0.596 and applying to the average nameplate SEER rating of all early replacement qualifying equipment in Ameren PY2003-PY2004. This estimation methodology appears to provide a result within 10% of actual HSPF.

²⁹⁰ 1 Ton = 12 kBtu/hr.

SEER2_{ee} = Actual installed
= SEER rating of new equipment
= Actual installed²⁹¹
SEER2_{base} = SEER2 rating of the baseline unit (kBtu/kWh). When using the formulas in the table below, convert the baseline EER to SEER2 using the EER conversion formula.²⁹²

Equipment Type	Unit Size	Federal Regulations Minimum Efficiency (EER)
PTAC (Cooling mode)	Standard Sized	14.0 – (0.300 * Capacity _{cool} / 1,000)
PTAC (Cooling mode)	Non-Standard Size*	10.9 – (0.213 * Capacity _{cool} / 1,000)
PTHP (Cooling mode)	Standard Sized	14.0 – (0.300 * Capacity _{cool} / 1,000)
PTHP (Cooling mode)	Non-Standard Size*	10.8 – (0.213 * Capacity _{cool} / 1,000)

* Non-Standard Size apply only to units with existing sleeves less than 16 inches (406mm) in height and less than 42 inches (1067 mm) in width.

SEER2_{exist} = Actual SEER rating of existing equipment. If unknown, assume:

Existing Cooling System	SEER2 _{exist} ²⁹³
PTHP	6.91
PTAC	6.53

EFLH_{cool} = Equivalent Full Load Hours for cooling.
= Custom input if program or regional evaluation results are available, otherwise, per the following table.²⁹⁴

Weather Basis (Ameren Missouri Average)	EFLH _{cool}
SF or MF	617

ISR = In-service rate. Actual, or if unknown, assume 100%²⁹⁵

SUMMER COINCIDENT PEAK DEMAND SAVINGS

Time of sale:

$\Delta kW = \Delta kWh_{Cooling} * CF$

Where:

$\Delta kWh_{Cooling}$ = Electric energy savings for cooling, calculated above
CF = 0.0009474181

NATURAL GAS ENERGY SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

²⁹¹ If only an EER2 rating is available, use the following conversion equation to estimate SEER2; SEER = (1.12 + (1.2544 - 0.08 * EER)^{0.5}) / 0.04. This is the inverse of EER = (- 0.02 * SEER²) + (1.12 * SEER). From Wassmer, M. (2003), “A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations,” (Masters thesis), University of Colorado at Boulder.

²⁹² Ibid.

²⁹³ ASHP existing efficiency assumes degradation and is sourced from the Ameren Missouri Heating and Cooling Program Impact and Process Evaluation: Program Year 2015. CAC assumed to follow the same trend in degradation as the ASHP: 9.12 SEER nameplate to 7.2 (6.91 SEER2) operations SEER represents degradation to 78.9% of nameplate. 78.9% of 8.6 SEER CAC nameplate gives an operational SEER of 6.8 (6.53 SEER2).

²⁹⁴ Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations.

²⁹⁵ Ameren Missouri HVAC Evaluation: PY2020, <https://www.efis.psc.mo.gov/Document/Display/13831>, page 53.

DEEMED O&M COST ADJUSTMENT CALCULATION
N/A

MEASURE CODE:

3.4.11 Room Air Conditioner

DESCRIPTION

This measure relates to the purchase and installation of a room air conditioning unit that meets the ENERGY STAR® minimum qualifying efficiency specifications, in place of a baseline unit meeting minimum federal standard efficiency ratings presented below:²⁹⁶

Product Type and Class (Btu/hr)		Federal Standard with louvered sides (CEER) ²⁹⁷	Federal Standard without louvered sides (CEER)	ENERGY STAR® v4.0 / CEE Tier 1 with louvered sides (CEER) ²⁹⁸	ENERGY STAR® v4.0 / CEE Tier 1 without louvered sides (CEER)	CEE Tier 2 (CEER) ²⁹⁹
Without Reverse Cycle	< 8,000	11.0	10.0	12.1	11.0	12.7
	8,000 to 10,999	10.9	9.6	12.0	10.6	12.5
	11,000 to 13,999	10.9	9.5	12.0	10.5	12.5
	14,000 to 19,999	10.7	9.3	11.8	10.2	12.3
	20,000 to 27,999	9.4	9.4	10.3	10.3	10.8
	>=28,000	9.0	9.4	9.9	10.3	10.4
With Reverse Cycle	<14,000	9.8	9.3	10.8	10.2	12.5
	14,000 to 19,999	9.8	8.7	10.8	9.6	12.3
	>=20,000	9.3	8.7	10.2	9.6	10.4
Casement only		9.5		10.5		
Casement-Slider		10.4		11.4		

This measure was developed to be applicable to the following program type: TOS and EREP.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

To qualify for this measure the new room air conditioning unit must meet the ENERGY STAR® efficiency standards presented above.

DEFINITION OF BASELINE EQUIPMENT

For programs other than low-income programs, the baseline assumption is a new room air conditioning unit that meets the current minimum federal efficiency standards presented above.

For low income programs, for both Time of Sale and Early Replacement the baseline assumption is an inefficient unit either existing in the home or being purchased or acquired via the secondary market.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be 12 years.³⁰⁰

For low income programs, since the baseline unit is assumed to be purchased from the secondary market, it is assumed that the remaining life of the baseline unit is 6 years and would need to be replaced with another unit from the secondary market at that point.³⁰¹

²⁹⁶Side louvers that extend from a room air conditioner model in order to position the unit in a window. A model without louvered sides is placed in a built-in wall sleeve and are commonly referred to as "through-the-wall" or "built-in" models.

Casement-only refers to a room air conditioner designed for mounting in a casement window of a specific size.

Casement-slider refers to a room air conditioner with an encased assembly designed for mounting in a sliding or casement window of a specific size. Reverse cycle refers to the heating function found in certain room air conditioner models. https://www.energystar.gov/products/heating_cooling/air_conditioning_room/key_product_criteria

²⁹⁷ See DOE's Appliance and Equipment Standards for Room AC;

²⁹⁸ ENERGY STAR Version 4.0 Room Air Conditioners Program Requirements

²⁹⁹ The Consortium for Energy Efficiency Super Efficient Home Appliance Initiative, Room Air Conditioner Specification, CEE Advanced Tier (CEER), effective January 31, 2017. Please see file 'CEE_ResApp_RoomAirConditionerSpecification_2017.pdf'.

https://library.cee1.org/system/files/library/13069/CEE_ResApp_RoomAirConditionerSpecification_2017.pdf

³⁰⁰ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, <https://energizect.com/sites/default/files/documents/Measure%20Life%20Report%202007.pdf>; 'Measure Life Report 2007.pdf', page 1-3.

³⁰¹ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 68.

DEEMED MEASURE COST

For programs other than low-income programs, the incremental cost for this measure is assumed to be \$40 for a CEER Tier 1 or ENERGY STAR unit and \$100 for a CEE Tier 2 unit.³⁰²

For low income programs, the actual full cost of the ENERGY STAR[®] unit should be used. If unavailable assume \$300.³⁰³ If a CEE Tier 2 unit is installed assume \$508.³⁰⁴

LOADSHAPE
Cooling RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$\Delta kWh = (FLH_{RoomAC} * Btuh * (1/CEER_{base} - 1/CEER_{ce})) / 1,000$

Where:

FLH_{RoomAC} = Full Load Hours of room air conditioning unit:

Weather Basis (City based upon)	Hours ³⁰⁵
St Louis, MO	860 for primary use and 556 for secondary use

Btu/H = Size of unit
= Actual. If unknown assume 8,500 Btu/hr ³⁰⁶

CEER_{base} = Efficiency of baseline unit
= For programs other than low-income programs, as provided in tables above
= For low income programs, actual CEER of the existing unit; if unknown, assume 7.7³⁰⁷

CEER_{ce} = Efficiency of ENERGY STAR[®] unit
= Actual. If unknown assume minimum qualifying standard as provided in tables above

ISR = In Service Rate
=Actual, or if unknown, reference values in the table below dependent on program type

Program Type	ISR
SFIE ³⁰⁸	98%

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW = \Delta kWh * CF$

³⁰²Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 41. CEE Tier 1 cost based on field study conducted by Efficiency Vermont and Tier 2 based on professional judgement.

³⁰³ Ibid.

³⁰⁴ Consistent with Non IQ version of the measure.

³⁰⁵ Primary is based upon Ameren Missouri PY2013 CoolSavers Evaluation data, Secondary is based upon Ameren Missouri Efficient Products PY2016 Evaluation, <https://www.efis.psc.mo.gov/Document/Display/17349>, page 64.

³⁰⁶Based on maximum capacity average from the RLW Report: Final Report Coincidence Factor Study Residential Room Air Conditioners, June 23, 2008, https://www.puc.nh.gov/Electric/Monitoring%20and%20Evaluation%20Reports/National%20Grid/122_SPWG%20Room%20%20AC%20Evaluation%20FINALReport%20June%202023%20ver7.pdf; ‘122_SPWG Room AC Evaluation FINALReport June 23 ver7.pdf’.

³⁰⁷ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 70.

³⁰⁸Ameren Missouri Efficient Products Evaluation PY2016, <https://www.efis.psc.mo.gov/Document/Display/17349>, page 63.

Where:

ΔkWh = Electric energy savings, as calculated above
CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0009474181³⁰⁹

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

³⁰⁹ Based on Ameren Missouri TRM Volume 1 - Appendix G: "Table 1 – Residential End Use Category Monthly Shapes and Coincident Peak Factors"for residential cooling end-use.

3.4.12 Ground Source Heat Pump

DESCRIPTION

A heat pump provides heating or cooling by moving heat between indoor and the ground.

This measure characterizes:

1. TOS: The installation of a new residential sized ground source heat pump. This could relate to the replacement of an existing unit at the end of its useful life, or the installation of a new system in a new home.
2. EREP: The early removal of functioning electric heating and cooling systems from service, prior to its natural end of life, and replacement with a new high efficiency ground source heat pump unit. To qualify as early replacement, the existing unit must be operational when replaced. If the SEER of the existing unit is known, the baseline SEER is the actual SEER value of the unit replaced, and if unknown use assumptions in the variable list below (SEER_{2exist} and HSPF_{2exist}). If the operational status of the existing unit is unknown, use TOS assumptions.

This measure was developed to be applicable to the following program types: TOS, NC, and EREP.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

A new residential sized ground source heat pump with specifications to be determined by program.

DEFINITION OF BASELINE EQUIPMENT

The baseline for the TOS measure is federal standard efficiency level as of: 14.3 SEER2 and 7.5 HSPF2 when replacing an existing air source heat pump or existing ground source heat pump, and 13.4 SEER2 and 3.41 HSPF2 when replacing a central air conditioner and electric resistance heating.

For the early replacement baseline to apply, program participants or installation contractors must provide documentation asserting that the existing units operated when turned on—whether or not they provided cooling—and the existing unit brand name and model number must be documented; or, if the nameplate is not readable, a photograph of the unit in context and the nameplate specifically must be provided. The baseline for the early replacement measure is the efficiency of the existing equipment for the assumed remaining useful life of the unit and the new baseline as defined above for the remainder of the measure life.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 25 years.³¹⁰

For early replacement, the remaining life of existing equipment is assumed to be 6 years for GSHP, ASHP and CAC and 25 years for electric resistance.

DEEMED MEASURE COST

New Construction and Time of Sale: The actual installed cost of the Ground Source Heat Pump should be used (default of \$3957 per ton),³¹¹ minus the assumed installation cost of the baseline equipment (\$6562 + \$600 per ton for ASHP³¹² or \$2011 for a new baseline 80% AFUE furnace³¹³ and \$3,338 for new baseline Central AC replacement³¹⁴).

Early Replacement: The actual full installation cost of the Ground Source Heat Pump should be used (default of \$3957 per ton).

The assumed deferred cost of replacing existing equipment with a new baseline unit is assumed to be \$7,527 + \$688 per ton for a new baseline Air Source Heat Pump, or \$2,296 for a new baseline 80% AFUE furnace and \$3,670 for new baseline Central AC replacement.³¹⁵ This future cost should be discounted to present value using a 2.31% nominal societal discount rate, based on the ten year average (1/1/2014 – 12/31/2023) of the 10 year Treasury bond yield rates.³¹⁶

³¹⁰ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, p. 166. System life of indoor components as per DOE estimate (see 'Geothermal Heat Pumps Department of Energy.htm'). The ground loop has a much longer life, but the compressor and other mechanical components are the same as an ASHP.

³¹¹ Based on data provided in 'Results of HomeE geothermal and air source heat pump rebate incentives documented by IL electric cooperatives'.

³¹² Full install ASHP costs are based upon data provided by Ameren. See 'ASHP Costs_06242022'.

³¹³ See 'Technical Standard Document_APPENDIX_E.pdf'.

³¹⁴ See 'CAC Costs 09.02.2024.xlsx'.

³¹⁵ All baseline replacement costs are consistent with their respective measures and include inflation rate of 1.91%.

³¹⁶ 'Societal_Discount_Rate_Calculation_08082024.xlsx'.

LOADSHAPE
Cooling RES
Heating RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

TOS:

$$\Delta kWh = \frac{[(EFLH_{cool} * Capacity_{cool} * (1/EER2_{base} - 1/EER2_{ee}) / 1000) + (EFLH_{heat} * Capacity_{heat} * (1/HSPF2_{base} - 1/HSFP2_{ee}) / 1000)]}{ISR}$$

EREP:³¹⁷

ΔkWh for remaining life of existing unit (1st 6 years for replacing an ASHP or GSHP, 18 years for replacing electric resistance):

$$= \frac{[(EFLH_{cool} * Capacity_{cool} * (1/EER2_{exist} - 1/EER2_{ee}) / 1000) + (EFLH_{heat} * Capacity_{heat} * (1/HSPF2_{exist} - 1/HSFP2_{ee}) / 1000)]}{ISR}$$

ΔkWh for remaining measure life (next 12 years if replacing an ASHP or GSHP):

$$= \frac{[(EFLH_{cool} * Capacity_{cool} * (1/EER2_{base} - 1/EER2_{ee}) / 1000) + (EFLH_{heat} * Capacity_{heat} * (1/HSPF2_{base} - 1/HSFP2_{ee}) / 1000)]}{ISR}$$

Where:

EFLH_{cool} = Equivalent full load hours of air conditioning:³¹⁸

Weather Basis (City based upon)	EFLH _{cool} (Hours)
St Louis, MO	869

Capacity_{cool} = Cooling capacity of air source heat pump (Btu/hr)

= Actual (1 ton = 12,000Btu/hr)

EER2_{exist} = Seasonal Energy Efficiency Ratio of existing unit (kBtu/kWh)

= Use actual SEER2³¹⁹ rating where it is possible to measure or reasonably estimate. If using rated efficiencies, derate the efficiency value based on the age of the existing equipment (up to a maximum of 30 years) to account for degradation over time.³²⁰ If age is unknown, use 12 years.

= SEER2 * (1-0.01)^{Age}

If rated efficiency is unknown, use defaults provided below, which should not be further adjusted to account for age-related degradation:

³¹⁷ The two equations are provided to show how savings are determined during the initial phase of the measure (existing to efficient) and the remaining phase (new baseline to efficient). In practice, the screening tools used may either require a first year savings (using the first equation) and then a “number of years to adjustment” and “savings adjustment” input which would be the (new base to efficient savings)/(existing to efficient savings).

³¹⁸ PY2019 Evaluation Report, <https://www.efis.psc.mo.gov/Document/Display/15876>, page 30.

³¹⁹ Part load EER2 is paired with SEER2exist, consistent with the approach presented in section 3.4.2, Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, p. 508.

³²⁰ Ibid., page 112. Justification for degradation factors can be found on page 14 of ‘AIC HVAC Metering Study Memo FINAL 2_28_2018.docx’. Estimate efficiency as (Rated Efficiency * (1-0.01)^{Equipment Age}).

Existing Cooling System	SEER _{2exist}
Air Source Heat Pump	6.91 ³²¹
Ground Source Heat Pump	13.4 ³²²
Central AC	6.53
No central cooling ³²³	Let '1/SEER _{2exist} ' = 0

- EER_{2base} = Seasonal Energy Efficiency Ratio of baseline Air Source Heat Pump (kBtu/kWh)
= 14.3 if replacing air source heat pump or ground source heat pump; 13.4 if replacing central air conditioner
- EER_{2ee} = Seasonal Energy Efficiency Ratio of efficient Air Source Heat Pump (kBtu/kWh)
= Actual
- EFLH_{heat} = Equivalent full load hours of heating
= Dependent on location:³²⁴

Weather Basis (City based upon)	EFLH _{heat} (Hours)
St Louis, MO	1496

- Capacity_{heat} = Heating Capacity of Air Source Heat Pump (Btu/hr)
= Actual (1 ton = 12,000Btu/hr)
- HSPF_{2exist} = Heating System Performance Factor of existing heating system (kBtu/kWh)
= Use actual HSPF2 rating where it is possible to measure or reasonably estimate. If using rated efficiencies, derate the efficiency value based on the age of the existing equipment (up to a maximum of 30 years) to account for degradation over time.³²⁵ If age is unknown, use 12 years.
= HSPF2 * (1-0.01)^{Age}
= If rated efficiency is unknown, use defaults provided below, which should not be further adjusted to account for age-related degradation:³²⁶

Existing Heating System	HSPF _{2exist}
Air Source Heat Pump	4.91
Ground Source Heat Pump	7.5
Electric Resistance	3.41

- HSPF_{2base} = Heating System Performance Factor of baseline Air Source Heat Pump (kBtu/kWh)
= 7.5 if replacing air source heat pump or ground source heat pump; 3.41 if replacing electric resistance heating
- HSFP_{2ee} = Heating System Performance Factor of efficient Air Source Heat Pump (kBtu/kWh)
- ISR = In-service rate. Actual, or if unknown, assume 100%³²⁷

SUMMER COINCIDENT PEAK DEMAND SAVINGS

TOS:

$$\Delta kW = \Delta kWh * CF$$

³²¹ ASHP existing efficiency assumes degradation and is sourced from the Ameren Missouri Heating and Cooling Program Impact and Process Evaluation: Program Year 2015. CAC assumed to follow the same trend in degradation as the ASHP: 9.12 SEER nameplate to 7.2 (6.91 SEER2) operations SEER represents degradation to 78.9% of nameplate. 78.9% of 8.6 SEER CAC nameplate gives an operational SEER of 6.8 (6.53 SEER2).

³²² Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 169.

³²³ If there is no central cooling in place but the incentive encourages installation of a new ASHP with cooling, the added cooling load should be subtracted from any heating benefit.

³²⁴ PY2019 Residential Evaluation Report, <https://www.efis.psc.mo.gov/Document/Display/15876>, page 30.

³²⁵ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 112. Justification for degradation factors can be found on page 14 of 'AIC HVAC Metering Study Memo FINAL 2_28_2018.docx'. Estimate efficiency as (Rated Efficiency * (1-0.01)^{Equipment Age}).

³²⁶ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 171.

³²⁷ Ameren Missouri HVAC Evaluation: PY2020, <https://www.efis.psc.mo.gov/Document/Display/13831>, page 53.

Where:

- ΔkWh = Electric energy savings, as calculated above
- CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
- = 0.0009474181

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

3.5 Lighting

3.5.1 LED Screw Based Omnidirectional Bulb

DESCRIPTION

This measure provides savings assumptions for LED screw-based omnidirectional (e.g., A-Type) lamps installed in a known location (i.e., residential and in-unit interior or exterior) or, if the implementation strategy does not allow for the installation location to be known (e.g., an upstream retail program or efficiency kit), an unknown residential location. For upstream programs, utilities should develop an assumption of the Residential v Commercial split and apply the relevant assumptions to each portion.

Federal legislation stemming from the Energy Independence and Security Act of 2007 (EISA) requires all general-purpose light bulbs between 40W and 100W to be approximately 30% more energy efficient than standard incandescent bulbs. Production of 100W, standard efficacy incandescent lamps ended in 2012, followed by restrictions on 75W lamps in 2013 and 60W and 40W lamps in 2014. The baseline for this measure has therefore become bulbs (improved incandescent or halogen) that meet the new standard.

A provision in the EISA regulations required that by January 1, 2020, all lamps meet efficiency criteria of at least 45 lumens per watt, in effect making the baseline equivalent to a current day CFL. In 2019, the Department of Energy issued a final determination and clarified that a) the EISA backstop provision had not been triggered and therefore b) the efficiency standard would not change in 2020. In May 2022, DOE reversed this decision by issuing a final rule that expanded the General Service Lamp (GSL) and General Service Incandescent Lamp (GSIL) definitions and reinstated the 45 lumen per watt backstop provision with phased enforcement between January 2023 and July 2023.³²⁸

This measure was developed to be applicable to the following program types: TOS, NC, and RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this measure to apply, new lamps must be ENERGY STAR[®] labeled based upon the ENERGY STAR[®] specification v2.0 which became effective on 1/2/2017.³²⁹

Qualification could also be based on the Design Light Consortium's qualified product list.³³⁰

DEFINITION OF BASELINE EQUIPMENT

In 2012, Federal legislation stemming from the Energy Independence and Security Act of 2007 (EISA) will require all general-purpose light bulbs between 40 watts and 100 watts to have ~30% increased efficiency, essentially phasing out standard incandescent technology. In 2012, the 100 w lamp standards apply; in 2013 the 75 w lamp standards will apply, followed by restrictions on the 60 w and 40 w lamps in 2014. Since measures installed under this TRM all occur after 2014, baseline equipment are the values after EISA. These are shown in the baseline table below.

Additionally, an EISA backstop provision was included that would require replacement baseline lamps to meet an efficacy requirement of 45 lumens/watt or higher beginning on 1/1/2020. In December 2019, DOE issued a final determination for General Service Incandescent Lamps (GSILs), finding that this more stringent standard was not economically justified. However, in May 2022 DOE reversed this decision by issuing a Final rule for both the broadened General Service Lamp definition as well as the implementation of the 45 lumen per watt backstop. DOE stated that it will use its enforcement discretion to minimize impacts on the supply chain and effectively allow companies to continue the manufacture and import of noncompliant bulbs through the remainder of 2022, and allow retailers to continue selling them with limited enforcement until July 2023.

No savings are claimed for non-income qualified programs unless via direct install programs.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be two years for Direct Install in non-income eligible populations and eight years for income eligible populations.³³¹

DEEMED MEASURE COST

The deemed measures cost for a LED screw based omnidirectional bulb is \$1.45 per bulb.³³²

³²⁸ DOE 87 FR 27439

³²⁹ https://www.energystar.gov/sites/default/files/ENERGY%20STAR%20Lamps%20V2_0%20Revised%20AUG-2016.pdf; 'ENERGY STAR Lamps V2_0 Revised AUG-2016.pdf'.

³³⁰ <https://www.designlights.org/QPL>.

³³¹ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 327.

³³² Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 329.

LOADSHAPE
Lighting RES
Lighting BUS

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$\Delta kWh = (Watt_{Base} - Watt_{EE}) * ISR * (1 - LKG) * Hours * WHF / 1,000$

Where:

$Watt_{Base}$ = Based on lumens of LED bulb installed. If lumens of LED bulb are unknown, refer to table below.
 $Watt_{EE}$ = Actual wattage of LED purchased / installed - If unknown, use default provided below.³³³

Minimum Lumens	Maximum Lumens	LED Wattage (WattsEE)	Baseline (WattsBase)	Delta Watts (WattsEE)
310	399	4.0	25	21.0
400	749	6.6	29	22.4
750	899	9.6	43	33.4
900	1,399	13.1	53	39.9
1,400	1,999	16.0	72	56.0
2,000	2,999	21.8	150	128.2
3,000	3,299	28.9	200	171.1

LKG = leakage rate (program bulbs installed outside Ameren Missouri's service area)
= Actual, or if unknown, assume 0%³³⁴

ISR = In Service Rate, the percentage of units rebated that are actually in service. Actual, or if unknown, assume:

Program	Discounted In Service Rate (ISR)
Direct Install (MFLI) ³³⁵	98.2%
Efficiency Kit (MF) ³³⁶	100%
Low Income Kits	90%
Pay As You Save ³³⁷	87%

Hours = Average hours of use per year for bulbs in residential homes. Use custom value or table below.

³³³ Ibid., page 328.
³³⁴ Assumed based on program delivery channels.
³³⁵ Ameren Missouri Community Savers Evaluation: PY2018, . <https://www.efis.psc.mo.gov/Document/Display/36053>, page 17.
³³⁶ Ameren Missouri Efficient Kits Impact and Process Evaluation: Program Year 2018, <https://www.efis.psc.mo.gov/Document/Display/15870>, page 38.
³³⁷ Ameren Missouri Pay As You Save (PAYS®) Evaluation: PY2022 Participant Survey

Program	HOU Res
Residential	995.18 ³³⁸
Efficient Kits	995.18
Income Eligible RES	674.18 ³³⁹
MFMR	693.50 ³⁴⁰

WHFe = Waste Heat Factor for energy to account for the impact from reducing waste heat from efficient lighting on electric cooling and heating loads in residential homes.

= 0.99 if unknown³⁴¹

WHFe_{Heat} = Waste Heat Factor for energy to account for electric heating increase from reducing waste heat from efficient lighting (if fossil fuel heating, see calculation of heating penalty in that section).

= $1 - ((HF / \eta_{Heat}) * \%ElecHeat)$.

= If unknown assume 0.88³⁴²

Where:

HF = Heating Factor or percentage of light savings that must now be heated

= 53%³⁴³ for interior or unknown location

= 0% for exterior or unheated location

$\eta_{HeatElectric}$ = Efficiency in COP of Heating equipment

= Actual - If not available, use:³⁴⁴

System Type	Age of Equipment	HSPF2 Estimate	η_{Heat} (COP Estimate) = (HSPF2/3.413)*0.85
Heat Pump (if age unknown assume 2006-2014)	Before 2006	5.8	1.44
	After 2006 - 2014	6.5	1.62
	2015 on	7.0	1.74
Resistance	N/A	N/A	1.00
Unknown ³⁴⁵	N/A	N/A	1.28

%ElecHeat = Percentage of heating savings assumed to be electric

Heating fuel	%ElectricHeat
Electric	100%
Natural Gas	0%
Unknown	35% ³⁴⁶

WHFe_{Cool} = Waste Heat Factor for energy to account for cooling savings from reducing waste heat from efficient lighting

³³⁸ Ameren Missouri Lighting Evaluation PY2018, <https://www.efis.psc.mo.gov/Document/Display/15873>, page 36.

³³⁹ Ameren Missouri Community Savers Evaluation PY2018 workpapers- Weighted Avg. HOU from ADM workpapers.

³⁴⁰ ADM 2017 Community Savers EM&V

³⁴¹ Ameren Missouri PY2014 Evaluation, <https://www.efis.psc.mo.gov/Document/Display/14194>, page 45.

³⁴² Calculated using defaults: $1 - ((0.53/1.57) * 0.35) = 0.88$.

³⁴³ This means that heating loads increase by 53% of the lighting savings. This is based on the average result from REMRate modeling of several different building configurations in Iowa (Des Moines, Mason City, and Burlington). These results were judged to be equally applicable to Missouri.

³⁴⁴ These default system efficiencies are based on the applicable minimum federal standards. In 2006 and 2015, the federal standard for heat pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

³⁴⁵ Calculation assumes 35% Heat Pump and 65% Resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, <https://www.eia.gov/consumption/residential/data/2009/hc/hc6.9.xls>; 'hc6.9.xls'. Average efficiency of heat pump is based on assumption that 50% are units from before 2006 and 50% from 2006-2014. Program or evaluation data should be used to improve this assumption if available.

³⁴⁶ Average (default) value of 35% electric space heating from 2009 Residential Energy Consumption Survey for Missouri. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then they should be used.

Bulb Location	WHFeCool
Building with cooling	1.12 ³⁴⁷
Building without cooling or exterior	1.0
Unknown	1.11 ³⁴⁸

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW = \Delta kWh * CF$

Where:

- ΔkWh = Electric energy savings, as calculated above.
- CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0001492529 for Lighting RES (Residential)

NATURAL GAS SAVINGS

Heating Penalty for Natural Gas heated homes:³⁴⁹

$\Delta Therms = -((Watts_{Base} - Watts_{EE}) / 1,000 * ISR * Hours * HF * 0.03412) / \eta_{Heat} * \%GasHeat$

Where:

- HF = Heating Factor or percentage of light savings that must now be heated
= 53%³⁵⁰ for interior or unknown location
= 0% for exterior or unheated location
- 0.03412 = Converts kWh to therms
- $\eta_{HeatGas}$ = Efficiency of heating system
= 71%³⁵¹
- %GasHeat = Percentage of heating savings assumed to be natural gas

Heating fuel	%GasHeat
Electric	0%
Natural Gas	100%
Unknown	65% ³⁵²

MEASURE CODE:

³⁴⁷ The value is estimated at 1.12 (calculated as 1 + (0.34 / 2.8)), and it is based on cooling loads decreasing by 34% of the lighting savings (average result from REMRate modeling of several different building configurations in Iowa (Des Moines, Mason City, and Burlington)). The estimate also assumes typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm (-0.02 * SEER²) + (1.12 * SEER) (from Wassmer, M. (2003); A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to COP = EER/3.412 = 2.8COP). Results of the Iowa study are assumed to be applicable to Missouri.

³⁴⁸ The value is estimated at 1.11 (calculated as 1 + (0.91*(0.34 / 2.8)), which is based on assumption that 91% of homes have central cooling (based on 2009 Residential Energy Consumption Survey, <https://www.eia.gov/consumption/residential/data/2009/hc/hc6.9.xls>; 'hc6.9.xls') .

³⁴⁹ Negative value because this is an increase in heating consumption due to the efficient lighting.

³⁵⁰ This means that heating loads increase by 53% of the lighting savings. This is based on the average result from REMRate modeling of several different building configurations in Iowa (Des Moines, Mason City, and Burlington). Results of the Iowa study are judged to be equally applicable to Missouri.

³⁵¹ This has been estimated assuming that natural gas central furnace heating is typical for Missouri residences (the predominant heating is gas furnace with 48% of Missouri homes (based on Energy Information Administration, 2009 Residential Energy Consumption Survey)). See reference <https://www.eia.gov/consumption/residential/data/2009/hc/hc6.9.xls>; 'hc6.9.xls'. In 2000, 29% of furnaces purchased in Missouri were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years, so units purchased 15 years ago provide a reasonable proxy for the current mix of furnaces in the state. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: ((0.29*0.92) + (0.71*0.8)) * (1-0.15) = 0.71.

³⁵² Average (default) value of 65% gas space heating from 2009 Residential Energy Consumption Survey for Missouri. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then they should be used.

3.5.2 LED Specialty Lamp

DESCRIPTION

This measure provides savings assumptions for LED directional, decorative, and globe lamps when the LED is installed in a known location (i.e., residential and in-unit interior or exterior) or, if the implementation strategy does not allow for the installation location to be known (e.g., an upstream retail program or efficiency kit), an unknown residential location. For upstream programs, utilities should develop an assumption of the Residential v Nonresidential split and apply the relevant assumptions to each portion.

Federal legislation stemming from the Energy Independence and Security Act of 2007 (EISA) requires all general-purpose light bulbs between 40W and 100W to be approximately 30% more energy efficient than standard incandescent bulbs. Production of 100W, standard efficacy incandescent lamps ended in 2012, followed by restrictions on 75W lamps in 2013 and 60W and 40W lamps in 2014. The baseline for this measure has therefore become bulbs (improved incandescent or halogen) that meet the new standard.

A provision in the EISA regulations required that by January 1, 2020, all lamps meet efficiency criteria of at least 45 lumens per watt, in effect making the baseline equivalent to a current day CFL. In 2019, the Department of Energy issued a final determination and clarified that a) the EISA backstop provision had not been triggered and therefore b) the efficiency standard would not change in 2020. In May 2022, DOE reversed this decision by issuing a final rule that expanded the General Service Lamp (GSL) and General Service Incandescent Lamp (GSIL) definitions and reinstated the 45 lumen per watt backstop provision with phased enforcement between January 2023 and July 2023.³⁵³

This measure was developed to be applicable to the following program types: TOS, NC, and RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

In order for this characterization to apply, new lamps must be ENERGY STAR[®] labeled based upon the ENERGY STAR[®] specification v2.0 which became effective on 1/2/2017.³⁵⁴ Qualification could also be based on the Design Light Consortium's qualified product list.³⁵⁵

DEFINITION OF BASELINE EQUIPMENT

Starting August 1, 2023, the EISA backstop provision became effective, limiting the sale, manufacture, and import of non-compliant lamps. Therefore, the baseline condition for this measure is a reflection of the 2022 DOE final rule reinstating the 45 lumen per watt backstop provisions for all GSL and GSILs between 310 and 3,300 lumens. All other lamps, i.e., those below 310 lumens and above 3,300 lumens, the baseline condition is a reflection of products available in the market and standards agreed upon in practice.

No savings are claimed for non-income qualified programs unless via direct install programs.

Direct Install programs where it can be shown that the LED is replacing working inefficient lighting should continue to use the existing inefficient lighting as baseline and also assume a measure life of 2 years.

Direct Install programs where it can be shown that the LED is replacing working inefficient lighting should continue to use the existing inefficient lighting as baseline and also assume a measure life of 8 years.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The measure life is assumed to be two years for Direct Install in non-income eligible populations and eight years for income eligible populations.³⁵⁶

DEEMED MEASURE COST

The deemed measures cost for a specialty LED is \$1.66 per lamp.³⁵⁷

LOADSHAPE

Lighting RES

Lighting BUS

Algorithm

³⁵³ DOE 87 FR 27439.

³⁵⁴ https://www.energystar.gov/sites/default/files/ENERGY%20STAR%20Lamps%20V2_0%20Revised%20AUG-2016.pdf; 'ENERGY STAR Lamps V2_0 Revised AUG-2016.pdf'.

³⁵⁵ <https://www.designlights.org/QPL>.

³⁵⁶ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 311.

³⁵⁷ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 311.

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta kWh = (Watt_{Base} - Watt_{EE}) * ISR * (1 - LKG) * Hours * WHF / 1,000$$

Where:

- Watts_{Base}

= Based on bulb type and lumens of LED bulb installed. See tables below.
- Watts_{EE}

= Actual wattage of LED purchased / installed - If unknown, use default provided below:

Decorative Lamps – ENERGY STAR Minimum Luminous Efficacy = 65Lm/W for all lamps:

Bulb Type	Minimum Lumens	Maximum Lumens	LED Wattage (Watts _E)	Baseline (Watts _{Base})	Delta Watts (Watts _E)
Omni-Directional 3-Way	1,100	1,999	14.7	100	85.3
	2,000	2,700	22.6	150	127.4
Globe (medium and intermediate bases less than 750 lumens)	310	349	3.0	25	22
	350	499	4.7	40	35.3
	500	574	5.7	60	54.3
	575	649	6.5	75	68.5
	650	1,000	8.2	100	91.8
Globe (candelabra bases less than 1050 lumens)	310	349	3.5	25	21.5
	350	499	4.4	40	35.6
	500	574	5.5	60	54.5
Decorative (Shapes B, BA, C, CA, DC, F, G, medium and intermediate bases less than 750 lumens)	310	499	4.3	40	35.7
	500	800	5.8	60	54.2
Decorative (Shapes B, BA, C, CA, DC, F, G, candelabra bases less than 1050 lumens)	310	499	4.2	40	35.8
	500	650	5.5	60	54.5
Decorative (Shape ST)	310	499	6.5	40	33.5
	500	999	8.8	60	51.2
	1000	1500	10.0	100	90.0
Decorative (Shape S)	310	340	2.25	25	22.8

Directional Lamps - ENERGY STAR Minimum Luminous Efficacy = 70Lm/W for <90 CRI lamps and 61 Lm/W for >=90CRI lamps.
*Directional R, BR, and ER lamp types.*³⁵⁸

³⁵⁸ From pg. 13 of the ENERGY STAR Specification for lamps v2.1.

Bulb Type	Minimum Lumens	Maximum Lumens	LED Wattage (Watts _E)	Baseline (Watts _{Base})	Delta Watts (Watts _E)
Reflector lamp types with medium screw bases (PAR20, PAR30(S,L), PAR38, R40, etc.) w/ diameter >2.25" (*see exceptions below)	400	649	7.0	50	43
	650	899	10.7	75	64.3
	900	1,049	13.9	90	76.1
	1,050	1,199	13.8	100	86.2
	1,200	1,499	15.9	120	104.1
	1,500	1,999	18.9	150	131.1
	2,000	3,299	27.3	250	222.7
Reflector lamp types with medium screw bases (PAR16, R14, R16, etc.) w/ diameter <2.25" (*see exceptions below)	310	374	4.6	35	30.4
	375	600	6.4	50	43.6
*BR30, BR40, or ER40	650	949	9.3	65	55.7
	950	1,099	12.7	75	62.3
	1,100	1,399	14.4	85	70.6
	1,400	1,600	16.6	100	83.4
	1,601	1,800	22.2	120	97.8
*R20	450	524	6.0	40	34.0
	525	750	7.1	45	37.9
*MR16	310	324	3.8	20.0	16.2
	325	369	4.8	25.0	20.2
	370	400	4.9	25.0	20.1

For PAR, MR, and MRX Lamps Types:

For these highly focused directional lamp types, it is necessary to have Center Beam Candle Power (CBCP) and beam angle measurements to accurately estimate the equivalent baseline wattage. The formula below is based on the ENERGY STAR Center Beam Candle Power tool.³⁵⁹ If CBCP and beam angle information are not available or if the equation below returns a negative value (or undefined), use the manufacturer’s recommended baseline wattage equivalent.³⁶⁰

WattsBase

= 375.1 – 4.355(D) – (227,800 – (937.9 * D) – (0.9903 * D^2) – (1,479 *BA) – (12.02 * D * BA) + (14.69 * (BA^2)) - 16,720 * ln(CBCP))^0.5

Where:

- D
- = Bulb diameter (e.g. for PAR20 D = 20)
- BA
- = Beam angle
- CBCP
- = Center beam candle power

The result of the equation above should be rounded DOWN to the nearest wattage established by ENERGY STAR:

³⁵⁹ See ‘ESLampCenterBeamTool.xlsx’.

³⁶⁰ The ENERGY STAR Center Beam Candle Power tool does not accurately model baseline wattages for lamps with certain bulb characteristic combinations – specifically for lamps with very high CBCP.

Diameter	Permitted Wattages
16	20, 35, 40, 45, 50, 60, 75
20	50
30S	40, 45, 50, 60, 75
30L	50, 75
38	40, 45, 50, 55, 60, 65, 75, 85, 90, 100, 120, 150, 250

Additional EISA non-exempt bulb types:

Bulb Type	Minimum Lumens	Maximum Lumens	LED Wattage (Watts _{EE})	Baseline (Watts _{Base})	Delta Watts (Watts _{EE})
Dimmable Twist, Globe (less than 5" in diameter and > 749 lumens), candle (shapes B, BA, CA > 749 lumens), Candelabra Base Lamps (>1049 lumens), Intermediate Base Lamps (>749 lumens)	310	399	4.0	25	21.0
	400	749	6.6	29	22.4
	750	899	9.6	43	33.4
	900	1,399	13.1	53	39.9
	1,400	1,999	16.0	72	56.0

LKG

= leakage rate (program bulbs installed outside Ameren Missouri's service area)
= Actual, or if unknown, assume 0%³⁶¹

ISR

= In Service Rate, the percentage of units rebated that are actually in service. Actual, or if unknown, assume:

Program	Discounted In Service Rate (ISR)
MFIE ³⁶²	98.2%
SFIE and PAYS ³⁶³	100%

Hours

= Average hours of use per year for bulbs in residential homes. Custom, or if unknown assume 1,314³⁶⁴ for exterior, or or if interior use table below.

Program	HOU Res
Residential	995.18 ³⁶⁵
Income Eligible RES	674.18 ³⁶⁶

WHFe

= Waste Heat Factor for energy to account for the impact from reducing waste heat from efficient lighting on electric cooling and heating loads in residential homes.
= 0.99 if unknown³⁶⁷

WHFe_{Heat}

= Waste Heat Factor for energy to account for electric heating increase from reducing waste heat from efficient lighting (if fossil fuel heating, see calculation of heating penalty in that section).
= 1 - ((HF / ηHeat) * %ElecHeat).
= If unknown assume 0.88³⁶⁸

³⁶¹ Assumed based on program delivery channels.

³⁶² Ameren Missouri Community Savers Evaluation: PY2018, . <https://www.efis.psc.mo.gov/Document/Display/36053>, page 17.

³⁶³ Ameren Missouri Efficient Kits Impact and Process Evaluation: Program Year 2018, <https://www.efis.psc.mo.gov/Document/Display/15870>, page 38.

³⁶⁴ Ameren Missouri Lighting Impact and Process Evaluation: Program Year 2015. Average daily HOU for efficient bulbs is listed as 3.6 for outside bulbs and a weighted (by inventory) average of 1.99 for inside spaces. Unknown location is weighted average (by inventory) of all bulbs. See ‘MO Lamp Hours.xlsx’ for calculations.

³⁶⁵ Ameren Missouri Lighting Evaluation PY2018, <https://www.efis.psc.mo.gov/Document/Display/15873>, page 36.

³⁶⁶ Ameren Missouri Community Savers Evaluation PY2018 workpapers- Weighted Avg. HOU from ADM workpapers.

³⁶⁷ Ameren Missouri PY2014 Evaluation, <https://www.efis.psc.mo.gov/Document/Display/14194>, page 45.

³⁶⁸ Calculated using defaults: 1-((0.53/1.57) * 0.35) = 0.88.

Where:

- HF

= Heating Factor or percentage of light savings that must now be heated
= 53%³⁶⁹ for interior or unknown location
= 0% for exterior or unheated location
- $\eta_{\text{HeatElectric}}$

= Efficiency in COP of Heating equipment
= Actual - If not available, use:³⁷⁰

System Type	Age of Equipment	HSPF2 Estimate	η_{Heat} (COP Estimate) = (HSPF2/3.413)*0.85
Heat Pump (if age unknown assume 2006-2014)	Before 2006	5.8	1.44
	After 2006 - 2014	6.5	1.62
	2015 on	7.0	1.74
Resistance	N/A	N/A	1.00
Unknown ³⁷¹	N/A	N/A	1.28

- %ElecHeat

= Percentage of heating savings assumed to be electric

Heating fuel	%ElectricHeat
Electric	100%
Natural Gas	0%
Unknown	35% ³⁷²

- WHFeCool

= Waste Heat Factor for energy to account for cooling savings from reducing waste heat from efficient lighting

Bulb Location	WHFeCool
Building with cooling	1.12 ³⁷³
Building without cooling or exterior	1.0
Unknown	1.11 ³⁷⁴

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW = \Delta kWh * CF$

Where:

- ΔkWh

= Electric energy savings, as calculated above.
- CF

= Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0001492529 for Lighting RES (Residential)

³⁶⁹ This means that heating loads increase by 53% of the lighting savings. This is based on the average result from REMRate modeling of several different building configurations in Iowa (Des Moines, Mason City, and Burlington). These results were judged to be equally applicable to Missouri.

³⁷⁰ These default system efficiencies are based on the applicable minimum federal standards. In 2006 and 2015, the federal standard for heat pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

³⁷¹ Calculation assumes 35% Heat Pump and 65% Resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, <https://www.eia.gov/consumption/residential/data/2009/hc/hc6.9.xls>; 'hc6.9.xls'. Average efficiency of heat pump is based on assumption that 50% are units from before 2006 and 50% from 2006-2014. Program or evaluation data should be used to improve this assumption if available.

³⁷² Average (default) value of 35% electric space heating from 2009 Residential Energy Consumption Survey for Missouri. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then they should be used.

³⁷³ The value is estimated at 1.12 (calculated as $1 + (0.34 / 2.8)$), and it is based on cooling loads decreasing by 34% of the lighting savings (average result from REMRate modeling of several different building configurations in Iowa (Des Moines, Mason City, and Burlington)). The estimate also assumes typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm $(-0.02 * SEER^2) + (1.12 * SEER)$ (from Wassmer, M. (2003); A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to $COP = EER/3.412 = 2.8COP$). Results of the Iowa study are assumed to be applicable to Missouri.

³⁷⁴ The value is estimated at 1.11 (calculated as $1 + (0.91*(0.34 / 2.8))$), which is based on assumption that 91% of homes have central cooling (based on 2009 Residential Energy Consumption Survey, see <https://www.eia.gov/consumption/residential/data/2009/hc/hc6.9.xls>; 'hc6.9.xls').

NATURAL GAS SAVINGS

Heating Penalty for Natural Gas heated homes³⁷⁵

$$\Delta \text{Therms} = -((\text{Watts}_{\text{Base}} - \text{Watts}_{\text{EE}}) / 1,000 * \text{ISR} * \text{Hours} * \text{HF} * 0.03412) / \eta_{\text{Heat}} * \% \text{GasHeat}$$

Where:

- HF = Heating Factor or percentage of light savings that must be heated
= 53%³⁷⁶ for interior or unknown location
= 0% for exterior or unheated location
- 0.03412 = Converts kWh to therms
- η_{HeatGas} = Efficiency of heating system
= 71%³⁷⁷
- %GasHeat = Percentage of homes with gas heat

Heating fuel	%GasHeat
Electric	0%
Gas	100%
Unknown	65% ³⁷⁸

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

MEASURE CODE:

³⁷⁵ Negative value because this is an increase in heating consumption due to the efficient lighting.

³⁷⁶ This means that heating loads increase by 53% of the lighting savings. This is based on the average result from REMRate modeling of several different building configurations in Des Moines, Mason City, and Burlington, Iowa. Results of the Iowa study were judged to be equally applicable to Missouri.

³⁷⁷ This has been estimated assuming that natural gas central furnace heating is typical for Missouri residences (the predominant heating is gas furnace with 48% of Missouri homes (based on Energy Information Administration, 2009 Residential Energy Consumption Survey)). See reference <https://www.eia.gov/consumption/residential/data/2009/hc/hc6.9.xls>; 'hc6.9.xls'. In 2000, 29% of furnaces purchased in Missouri were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years, so units purchased 15 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: ((0.29*0.92) + (0.71*0.8)) * (1-0.15) = 0.71.

³⁷⁸ Average (default) value of 65% gas space heating from 2009 Residential Energy Consumption Survey for Missouri. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then they should be used.

3.5.3 LED Nightlights

DESCRIPTION

This measure describes savings from LED nightlights. This characterization assumes that the LED nightlight is installed in a residential location. This measure was developed to be applicable to the following program types: TOS, NC.

DEFINITION OF BASELINE EQUIPMENT

For this characterization to apply, the high-efficiency equipment must be a qualified LED nightlight.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is assumed to be an incandescent/halogen nightlight.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The estimated useful life of the is estimated is 8 years.³⁷⁹

DEEMED MEASURE COST

Where possible, the actual cost should be used and compared to the baseline cost. If the incremental cost is unknown, assume \$3.35.³⁸⁰

LOADSHAPE

Lighting RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$$\Delta kWh = ((WattsBase - WattsEE) / 1000) * ISR * (1 - Leakage) * Hours * WHFe$$

Where:

- Watts_{base} = Actual wattage if known, if unknown, assume 7W.³⁸¹
- Watts_{EE} = Actual wattage of LED purchased / installed.
- ISR = In Service Rate or the percentage of nightlights rebated that get installed
- Leakage = Adjustment to account for the percentage of program bulbs that move out.
= Actual, or if unknown, 0% for Online Store and 4% for Upstream Lighting or 3.98% if unknown ³⁸²
- Hours = Average hours of use per year
= 4,380³⁸³
- LKG = leakage rate (program bulbs installed outside Ameren Missouri's service area)
= Actual, or if unknown, assume 0%³⁸⁴
- ISR = In Service Rate, the percentage of units rebated that are actually in service. Actual, or if unknown, assume:

Program	Discounted In Service Rate (ISR)
MFIE ³⁸⁵	98.2%
SFIE, PAYS ³⁸⁶	100%

WHFe = Waste Heat Factor for energy to account for the impact from reducing waste heat from efficient lighting on electric cooling and heating loads in residential homes.

³⁷⁹ Southern California Edison Company, “LED, Electroluminescent & Fluorescent Night Lights”, Work Paper WPSCRELG0029 Rev. 1, February 2009, p. 2. and p.3.
³⁸⁰ Average cost data provided in Stanley Mertz, “LED Nightlights Energy Efficiency Retail products programs”, March 2018.
³⁸¹ Based on Stanley Mertz, “LED Nightlights Energy Efficiency Retail products programs”, March 2018.
³⁸² Ameren Missouri Lighting Evaluation: PY2019. 3.98% is the weighted average for bulbs sold through the Online Store and Upstream Programs based on evaluation results and the distribution of bulbs in the PY19 program, <https://www.efis.psc.mo.gov/Document/Display/15876>, page 13 .
³⁸³ Assumes nightlight is operating 12 hours per day, consistent with the 2016 Pennsylvania TRM.
³⁸⁴ Assumed based on program delivery channels.
³⁸⁵ Ameren Missouri Community Savers Evaluation: PY2018, . <https://www.efis.psc.mo.gov/Document/Display/36053>, page 17.
³⁸⁶ Ameren Missouri Efficient Kits Impact and Process Evaluation: Program Year 2018, <https://www.efis.psc.mo.gov/Document/Display/15870>, page 38.

$WHFe_{Heat} = 0.99$ if unknown³⁸⁷
= Waste Heat Factor for energy to account for electric heating increase from reducing waste heat from efficient lighting (if fossil fuel heating, see calculation of heating penalty in that section).
 $= 1 - ((HF / \eta_{Heat}) * \%ElecHeat)$.
= If unknown assume 0.88³⁸⁸

Where:

HF = Heating Factor or percentage of light savings that must now be heated
= 53%³⁸⁹ for interior location
 $\eta_{Heat_{Electric}}$ = Efficiency in COP of Heating equipment
= Actual - If not available, use:³⁹⁰

System Type	Age of Equipment	HSPF2 Estimate	η_{Heat} (COP Estimate) = $(HSPF2/3.413)*0.85$
Heat Pump (if age unknown assume 2006-2014)	Before 2006	5.8	1.44
	After 2006 - 2014	6.5	1.62
	2015 on	7.0	1.74
Resistance	N/A	N/A	1.00
Unknown ³⁹¹	N/A	N/A	1.28

$\%ElecHeat$ = Percentage of heating savings assumed to be electric

Heating fuel	$\%ElectricHeat$
Electric	100%
Natural Gas	0%
Unknown	35% ³⁹²

$WHFe_{Cool}$ = Waste Heat Factor for energy to account for cooling savings from reducing waste heat from efficient lighting

Bulb Location	$WHFe_{Cool}$
Building with cooling	1.12 ³⁹³
Building without cooling	1.0
Unknown	1.11 ³⁹⁴

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW = \Delta kWh * CF$

³⁸⁷ Ameren Missouri PY2014 Evaluation, <https://www.efis.psc.mo.gov/Document/Display/14194>, page 45.

³⁸⁸ Calculated using defaults: $1 - ((0.53/1.57) * 0.35) = 0.88$.

³⁸⁹ This means that heating loads increase by 53% of the lighting savings. This is based on the average result from REMRate modeling of several different building configurations in Iowa (Des Moines, Mason City, and Burlington). These results were judged to be equally applicable to Missouri.

³⁹⁰ These default system efficiencies are based on the applicable minimum federal standards. In 2006 and 2015, the federal standard for heat pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate.

³⁹¹ Calculation assumes 35% Heat Pump and 65% Resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, <https://www.eia.gov/consumption/residential/data/2009/hc/hc6.9.xls>; 'hc6.9.xls'. Average efficiency of heat pump is based on assumption that 50% are units from before 2006 and 50% from 2006-2014. Program or evaluation data should be used to improve this assumption if available.

³⁹² Average (default) value of 35% electric space heating from 2009 Residential Energy Consumption Survey for Missouri. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then they should be used.

³⁹³ The value is estimated at 1.12 (calculated as $1 + (0.34 / 2.8)$), and it is based on cooling loads decreasing by 34% of the lighting savings (average result from REMRate modeling of several different building configurations in Iowa (Des Moines, Mason City, and Burlington)). The estimate also assumes typical cooling system operating efficiency of 2.8 COP (starting from standard assumption of SEER 10.5 central AC unit, converted to 9.5 EER using algorithm $(-0.02 * SEER^2) + (1.12 * SEER)$ (from Wassmer, M. (2003); A Component-Based Model for Residential Air Conditioner and Heat Pump Energy Calculations. Masters Thesis, University of Colorado at Boulder), converted to $COP = EER/3.412 = 2.8COP$). Results of the Iowa study are assumed to be applicable to Missouri.

³⁹⁴ The value is estimated at 1.11 (calculated as $1 + (0.91*(0.34 / 2.8))$), which is based on assumption that 91% of homes have central cooling (based on 2009 Residential Energy Consumption Survey, see <https://www.eia.gov/consumption/residential/data/2009/hc/hc6.9.xls>; 'hc6.9.xls').

Where:

- ΔkWh = Electric energy savings, as calculated above.
- CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0001492529 for Lighting RES (Residential)

NATURAL GAS SAVINGS

Heating Penalty for Natural Gas heated homes:³⁹⁵

$\Delta Therms = -((Watts_{Base} - Watts_{EE}) / 1,000 * ISR * Hours * HF * 0.03412) / \eta_{Heat} * \%GasHeat$

Where:

- HF = Heating Factor or percentage of light savings that must be heated
= 53%³⁹⁶ for interior or unknown location
= 0% for exterior or unheated location
- 0.03412 = Converts kWh to therms
- $\eta_{Heat_{Gas}}$ = Efficiency of heating system
= 71%³⁹⁷
- $\%GasHeat$ = Percentage of homes with gas heat

Heating fuel	$\%GasHeat$
Electric	0%
Gas	100%
Unknown	65% ³⁹⁸

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

MEASURE CODE:

³⁹⁵ Negative value because this is an increase in heating consumption due to the efficient lighting.

³⁹⁶ This means that heating loads increase by 53% of the lighting savings. This is based on the average result from REMRate modeling of several different building configurations in Des Moines, Mason City, and Burlington, Iowa. Results of the Iowa study were judged to be equally applicable to Missouri.

³⁹⁷ This has been estimated assuming that natural gas central furnace heating is typical for Missouri residences (the predominant heating is gas furnace with 48% of Missouri homes (based on Energy Information Administration, 2009 Residential Energy Consumption Survey)). See reference <https://www.eia.gov/consumption/residential/data/2009/hc/hc6.9.xls>; 'hc6.9.xls'. In 2000, 29% of furnaces purchased in Missouri were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years, so units purchased 15 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: $((0.29 * 0.92) + (0.71 * 0.8)) * (1 - 0.15) = 0.71$.

³⁹⁸ Average (default) value of 65% gas space heating from 2009 Residential Energy Consumption Survey for Missouri. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then they should be used.

3.6 Building Shell

3.6.1 Air Sealing

DESCRIPTION

Thermal shell air leaks are sealed through strategic use and location of air-tight materials. An estimate of savings is provided in two ways. It is highly recommended that leaks be detected and pre- and post-sealing leakage rates measured with the assistance of a blower-door by qualified/certified inspectors.³⁹⁹ Where this occurs, an algorithm is provided to estimate the site-specific savings. Where test in/test out has not occurred, a prescriptive savings assumption is provided.

This measure was developed to be applicable to the following program type: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

Air sealing materials and diagnostic testing should meet all eligibility program qualification criteria. The initial and final tested leakage rates should be assessed in such a manner that the identified reductions can be properly discerned, particularly in situations wherein multiple building envelope measures may be implemented simultaneously.

DEFINITION OF BASELINE EQUIPMENT

The existing air leakage should be determined through approved and appropriate test methods using a blower door. The baseline condition of a building upon first inspection significantly affects the opportunity for cost-effective energy savings through air sealing.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 20 years.⁴⁰⁰

DEEMED MEASURE COST

The actual capital cost for this measure should be used.

LOADSHAPE

Building Shell RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Methodology 1: Test In / Test Out Approach

$\Delta kWh = \Delta kWh_{cooling} + \Delta kWh_{heating}$

If the home has central cooling, the electric energy saved in annual cooling due to the air sealing is:

$\Delta kWh_{cooling} = (((CFM50_{Pre} - CFM50_{Post}) / N_{cool}) * 60 * 24 * CDD * DUA * 0.018 * LM) / ((1,000 * \eta_{Cool}))$

Where:

$CFM50_{Pre} = \text{Infiltration at 50 Pascals as measured by blower door before air sealing}$
 $= Actual^{401}$

³⁹⁹ Refer to the Energy Conservatory Blower Door Manual for more information on testing methodologies.
⁴⁰⁰ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 375. As recommended in Navigant ComEd Effective Useful Life Research Report, ‘ComEd Effective Useful Life Research Report.pdf’, May 2018.
⁴⁰¹ Because the pre- and post-sealing blower door test will occur on different days, there is a potential for the wind and temperature conditions on the two days to affect the readings. There are methodologies to account for these effects. For wind – first, if possible, avoid testing in high wind, place blower door on downwind side, take a pre-test baseline house pressure reading, adjust house pressure readings by subtracting the baseline reading, and use the time averaging feature on the digital gauge, etc. Corrections for air density due to temperature swings can be accounted for with air density correction factors. Refer to the Energy Conservatory Blower Door Manual for more information.

CFM50_{post} = Infiltration at 50 Pascals as measured by blower door after air sealing
= Actual
N_{cool} = Conversion factor from leakage at 50 Pascal to leakage at natural conditions
=Dependent on number of stories:⁴⁰²

Weather Basis (City based upon)	N _{cool} (by # of stories)			
	1	1.5	2	3
St Louis, MO	34.9	30.9	28.3	25.1

60 * 24 = Converts cubic feet per minute to cubic feet per day
CDD = Cooling Degree Days:⁴⁰³

Weather Basis (City based upon)	CDD 65
St Louis, MO	1,646

DUA = Discretionary Use Adjustment (reflects the fact that people do not always operate their AC when conditions may call for it)
= 0.75⁴⁰⁴
0.018 = Specific heat capacity of air (Btu/ft³*°F)
1000 = Converts Btu to kBtu
ηCool = Efficiency (SEER2) of air conditioning equipment (kBtu/kWh)
= Actual (where it is possible to measure or reasonably estimate) - if unknown, assume the following: ⁴⁰⁵

Age of Equipment	ηCool Estimate
Before 2006	9.5
2006 - 2014	12.4
Central AC After 1/1/2015	12.4
Heat Pump After 1/1/2015	13.3

LM = Latent multiplier to account for latent cooling demand: ⁴⁰⁶

Weather Basis (City based upon)	LM
St Louis, MO	3.0

If the home is heated with electric heat (resistance or heat pump), the electric energy saved in annual heating due to the air sealing is:

$$\Delta kWh_{\text{HeatingElectricGas}} = (((CFM50_{\text{Pre}} - CFM50_{\text{Post}}) / N_{\text{heat}}) * 60 * 24 * HDD * 0.018) / (\eta_{\text{Heat}} * 3,412)$$

Where:

N_{heat} = Conversion factor from leakage at 50 Pascal to leakage at natural conditions

⁴⁰² N-factor is used to convert 50-pascal blower door air flows to natural air flows and is dependent on geographic location and # of stories. These were developed by applying the LBNL infiltration model (see LBNL paper 21040, *Exegesis of Proposed ASHRAE Standard 119: Air Leakage Performance for Detached Single-Family Residential Buildings*; Sherman, 1986; page v-vi, Appendix page 7-9, https://eta-publications.lbl.gov/sites/default/files/exegesis_of_proposed_ashrae_std_119.pdf; 'exegesis_of_proposed_ashrae_std_119.pdf') to the reported wind speeds and outdoor temperatures provided by the NRDC 30-year climate normals. For more information see Bruce Harley, CLEAResult 'Infiltration Factor Calculations Methodology-20151123.docx' and calculation worksheets.

⁴⁰³ Based on climate normals data with a base temperature of 65°F.

⁴⁰⁴ This factor's source: Energy Center of Wisconsin, May 2008 metering study; "Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research," 'Energy Center of WI Central AC in WI 2008.pdf', p. 31.

⁴⁰⁵ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. Note all ratings have been converted to SEER2 equivalents – since the new rating better reflects the actual efficiency of the units.

⁴⁰⁶ The LM is used to convert the sensible cooling savings calculated to a value representing sensible and latent cooling loads. The values are derived from the methodology outlined in Infiltration Factor Calculation Methodology by Bruce Harley, Senior Manager, Applied Building Science, CLEAResult 11/18/2015 and is based upon an 8760 analysis of sensible and total heat loads using hourly climate data (see 'Infiltration Factor Calculations Methodology-20151123.docx').

= Based on building height:⁴⁰⁷

Weather Basis (City based upon)	N _{heat} (by # of stories)			
	1	1.5	2	3
St Louis, MO	24.0	21.3	19.5	17.3

HDD = Heating Degree Days

Weather Basis (City based upon)	HDD 65
St Louis, MO	4,486

η_{Heat} = Efficiency of heating system
= Actual - if not available refer to default table below:⁴⁰⁸

System Type	Age of Equipment	HSPF2 Estimate	η_{Heat} (COP Estimate) = (HSPF2/3.413)*0.85
Heat Pump (if age unknown assume 2006-2014)	Before 2006	5.8	1.44
	After 2006 - 2014	6.5	1.62
	2015 on	7.0	1.74
Resistance	N/A	N/A	1.00
Unknown ⁴⁰⁹	N/A	N/A	1.28

3412 = Converts Btu to kWh

If the building is heated with a gas furnace, there will be some electric savings in heating the building attributed to air sealing since the furnace fans will run less.

$\Delta \text{kWh}_{\text{HeatingGas}}$ = $\Delta \text{Therms} * F_e * 29.3$

Where:

F_e = Furnace fan energy consumption as a percentage of annual fuel consumption
= 3.14%⁴¹⁰
29.3 = kWh per therm

⁴⁰⁷ N-factor is used to convert 50-pascal blower door air flows to natural air flows and is dependent on geographic location and # of stories. These were developed by applying the LBNL infiltration model (see LBNL paper 21040, *Exegesis of Proposed ASHRAE Standard 119: Air Leakage Performance for Detached Single-Family Residential Buildings*; Sherman, 1986; page v-vi, Appendix page 7-9, https://eta-publications.lbl.gov/sites/default/files/exegesis_of_proposed_ashrae_std_119.pdf) to the reported wind speeds and outdoor temperatures provided by the NRDC 30 year climate normals. For more information see Bruce Harley, CLEAResult “Infiltration Factor Calculations Methodology.doc” and calculation worksheets.

⁴⁰⁸ These default system efficiencies are based on the applicable minimum federal standards. In 2006 the federal standard for heat pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time means that using the minimum standard is appropriate. An 85% distribution efficiency is then applied to account for duct losses for heat pumps.

⁴⁰⁹ Calculation assumes 35% Heat Pump and 65% Resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, <https://www.eia.gov/consumption/residential/data/2009/hc/hc6.9.xls>; ‘hc6.9.xls’. Average efficiency of heat pump is based on assumption that 50% are units from before 2006 and 50% from 2006-2014. Program or evaluation data should be used to improve this assumption if available.

⁴¹⁰ F_e is not one of the AHRI certified ratings provided for residential furnaces but can be reasonably estimated from a calculation based on the certified values for fuel energy (E_f in MMBtu/yr) and E_{ae} (kWh/yr). An average of a 300-record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the ENERGY STAR® version 3 criteria for 2% F_e . See ‘Furnace Fan Analysis.xlsx’ for reference.

Methodology 2: Prescriptive Infiltration Reduction Measures⁴¹¹

Savings shall only be calculated via Methodology 2 if a blower door test is not conducted.

$\Delta kWh = (\Delta kWh_{cooling} + \Delta kWh_{heating})$

If the home has central cooling, the electric energy saved in annual cooling due to the air sealing is:

$\Delta kWh_{cooling} = (\Delta kWh_{cool_gasket} * n_{gasket} + \Delta kWh_{cool_sweep} * n_{sweep} + \Delta kWh_{cool_sealing} * lf_{sealing} + \Delta kWh_{cool_wx} * lf_{wx}) * ADJ_{RxAirsealing}$

If the home is heated with electric heat (resistance or heat pump), the electric energy saved in annual heating due to the air sealing is:

$\Delta kWh_{heatingelectric} = (\Delta kWh_{heat_gasket} * n_{gasket} + \Delta kWh_{heat_sweep} * n_{sweep} + \Delta kWh_{heat_sealing} * lf_{sealing} + \Delta kWh_{heat_wx} * lf_{wx}) * \%ElectricHeat * ADJ_{RxAirsealing}$

Where:

- n_{gasket} = Number of gaskets installed
- n_{sweep} = Number of sweeps installed
- $lf_{sealing}$ = Linear feet of caulking, sealing, or polyethylene tape
- lf_{wx} = Linear feet of window weatherstripping or door weatherstripping

Measure	Savings Variable Names	$\Delta kWh_{heat}/Unit$			$\Delta kWh_{cool}/Unit$	
		Electric Resistance	Heat Pump	Electric Heat Type Unknown ⁴¹²	With Cooling	Unknown Cooling
Outlet Gasket	ΔkWh_{cool_gasket}	7.19	3.59	5.9	1.63	1.07
	ΔkWh_{heat_gasket}					
Door Sweep	ΔkWh_{cool_sweep}	138.2	69.1	114.0	6.39	4.22
	ΔkWh_{heat_sweep}					
Caulking/Sealing/Polyethylene Tape	$\Delta kWh_{cool_sealing}$	7.91	3.95	6.5	0.17	0.11
	$\Delta kWh_{heat_sealing}$					
Window or door weatherstripping	ΔkWh_{cool_wx}	9.19	4.59	7.6	0.16	0.11
	ΔkWh_{heat_wx}					

$\%ElectricHeat$ = Percentage of homes with electric heat

Heating fuel	$\%ElectricHeat$
Electric	100%
Natural Gas	0%
Unknown	35% ⁴¹³

⁴¹¹ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 382. Prescriptive savings are based upon “Evaluation of the Weatherization Residential Assistance Partnership and Helps Programs (WRAP/Helps).” Middletown, CT: KEMA, 2010. Accessed July 30, 2015, and adjusted for relative HDD of Bridgeport/Hartford CT with the applicable weather data. Cooling savings derived using savings assumptions pulled from ASHRAE, 2001 AHSRAE Handbook – Fundamentals, Chapter 26, Table 1. Effective Air Leakage Areas (Low-Rise Residential Applications Only). See ‘Air Sealing Prescriptive Savings 07.06.2024.xlsx’ for more information.

⁴¹² Calculation assumes 35% Heat Pump and 65% Resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, <https://www.eia.gov/consumption/residential/data/2009/hc/hc6.9.xls>; ‘hc6.9.xls’.

⁴¹³ Average (default) value of 35% electric space heating from 2009 Residential Energy Consumption Survey for Missouri. If utilities have specific evaluation results providing a more appropriate assumption for homes in a particular market or geographical area, then they should be used.

ADJ_{RxAirsealing} = Adjustment for air sealing savings to account for prescriptive estimates overclaiming savings⁴¹⁴
= 80%

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW = \Delta kWh * CF$

Where:

ΔkWh = Electric energy savings, as calculated above.
 CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0004660805⁴¹⁵

NATURAL GAS SAVINGS

Methodology 1: Test In / Test Out Approach

If natural gas heating:

$\Delta Therms = (((CFM50_{Pre} - CFM50_{Post}) / N_{heat}) * 60 * 24 * HDD * 0.018) / (\eta_{Heat} * 100,000)$

Where:

N_{heat} = Conversion factor from leakage at 50 Pascal to leakage at natural conditions
= Based on building height:⁴¹⁶

Weather Basis (City based upon)	N _{heat} (by # of stories)			
	1	1.5	2	3
St Louis, MO	24.0	21.3	19.5	17.3

HDD = Heating Degree Days

Weather Basis (City based upon)	HDD 65
St Louis, MO	4,486

η_{Heat} = Efficiency of heating system
= Equipment efficiency * distribution efficiency
= Actual⁴¹⁷ - if not available, use 71%⁴¹⁸

⁴¹⁴ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, p. 384. Though we do not have a specific evaluation to point to, modeled savings have often been found to overclaim. Further VEIC reviewed these deemed estimates and consider them to likely be a high estimate. As such an 80% adjustment is applied, and this could be further refined with future evaluations.

⁴¹⁵ Based on Ameren Missouri TRM Volume 1 - Appendix G: "Table 1 – Residential End Use Category Monthly Shapes and Coincident Peak Factors" for residential HVAC end-use.

⁴¹⁶ N-factor is used to convert 50-pascal blower door air flows to natural air flows and is dependent on geographic location and # of stories. These were developed by applying the LBNL infiltration model (see LBNL paper 21040, *Exegesis of Proposed ASHRAE Standard 119: Air Leakage Performance for Detached Single-Family Residential Buildings*; Sherman, 1986; page v-vi, Appendix page 7-9, https://eta-publications.lbl.gov/sites/default/files/exegesis_of_proposed_ashrae_std_119.pdf) to the reported wind speeds and outdoor temperatures provided by the NRDC 30-year climate normals. For more information see Bruce Harley, CLEAResult "Infiltration Factor Calculations Methodology.doc" and calculation worksheets.

⁴¹⁷ Ideally, the system efficiency should be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test. The distribution efficiency can be estimated via a visual inspection and by referring to a look up table such as that provided by the Building Performance Institute (https://www.bpi.org/_cms/docs/Guidance%20on%20Estimating%20Distribution%20Efficiency.pdf; 'Guidance on Estimating Distribution Efficiency.pdf') or by performing duct blaster testing.

⁴¹⁸ This has been estimated assuming that natural gas central furnace heating is typical for Missouri residences (the predominant heating is gas furnace with 48% of Missouri homes (based on Energy Information Administration, 2009 Residential Energy Consumption Survey). In 2000, 29% of furnaces purchased in Missouri were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years, so units purchased 16 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: $((0.29 * 0.92) + (0.71 * 0.8)) * (1 - 0.15) = 0.71$.

Other factors as defined above

Methodology 2: Prescriptive Infiltration Reduction Measures⁴¹⁹

Savings shall only be calculated via Methodology 2 if a blower door test is not conducted.

If the home is heated with electric heat (resistance or heat pump), the electric energy saved in annual heating due to the air sealing is:

$$\Delta\text{Therms}_{\text{ADJ}_{\text{RxAirsealing}}} = (\Delta\text{Therms}_{\text{gasket}} * n_{\text{gasket}} + \Delta\text{Therms}_{\text{sweep}} * n_{\text{sweep}} + \Delta\text{Therms}_{\text{sealing}} * \text{If}_{\text{sealing}} + \Delta\text{Therms}_{\text{wx}} * \text{If}_{\text{wx}}) * (1 - \% \text{ElectricHeat}) *$$

Where:

Measure	Savings Variable Names	ΔTherms/Unit
Outlet Gasket	ΔTherms _{gasket}	0.34
Door Sweep	ΔTherms _{sweep}	6.46
Caulking/Sealing/Polyethylene Tape	ΔTherms _{sealing}	0.37
Window or door weatherstripping	ΔTherms _{wx}	0.43

Other factors as defined above

Water Impact Descriptions and Calculation
N/A

DEEMED O&M COST ADJUSTMENT CALCULATION
N/A

MEASURE CODE:

⁴¹⁹ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 382. Prescriptive savings are based upon “Evaluation of the Weatherization Residential Assistance Partnership and Helps Programs (WRAP/Helps).” Middletown, CT: KEMA, 2010. Accessed July 30, 2015, and adjusted for relative HDD of Bridgeport/Hartford CT with the applicable weather data. Cooling savings derived using savings assumptions pulled from ASHRAE, 2001 AHSRAE Handbook – Fundamentals, Chapter 26, Table 1. Effective Air Leakage Areas (Low-Rise Residential Applications Only). See ‘Air Sealing Prescriptive Savings 07.06.2024.xlsx’ for more information.

3.6.2 Ceiling Insulation

DESCRIPTION

This measure describes savings from adding insulation to the attic/ceiling. This measure requires a member of the implementation staff evaluating the pre- and post-project R-values and to measure surface areas. The efficiency of the heating and cooling equipment in the home should also be evaluated if possible.

This measure was developed to be applicable to the following program type: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The requirements for participation in the program will be defined by the utilities.

DEFINITION OF BASELINE EQUIPMENT

The existing condition will be evaluated by implementation staff or a participating contractor.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 30 years.⁴²⁰

DEEMED MEASURE COST

The actual installed cost for this measure should be used in screening.

LOADSHAPE

Building Shell RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

$\Delta kWh = \Delta kWh_{cooling} + \Delta kWh_{heating}$

If the home has central cooling, the electric energy saved in annual cooling due to the ceiling insulation is:

$\Delta kWh_{cooling} = ((1/R_{Old} - 1/R_{Attic}) * A_{Attic} * (1 - FramingFactor_{Attic}) * CDD * 24 * DUA * ADJ_{AtticCool}) / (1,000 * \eta_{Cool})$

Where:

- R_{Attic} = R-value of new attic assembly including all layers between inside air and outside air (ft².°F.h/Btu)
- R_{Old} = R-value value of existing assembly and any existing insulation (Minimum of R-5 for uninsulated assemblies⁴²¹)
- A_{Attic} = Total area of insulated ceiling/attic (ft²)
- $FramingFactor_{Attic}$ = Adjustment to account for area of framing = 7%⁴²²
- CDD = Cooling Degree Days:⁴²³

⁴²⁰ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 421. As recommended in Guidehouse ‘EMV Group A, Deliverable 16 EUL Research – Residential Insulation’ (see <https://pda.energydataweb.com/api/view/2518/CPUC%20Insulation%20EUL%20Draft%20Report%2006292021.pdf>; ‘CPUC Insulation EUL Draft Report 06292021.pdf’), prepared for California Public Utilities Commission, June 2021.

⁴²¹ An estimate based on review of Madison Gas and Electric, Exterior Wall Insulation, R-value for no insulation in walls, and NREL’s Building Energy Simulation Test for Existing Homes (BESTEST-EX).

⁴²² ASHRAE, 2001, “Characterization of Framing Factors for New Low-Rise Residential Building Envelopes (904-RP),” ‘2001 - ASHRAE - Characterization of Framing Factors.pdf’, Table 7.1

⁴²³ Based on climate normals data with a base temp of 65°F.

Weather Basis (City based upon)	CDD 65
St Louis, MO	1,646

- 24 = Converts days to hours
DUA = Discretionary Use Adjustment (reflects the fact that people do not always operate their AC when conditions may call for it)
= 0.75⁴²⁴
1000 = Converts Btu to kBtu
 η_{Cool} = Seasonal energy efficiency ratio of cooling system (kBtu/kWh)
= Actual (where it is possible to measure or reasonably estimate) – if unknown, assume the following: ⁴²⁵

Age of Equipment	η_{Cool} Estimate
Before 2006	9.5
2006 - 2014	12.4
Central AC After 1/1/2015	12.4
Heat Pump After 1/1/2015	13.3
Unknown	12.4

- $ADJ_{AtticCool}$ = Adjustment to cooling savings to account for inaccuracies in engineering algorithms.
= 114%⁴²⁶

If the home is heated with electric heat (resistance or heat pump), the electric energy saved in annual heating due to the ceiling insulation is:

$$\Delta kWh_{HeatingElectric} = ((1/R_{Old} - 1/R_{Attic}) * A_{attic} * (1 - FramingFactor_{Attic}) * HDD * 24 * ADJ_{AtticHeat}) / (3,412 * \eta_{Heat})$$

Where:

- HDD = Heating Degree Days

Weather Basis (City based upon)	HDD 65
St Louis, MO	4,486

- η_{Heat} = Efficiency of heating system
= Actual - if not available, refer to default table below:⁴²⁷

System Type	Age of Equipment	HSPF2 Estimate	η_{Heat} (COP Estimate) = (HSPF2/3.413)*0.85
Heat Pump (if age unknown assume 2006-2014)	Before 2006	5.8	1.44
	After 2006 - 2014	6.5	1.62
	2015 on	7.0	1.74
Resistance	N/A	N/A	1.00
Unknown ⁴²⁸	N/A	N/A	1.28

⁴²⁴ This factor's source: Energy Center of Wisconsin, May 2008 metering study; "Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research," "Energy Center of WI Central AC in WI 2008.pdf", p. 31.

⁴²⁵ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. Note all ratings have been converted to SEER2 equivalents – since the new rating better reflects the actual efficiency of the units.

⁴²⁶ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 424. As demonstrated in air sealing and insulation research by Navigant, see Navigant (2018). ComEd and Nicor Gas Air Sealing and Insulation Research Report. Presented to Commonwealth Edison Company and Nicor Gas Company. Adjustment factor was derived from a consumption data regression analysis with an experimental design that does not require further net savings adjustment for non-income eligible populations. Applied the percent change of NCEI Annual Normals CDD65 from 30 yr data set (1981-2010) to more recent 15 yr data (2006-2020) for all cooling-related adjustment values.

⁴²⁷ These default system efficiencies are based on the applicable minimum federal standards. In 2006 the federal standard for heat pumps was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time means that using the minimum standard is appropriate. An 85% distribution efficiency is then applied to account for duct losses for heat pumps.

⁴²⁸ Calculation assumes 35% Heat Pump and 65% Resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, <https://www.eia.gov/consumption/residential/data/2009/hc/hc6.9.xls>; 'hc6.9.xls'. Average efficiency of heat pump is based on assumption that 50% are units from before 2006 and 50% from 2006-2014. Program or evaluation data should be used to improve this assumption if available.

3,412

ADJ_{AtticHeat}

= Converts Btu to kWh

= Adjustment to heating savings to account for to account for inaccuracies in engineering algorithms.

= 63%⁴²⁹

If the home is heated with electric heat (resistance or heat pump), the electric energy saved in annual heating due to the ceiling insulation is:

$\Delta kWh_{\text{HeatingElectricGas}}$

= $\Delta \text{Therms} * F_e * 29.3$

Where:

F_e

29.3

= Furnace fan energy consumption as a percentage of annual fuel consumption

= 3.14%⁴³⁰

= kWh per therm

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW = \Delta kWh * CF$

Where:

ΔkWh

CF

= Electric energy savings, as calculated above.

= Summer peak coincidence demand (kW) to annual energy (kWh) factor

= 0. 0004660805⁴³¹

NATURAL GAS SAVINGS

Methodology 1:

$\Delta \text{Therms (if Natural Gas heating)}$

= $((1/R_{\text{Old}} - 1/R_{\text{Attic}}) * A_{\text{attic}} * (1 - \text{FramingFactor}_{\text{Attic}}) * \text{HDD} * 24 * \text{ADJ}_{\text{AtticHeat}}) / (100,000 * \eta_{\text{Heat}})$

Where:

HDD

= Heating Degree Days

Weather Basis (City based upon)	HDD 65
St Louis, MO	4,486

η_{Heat}

= Efficiency of heating system

= Equipment efficiency * distribution efficiency

⁴²⁹ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 425. As demonstrated in air sealing and insulation research by Navigant, Navigant (2018). ComEd and Nicor Gas Air Sealing and Insulation Research Report. Presented to Commonwealth Edison Company and Nicor Gas Company. Applied the percent change of NCEI Annual Normals HDD60 from 30 yr data set (1981-2010) to more recent 15 yr data (2006-2020) for all heating-related adjustment values.

⁴³⁰ F_e is not one of the AHRI certified ratings provided for residential furnaces but can be reasonably estimated from a calculation based on the certified values for fuel energy (E_f in MMBtu/yr) and E_{ae} (kWh/yr). An average of a 300 record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the ENERGY STAR® version 3 criteria for 2% F_e . See ‘Furnace Fan Analysis.xlsx’ for reference.

⁴³¹ Based on Ameren Missouri TRM Volume 1 - Appendix G: "Table 1 – Residential End Use Category Monthly Shapes and Coincident Peak Factors" for residential HVAC end-use.

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100,000 = Actual.⁴³² If unknown, assume 71%.⁴³³
= Converts Btu to therms
Other factors as defined above.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

⁴³² Ideally, the system efficiency should be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test. The distribution efficiency can be estimated via a visual inspection and by referring to a look up table such as that provided by the Building Performance Institute (https://www.bpi.org/_cms/docs/Guidance%20on%20Estimating%20Distribution%20Efficiency.pdf; ‘Guidance on Estimating Distribution Efficiency.pdf’) or by performing duct blaster testing.

⁴³³ This has been estimated assuming that natural gas central furnace heating is typical for Missouri residences (the predominant heating is gas furnace with 48% of Missouri homes based on Energy Information Administration, 2009 Residential Energy Consumption Survey). In 2000, 29% of furnaces purchased in Missouri were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years, so units purchased 16 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: $((0.29*0.92) + (0.71*0.8)) * (1-0.15) = 0.71$.

3.6.3 Duct Insulation

DESCRIPTION

This measure describes evaluating the savings associated with performing duct insulation on the distribution system of homes with central cooling and/or a ducted heating system. While insulating ducts in conditioned space can help with control and comfort, energy savings are largely limited to insulating ducts in unconditioned space where the heat loss is to outside the thermal envelope. Therefore, for this measure to be applicable, at least 30% of ducts should be within unconditioned space (e.g., attic with floor insulation, vented crawlspace, unheated garages. Basements should be considered conditioned space).

This measure was developed to be applicable to the following program type: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient condition is insulated duct work throughout the unconditioned space in the home.

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is existing duct work with at least 30% of the ducts within the unconditioned space in the home.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 20 years.⁴³⁴

DEEMED MEASURE COST

The actual duct insulation measure cost should be used.

LOADSHAPE

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Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Electric energy savings is calculated as the sum of energy saved when cooling the home and energy saved when heating the home.

$\Delta kWh = \Delta kWh_{cooling} + \Delta kWh_{heating}$

If the home has central cooling, the electric energy saved in annual cooling due to the added insulation is:

$\Delta kWh_{Cooling} = (1/R_{existing} - 1/R_{new}) * Area * EFLH_{cool} * \Delta T_{AVG,cooling} / (1,000 * SEER2)$

Where:

- $R_{existing}$ = Duct heat loss coefficient with existing insulation ((hr-°F-ft²)/Btu)
= Actual
- R_{new} = Duct heat loss coefficient with new insulation (hr-°F-ft²)/Btu)
= Actual
- Area = Area of the duct surface exposed to the unconditioned space that has been insulated (ft²)
- $EFLH_{cool}$ = Equivalent Full Load Cooling Hours:

⁴³⁴ Measure Life Report, Residential and Commercial/Industrial Lighting and HVAC Measures, <https://energizect.com/sites/default/files/documents/Measure%20Life%20Report%202007.pdf>; ‘Measure Life Report 2007.pdf’, page 1-3.

Weather Basis (Ameren Missouri Average)	EFLH _{cool} (Hours)
SF or MF	869 ⁴³⁵

$\Delta T_{AVG,cooling}$ = Average temperature difference (⁰F) during cooling season between outdoor air temperature and assumed 60⁰F duct supply air temperature⁴³⁶

Weather Basis (City based upon)	OA _{AVG,cooling} [⁰ F] ⁴³⁷	$\Delta T_{AVG,cooling}$ [⁰ F]
St Louis, MO	80.8	20.8

1,000 = Converts Btu to kBtu

SEER2 = Seasonal Energy Efficiency Ratio of Air Conditioning equipment (kBtu/kWh)

= Actual - If not available, assume the following: ⁴³⁸

Age of Equipment	η_{Cool} Estimate
Before 2006	9.5
2006 - 2014	12.4
Central AC After 1/1/2015	12.4
Heat Pump After 1/1/2015	13.3

If the home is heated with electric heat (resistance or heat pump), the electric energy saved in annual heating due to the added insulation is:

$$\Delta kWh_{HeatingElectric} = (1/R_{existing} - 1/R_{new}) * Area * EFLH_{heat} * \Delta T_{AVG,heating} / (3,412 * COP)$$

Where:

EFLH_{heat} = Equivalent Full Load Heating Hours:⁴³⁹

Weather Basis (Ameren Missouri Average)	EFLH _{heat} (Hours)
SF or MF	1,496

$\Delta T_{AVG,heating}$ = Average temperature difference (⁰F) during heating season between outdoor air temperature and assumed 115⁰F duct supply temperature⁴⁴⁰

Weather Basis (City based upon)	OA _{AVG,heating} [⁰ F] ⁴⁴¹	$\Delta T_{AVG,heating}$ [⁰ F]
St Louis, MO	43.2	71.8

3,412 = Converts Btu to kWh

COP = Efficiency in COP of heating equipment

= Actual - if not available, use:

System Type	Age of Equipment	HSPF2 Estimate	η_{Heat}
-------------	------------------	----------------	---------------

⁴³⁵ PY2019 Residential Evaluation Report, <https://www.efis.psc.mo.gov/Document/Display/15876>, page 30.

⁴³⁶ Leaving coil air temperatures are typically about 55⁰F. Therefore, 60⁰F is used as an average temperature, recognizing that some heat transfer occurs between the ductwork and the environment it passes through.

⁴³⁷ National Solar Radiation Data Base -- 1991- 2005 Update: Typical Meteorological Year 3 <https://doe2.com/Download/Weather/TMY3/>. Heating season defined as September 17th through April 13th, cooling season defined as May 20 through August 15th. For cooling season, temperatures from 8AM to 8PM were used to establish average temperatures as this is when cooling systems are expected to be loaded.

⁴³⁸ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. Note all ratings have been converted to SEER2 equivalents – since the new rating better reflects the actual efficiency of the units.

⁴³⁹ Evaluation - Opinion Dynamics review PY2019. The recommended values are constructed based on weather conditions (heating degree days and cooling degree days) in select Missouri cities (St. Louis, Cape Girardeau, Kansas City), weighted by partial year 2019 installations.

⁴⁴⁰ Forced air supply temperatures are typically 130⁰F. 115⁰F is used as an average temperature, recognizing that some heat transfer occurs between the ductwork and the environment it passes through.

⁴⁴¹ National Solar Radiation Data Base -- 1991- 2005 Update: Typical Meteorological Year 3 <https://doe2.com/Download/Weather/TMY3/>. Heating season defined as September 17 through April 13, cooling season defined as May 20 through August 15. For cooling season, temperatures from 8AM to 8PM were used to establish average temperatures as this is when cooling systems are expected to be loaded.

			(COP Estimate) = (HSPF2/3.413)*0.85
Heat Pump (if age unknown assume 2006-2014)	Before 2006	5.8	1.44
	After 2006 - 2014	6.5	1.62
	2015 on	7.0	1.74
Resistance	N/A	N/A	1.00
Unknown ⁴⁴²	N/A	N/A	1.28

If the building is heated with a gas furnace, there will be some electric savings in heating the building attributed to extra insulation since the furnace fans will run less.

$\Delta kWh_{\text{HeatingGas}} = \Delta \text{Therms} * Fe * 29.3$

Where:

- ΔTherms = Therm savings as calculated in Natural Gas Savings
- Fe = Furnace fan energy consumption as a percentage of annual fuel consumption
= 3.14%⁴⁴³
- 29.3 = Converts therms to kWh

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW = \Delta kWh_{\text{Cooling}} * CF$

Where:

- $\Delta kWh_{\text{Cooling}}$ = Electric energy savings for cooling, calculated above
- CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0004660805

NATURAL GAS SAVINGS

If home uses a gas heating system, the savings resulting from the insulation is calculated with the following formula.

$\Delta \text{Therms} = (1/R_{\text{existing}} - 1/R_{\text{new}}) * \text{Area} * EFLH_{\text{heat}} * \Delta T_{\text{AVG,heating}} / (100,000 * \eta_{\text{HeatGas}})$

Where: η_{HeatGas} equals 71%⁴⁴⁴ and all factors as defined above.

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

MEASURE CODE:

⁴⁴² Calculation assumes 35% Heat Pump and 65% Resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, <https://www.eia.gov/consumption/residential/data/2009/hc/hc6.9.xls>; 'hc6.9.xls'. Average efficiency of heat pump is based on assumption that 50% are units from before 2006 and 50% from 2006-2014. Program or evaluation data should be used to improve this assumption if available.

⁴⁴³ Fe is not one of the AHRI certified ratings provided for residential furnaces but can be reasonably estimated from a calculation based on the certified values for fuel energy (E_f in MMBtu/yr) and E_{ae} (kWh/yr). An average of a 300-record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the ENERGY STAR[®] version 3 criteria for 2% Fe .

⁴⁴⁴ This has been estimated assuming that natural gas central furnace heating is typical for Missouri residences (the predominant heating is gas furnace with 48% of Missouri homes (based on Energy Information Administration, 2009 Residential Energy Consumption Survey). In 2000, 29% of furnaces purchased in Missouri were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years, so units purchased 16 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: $((0.29*0.92) + (0.71*0.8)) * (1-0.15) = 0.71$.

3.6.4 Floor Insulation

DESCRIPTION

Insulation is added to the floor above a vented crawl space that does not contain pipes or HVAC equipment. If there are pipes, HVAC, or a basement, it is desirable to keep them within the conditioned space by insulating the crawl space walls and ground. Insulating the floor separates the conditioned space above from the space below the floor and is only acceptable when there is nothing underneath that could freeze or would operate less efficiently in an environment resembling the outdoors. Even in the case of an empty, unvented crawl space, it is still considered best practice to seal and insulate the crawl space perimeter rather than the floor. Not only is there generally less area to insulate this way, but there are also moisture control benefits. There is a “Foundation Sidewall Insulation” measure for perimeter sealing and insulation. This measure assumes the insulation is installed above an unvented crawl space and should not be used in other situations.

This measure was developed to be applicable to the following program type: RF.

If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The requirements for participation in the program will be defined by the utilities.

DEFINITION OF BASELINE EQUIPMENT

The existing condition will be evaluated by implementation staff or a participating contractor and is likely to be no insulation on any surface surrounding a crawl space.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 30 years.⁴⁴⁵

DEEMED MEASURE COST

The actual installed cost for this measure should be used in screening.

LOADSHAPE

Building Shell RES

Algorithm

CALCULATION OF SAVINGS

ELECTRIC ENERGY SAVINGS

Where available, savings from shell insulation measures should be determined through a custom analysis. When that is not feasible, the following engineering algorithms can be used with the inclusion of an adjustment factor to de-rate the heating savings:

$\Delta kWh = \Delta kWh_{cooling} + \Delta kWh_{heating}$

If the home has central cooling, the electric energy saved in annual cooling due to the floor insulation is:

$\Delta kWh_{cooling} = ((1/R_{Old} - 1/(R_{Added} + R_{Old})) * Area * (1 - FramingFactor_{Floor}) * CDD * 24 * DUA * ADJ_{FloorCool}) / (1,000 * \eta_{Cool})$

Where:

- R_{Old} = R-value value of floor before insulation, assuming 3/4” plywood subfloor and carpet with pad
= Actual. If unknown, assume 3.53⁴⁴⁶
- R_{Added} = R-value of additional spray foam, rigid foam, or cavity insulation.
- $Area$ = Total floor area to be insulated

⁴⁴⁵ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 402. As recommended in Guidehouse ‘EMV Group A, Deliverable 16 EUL Research – Residential Insulation’ (see <https://pda.energydataweb.com/api/view/2518/CPUC%20Insulation%20EUL%20Draft%20Report%2006292021.pdf>; ‘CPUC Insulation EUL Draft Report 06292021.pdf’), prepared for California Public Utilities Commission, June 2021.

⁴⁴⁶ Ibid., page 404. Based on 2005 ASHRAE Handbook – Fundamentals: assuming ¾” subfloor, ½” carpet with rubber pad, and accounting for a still air film above and below: 0.68 + 0.94 + 1.23 + 0.68 = 3.53.

FramingFactor_{Floor} = Adjustment to account for area of framing
= 12%⁴⁴⁷
24 = Converts hours to days
CDD = Cooling Degree Days

Weather Basis (City based upon)	Unconditioned Space
	CDD 75 ⁴⁴⁸
St Louis, MO	762

DUA = Discretionary Use Adjustment (reflects the fact that people do not always operate their AC when conditions call for it).
= 0.75⁴⁴⁹
1000 = Converts Btu to kBtu
η_{Cool} = Seasonal energy efficiency ratio of cooling system (kBtu/kWh)
= Actual (where it is possible to measure or reasonably estimate). If unknown, assume the following: ⁴⁵⁰

Age of Equipment	η _{Cool} Estimate
Before 2006	9.5
2006 - 2014	12.4
Central AC After 1/1/2015	12.4
Heat Pump After 1/1/2015	13.3

ADJ_{FloorCool} = Adjustment to cooling savings to account for to account for inaccuracies in engineering algorithms.
= 75%⁴⁵¹

If the home is heated with electric heat (resistance or heat pump), the electric energy saved in annual heating due to the floor insulation is:

ΔkWh_{HeatingElectric} = ((1/R_{Old} - 1/(R_{Added} + R_{Old})) * Area * (1 – FramingFactor_{Floor}) * HDD * 24 * ADJ_{FloorHeat}) / (3,412 * η_{Heat})

Where:

HDD = Heating Degree Days:

Weather Basis Zone (City based upon)	Unconditioned Space
	HDD 50 ⁴⁵²
St Louis, MO	1,911

η_{Heat} = Efficiency of heating system
= Actual -- if not available, refer to default table below:

⁴⁴⁷ ASHRAE, 2001, “Characterization of Framing Factors for New Low-Rise Residential Building Envelopes (904-RP),” ‘2001 - ASHRAE - Characterization of Framing Factors.pdf’, Table 7.1.
⁴⁴⁸ The base temperature should be the outdoor temperature at which the desired indoor temperature stays constant in balance with heat loss or gain to the outside and internal gains. Since unconditioned basements are allowed to swing in temperature, are ground coupled, and are usually cool, they have a bigger delta between the two (heating and cooling) base temperatures. 75F for cooling and 50F for heating are used based on professional judgment. Five-year average cooling degree days with 75F base temp are provided from DegreeDays.net because the 30 year climate normals from NCDC are not available at base temps above 72F.
⁴⁴⁹ This factor's source: Energy Center of Wisconsin, May 2008 metering study; “Central Air Conditioning in Wisconsin, A Compilation of Recent Field Research,” ‘Energy Center of WI Central AC in WI 2008.pdf’, p. 31.
⁴⁵⁰ These default system efficiencies are based on the applicable minimum Federal Standards. In 2006 the Federal Standard for Central AC was adjusted. While one would expect the average system efficiency to be higher than this minimum, the likely degradation of efficiencies over time mean that using the minimum standard is appropriate. Note all ratings have been converted to SEER2 equivalents – since the new rating better reflects the actual efficiency of the units.
⁴⁵¹ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 405. As demonstrated in two years of metering evaluation by Opinion Dynamics, see Memo “Results for AIC PY6 HPwES Billing Analysis” (‘Memo on Ameren HPwES Billing Analysis FINAL 2015-03-06.pdf’), dated February 20, 2015. Applied the percent change of NCEI Annual Normals CDD65 from 30 yr data set (1981-2010) to more recent 15 yr data (2006-2020) for all cooling-related adjustment values.
⁴⁵² The base temperature should be the outdoor temperature at which the desired indoor temperature stays constant in balance with heat loss or gain to the outside and internal gains. Since unconditioned basements are allowed to swing in temperature, are ground coupled, and are usually cool, they have a bigger delta between the two (heating and cooling) base temperatures. 75F for cooling and 50F for heating are used based on professional judgment. National Climatic Data Center, calculated from 1981-2010 climate normals.

System Type	Age of Equipment	HSPF2 Estimate	η_{Heat} (COP Estimate) $= (\text{HSPF2}/3.413) \times 0.85$
Heat Pump (if age unknown assume 2006-2014)	Before 2006	5.8	1.44
	After 2006 - 2014	6.5	1.62
	2015 on	7.0	1.74
Resistance	N/A	N/A	1.00
Unknown ⁴⁵³	N/A	N/A	1.28

$\text{ADJ}_{\text{FloorHeat}}$ = Adjustment to heating savings to account for to account for inaccuracies in engineering algorithms.
= 63%⁴⁵⁴

If the home is heated with electric heat (resistance or heat pump), the electric energy saved in annual heating due to the floor insulation is:

$\Delta \text{kWh}_{\text{HeatingElectricGas}}$ = $\Delta \text{Therms} \times F_e \times 29.3$

Where:

F_e = Furnace fan energy consumption as a percentage of annual fuel consumption
= 3.14%⁴⁵⁵
29.3 = kWh per therm

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta \text{kW} = \Delta \text{kWh} \times \text{CF}$

Where:

ΔkWh = Electric energy savings, as calculated above.
 CF = Summer peak coincidence demand (kW) to annual energy (kWh) factor
= 0.0004660805⁴⁵⁶

NATURAL GAS SAVINGS

ΔTherms (if Natural Gas heating)

= $((1/R_{\text{Old}} - 1/(R_{\text{Added}} + R_{\text{Old}})) \times \text{Area} \times (1 - \text{FramingFactor}_{\text{Floor}}) \times \text{HDD} \times 24 \times \text{ADJ}_{\text{FloorHeat}}) / (100,000 \times \eta_{\text{Heat}})$

Where

η_{Heat} = Efficiency of heating system
= Equipment efficiency * distribution efficiency

⁴⁵³ Calculation assumes 35% Heat Pump and 65% Resistance, which is based upon data from Energy Information Administration, 2009 Residential Energy Consumption Survey, <https://www.eia.gov/consumption/residential/data/2009/hc/hc6.9.xls>; 'hc6.9.xls'. Average efficiency of heat pump is based on assumption that 50% are units from before 2006 and 50% from 2006-2014. Program or evaluation data should be used to improve this assumption if available.

⁴⁵⁴ Illinois TRM Version 12.0, https://www.ilsag.info/wp-content/uploads/IL-TRM_Effective_010124_v12.0_Vol_3_Res_09222023_FINAL_clean.pdf, page 406. As demonstrated in two years of metering evaluation by Opinion Dynamics, see Memo "Results for AIC PY6 HPwES Billing Analysis" ('Memo on Ameren HPwES Billing Analysis FINAL 2015-03-06.pdf'), dated February 20, 2015. Applied the percent change of NCEI Annual Normals HDD60 from 30 yr data set (1981-2010) to more recent 15 yr data (2006-2020) for all heating-related adjustment values.

⁴⁵⁵ F_e is not one of the AHRI certified ratings provided for residential furnaces but can be reasonably estimated from a calculation based on the certified values for fuel energy (E_f in MMBtu/yr) and E_{ae} (kWh/yr). An average of a 300-record sample (non-random) out of 1495 was 3.14%. This is, appropriately, ~50% greater than the ENERGY STAR® version 3 criteria for 2% F_e . See 'Programmable Thermostats Furnace Fan Analysis.xlsx' for reference.

⁴⁵⁶ Based on Ameren Missouri TRM Volume 1 - Appendix G: "Table 1 – Residential End Use Category Monthly Shapes and Coincident Peak Factors" for residential building shell end-use.

100,000

= Actual⁴⁵⁷ - If not available, use 71%⁴⁵⁸

= Converts Btu to therms

Other factors as defined above.

WATER IMPACT DESCRIPTIONS AND CALCULATION
N/A

DEEMED O&M COST ADJUSTMENT CALCULATION
N/A

MEASURE CODE:

⁴⁵⁷ Ideally, the system efficiency should be obtained either by recording the AFUE of the unit, or performing a steady state efficiency test. The distribution efficiency can be estimated via a visual inspection and by referring to a look up table such as that provided by the Building Performance Institute (https://www.bpi.org/_cms/docs/Guidance%20on%20Estimating%20Distribution%20Efficiency.pdf; ‘Guidance on Estimating Distribution Efficiency.pdf’) or by performing duct blaster testing.

⁴⁵⁸ This has been estimated assuming that natural gas central furnace heating is typical for Missouri residences (the predominant heating is gas furnace with 48% of Missouri homes (based on Energy Information Administration, 2009 Residential Energy Consumption Survey). In 2000, 29% of furnaces purchased in Missouri were condensing (based on data from GAMA, provided to Department of Energy during the federal standard setting process for residential heating equipment - see Furnace Penetration.xls). Furnaces tend to last up to 20 years, so units purchased 16 years ago provide a reasonable proxy for the current mix of furnaces in the State. Assuming typical efficiencies for condensing and non-condensing furnaces and duct losses, the average heating system efficiency is estimated as follows: $((0.29*0.92) + (0.71*0.8)) * (1-0.15) = 0.71$.

3.6.5 Cool Roof

DESCRIPTION

Cool (high albedo) roofing materials reduce the overall heat load on a home by reflecting more of the incident solar radiation, thus decreasing the total heat energy absorbed into the building system. This reduction in heat load provides space cooling energy savings during the cooling season but can increase heating energy use during the winter. Therefore, cool roofs are most beneficial in warmer climates and may not be recommended for homes where the primary heat source is electric resistance.

This measure is only applicable to existing buildings constructed before 2016 that have not undergone roof improvements since 2016.

This measure was developed to be applicable to the following program types: RF, DI. If applied to other program types, the measure savings should be verified.

DEFINITION OF EFFICIENT EQUIPMENT

The efficient condition is cool (high albedo) roofing material. Although, the ENERGY STAR cool roof rating was discontinued, the minimum thresholds are listed for the required minimum solar reflectance and thermal emittance by the roof slope. The *Cool Roof Rating Council* provides ratings at <https://coolroofs.org/directory/roof>.

Roof Slope/Pitch	Solar Reflectance 1 Year	Solar Reflectance 3 Year	Thermal Emittance
Low slope/≤2:12 pitch	≥0.65	≥0.5	≥0.75
Steep slope/>2:12 pitch	≥0.25	≥0.15	≥0.75

DEFINITION OF BASELINE EQUIPMENT

The baseline condition is assumed to be a conventional asphalt shingle roof of albedo 0.142. For other existing roofing materials, the reflectance and emittance values can be sourced, with savings determined by the calculators built by the Oak Ridge National Laboratory for low slope⁴⁵⁹ and steep slope⁴⁶⁰ roofs.

DEEMED LIFETIME OF EFFICIENT EQUIPMENT

The expected measure life is assumed to be 15 years.⁴⁶¹

DEEMED MEASURE COST

The actual implementation cost for applying cool (high albedo) roof should be used.

LOADSHAPE

Building Shell RES

Algorithm

CALCULATION OF SAVINGS

Mid-America Regional Council (MARC), in partnership with Lawrence Berkeley National Laboratory (LBNL), commissioned Leidos to study building energy consumption. The study area was a nine-county region identified by MARC. A whole-building energy modeling tool was used to evaluate urban heat island (UHI) countermeasure strategies for several common residential building categories based on models developed by the U.S. Department of Energy (DOE). Residential buildings were adapted to the Kansas City region to model changes in building energy consumption.

The LBNL study modeled four vintages of residential single and multi-family buildings. They are categorized as,

⁴⁵⁹Oak Ridge National Laboratory, Low slop roof savings calculator, <https://web.ornl.gov/sci/buildings/tools/cool-roof/>

⁴⁶⁰ Oak Ridge National Laboratory, Steep slope roof savings calculator, <https://web.ornl.gov/sci/buildings/tools/SteepSlopeCalc/>

⁴⁶¹ DEER READI (Remote Ex-Ante Database Interface). <http://www.deeresources.com/index.php/component/users/?view=login>.

Vintage Group	Year of construction	Adjusted Distribution %
Pre-1980	up to 1979	59%
Post-1980	1980-1999	25%
IECC 2006	2000-2009	13%
IECC 2012	2010-2015	3%

ELECTRIC ENERGY SAVINGS

$\Delta kWh = \text{Cooling Savings} * SF / 1,000 * \text{HeatingFactor}$

Where:

$\text{Cooling Savings} = \text{Dependent on home vintage and type (single family vs. multi-family): }^{462}$

Vintage Group	kWh/1000 ft² Cooling Savings	
	Single Family	Multi-Family
Up to 1979	136.0	114.0
1980-1999	73.9	58.1
2000-2009	33.3	24.9
2010-2015	23.9	19.5

$SF = \text{Area of cool roof in square feet.}$
 $\text{HeatingFactor} = 0^{463} \text{ for Electric Resistance heating}$
 $\text{HeatingFactor} = 0.42^{464} \text{ for Heat Pump heating}$
 $\text{HeatingFactor} = 1.0 \text{ for non-electric heating}$

SUMMER COINCIDENT PEAK DEMAND SAVINGS

$\Delta kW = \Delta kWh * CF$

Where:

$\Delta kWh = \text{Electric energy savings, as calculated above.}$
 $CF = \text{Summer peak coincidence demand (kW) to annual energy (kWh) factor}$
 $CF = 0.0004660805^{465}$

NATURAL GAS SAVINGS

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

⁴⁶² The annual site energy savings by roof area for each of the residential prototypes from the installation of cool roofs instead of conventional roofs (scenario RR-2 in Leidos 2015b). Leidos. (2015b) Energy savings of high albedo roofs for the Kansas City Area. Leidos report commissioned by the Mid-America Regional Council. September 2015.
⁴⁶³ Average reduction in savings due to electric heat, calculated with the ORNL Cool Roof calculator. Local file: “Residential Cool Roofs.xlsx”
⁴⁶⁴ Ibid.
⁴⁶⁵ Based on Ameren Missouri TRM Volume 1 - Appendix G: "Table 1 – Residential End Use Category Monthly Shapes and Coincident Peak Factors" for residential building shell end-use.

3.7 Residential Demand Response

3.7.1 Residential Demand Response Analysis Approach

DESCRIPTION

For residential demand response measures, the energy and demand impacts of residential demand response events will be analyzed using AMI interval data. The baseline for measuring impacts will be established by creating a control group of non-participant customers with similar energy usage patterns and weather conditions. This control group will be used to estimate what energy usage would have been in the absence of a demand response event.

The analysis will include weather normalization to ensure that the baseline reflects the weather conditions experienced during the event. Demand reduction will be calculated as the difference between the weather-normalized baseline and the actual energy use during the event period.

If AMI data is not available for all participants, results will be extrapolated to represent the broader participant population.

3.7.2 Demand Response Smart Thermostat

DESCRIPTION

This measure characterizes the demand savings achieved by managing customer energy loads during peak periods through a residential demand response (DR) program. It also characterizes the energy savings resulting from load shaping strategies employed during non-peak hours to reduce overall usage. Savings impacts will be evaluated using ex-post analysis, primarily leveraging AMI interval data to compare energy consumption with and without program intervention.

The evaluation approach includes establishing a control group through propensity score matching with non-participant customers who have similar energy usage patterns and weather conditions. The analysis involves weather normalization to ensure accurate baseline estimations. The savings impacts will be determined by comparing the treatment group's energy use during DR events to the weather-normalized baseline.

As smart thermostats evolve, some models include embedded optimization routines that can independently achieve energy savings. The program, however, will only attribute savings to the incremental impact of “program-driven optimization”—those savings achieved through the program’s influence in activating or enhancing the thermostat's optimization features. Energy savings that result from default or non-program-driven optimization will not be attributed to the program.

Due to the custom nature of the evaluation, ex-post demand and energy savings calculations will be based on the specific conditions and data available each year, with previous year results informing program planning and goal setting for subsequent cycles.

This measure was developed to be applicable to the following program type: DR.

DEFINITION OF EFFICIENT CASE

The efficient case is a customer who participates in the DR program, where the thermostat is under the control of the program. In this case, energy consumption is directly influenced by program-driven strategies, including load shaping during non-peak hours and demand reduction during peak periods.

DEFINITION OF BASELINE CASE

The baseline case is a customer who is not participating in the DR program and whose thermostat operates independently of program-driven strategies. The baseline for measuring impacts will be established by creating a control group of non-participant customers with similar energy usage patterns and weather conditions. This control group will be used to estimate what energy usage would have been in the absence of a demand response event. The analysis will include weather normalization to ensure that the baseline reflects the weather conditions experienced during the event. If AMI data is not available for all participants, results will be extrapolated to represent the broader participant population.

DEEMED LIFETIME OF PROGRAM SAVINGS

The expected measure life is assumed to be 1 year.

DEEMED MEASURE COST

It is assumed that program-controlled changes in residential settings are accomplished without homeowner investment in new equipment. Therefore, without evidence to the contrary, measure costs in such residential programs focused on program controlled changes in customer behavior may be defined as \$0.

LOADSHAPE

HVAC RES (for optimization routines that save energy during the cooling and heating seasons)

Cooling RES (for optimization routines that save energy only during the cooling season)

Heating RES (for optimization routines that save energy only during the heating season)

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

3.7.3 Demand Response Water Heater Switch

DESCRIPTION

This measure characterizes the demand savings achieved by controlling residential water heater loads during peak periods through a demand response (DR) program. Savings impacts will be evaluated using ex-post analysis, primarily leveraging AMI interval data to compare energy consumption during peak periods with and without program intervention.

The evaluation approach includes establishing a control group through propensity score matching with non-participant customers who have similar energy usage patterns and weather conditions. The analysis involves weather normalization to ensure accurate baseline estimations. The demand savings impacts will be determined by comparing the treatment group's energy use during DR events to the weather-normalized baseline.

Due to the custom nature of the evaluation, ex-post demand savings calculations will be based on the specific conditions and data available each year, with previous year results informing program planning and goal setting for subsequent cycles.

This measure was developed to be applicable to the following program type: DR.

DEFINITION OF EFFICIENT CASE

The efficient case is a customer who participates in the DR program, where the water heater is under the control of the program. In this case, demand reduction is directly influenced by program-driven strategies during peak periods.

DEFINITION OF BASELINE CASE

The baseline case is a customer who is not participating in the DR program and whose water heater operates independently of program-driven strategies. The baseline for measuring impacts will be established by creating a control group of non-participant customers with similar energy usage patterns and weather conditions. This control group will be used to estimate what demand would have been in the absence of a demand response event. The analysis will include weather normalization to ensure that the baseline reflects the weather conditions experienced during the event. If AMI data is not available for all participants, results will be extrapolated to represent the broader participant population.

DEEMED LIFETIME OF PROGRAM SAVINGS

The expected measure life is assumed to be 1 year.

DEEMED MEASURE COST

The incremental cost of the water heater switch is \$149.00.

LOADSHAPE

N/A

WATER IMPACT DESCRIPTIONS AND CALCULATION

N/A

DEEMED O&M COST ADJUSTMENT CALCULATION

N/A

3.7.4 Demand Response Electric Vehicle Charging

Description

This measure characterizes the demand savings achieved by controlling residential electric vehicle (EV) charger loads during peak periods through a demand response (DR) program. Savings impacts will be evaluated using ex-post analysis, primarily leveraging AMI interval data to compare energy consumption during peak periods with and without program intervention.

The evaluation approach includes establishing a control group through propensity score matching with non-participant customers who have similar energy usage patterns and weather conditions. The analysis involves weather normalization to ensure accurate baseline estimations. The demand savings impacts will be determined by comparing the treatment group's energy use during DR events to the weather-normalized baseline.

Due to the custom nature of the evaluation, ex-post demand savings calculations will be based on the specific conditions and data available each year, with previous year results informing program planning and goal setting for subsequent cycles.

This measure was developed to be applicable to the following program type: DR.

Definition of Efficient Case

The efficient case is a customer who participates in the DR program, where the EV charger is under the control of the program. In this case, demand reduction is directly influenced by program-driven strategies during peak periods.

Definition of Baseline Case

The baseline case is a customer who is not participating in the DR program and whose EV charger operates independently of program-driven strategies. The baseline for measuring impacts will be established by creating a control group of non-participant customers with similar energy usage patterns and weather conditions. This control group will be used to estimate what demand would have been in the absence of a demand response event. The analysis will include weather normalization to ensure that the baseline reflects the weather conditions experienced during the event. If AMI data is not available for all participants, results will be extrapolated to represent the broader participant population.

Deemed Lifetime of Program Savings

The expected measure life is assumed to be 1 year.

Deemed Measure Cost

Since customers do not need to invest in new equipment or incur additional expenses to participate in the program, the deemed measure cost is \$0.

Loadshape

N/A

Water Impact Descriptions and Calculation

N/A

Deemed O&M Cost Adjustment Calculation

N/A